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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

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Buckling - Shear - Bending

No. 910

SOME INVESTIGATIONS OF THE GENERAL INSTABILITY OF STIFFENED METAL CYLINDERS

VI - STIFFENED METAL CYLINDERS SUBJECTED TO COMBINED BENDING AND TRANSVERSE SHEAR

Guggenheim Aeronautical Laboratory
California Institute of Technology

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This is the sixth of a series of reports covering an investigation of the general instability problem by the California Institute of Technology. The first five reports of this series cover investigations of the general instability problem under the loading conditions of pure bending and were prepared under the sponsorship of the Civil Aeronautics Administration. This report and the succeeding reports of this series cover the work done on other loading conditions under the sponsorship of the National Advisory Committee for Aeronautics.

SUMMARY

This report summarizes the work that has been carried on in the experimental investigation of the problem of general instability of stiffened metal cylinders subjected to combined bending and transverse shear at the C.I.T. This part of the investigation includes tests on 55 sheet-covered specimens. Under a loading of combined bending and transverse shear, the failure of the specimens is characterized by two distinct types of failure, namely, a bending or a shear failure. It has also been determined that the failing loads of the specimens have not been influenced by the length of the specimens.

INTRODUCTION

It is intended to give, in this report, a summary of the experimental investigation of the general instability of stiffened metal cylinders subjected to combined bending and transverse shear. The earlier work at C.I.T. on the problem of general instability of stiffened metal cylinders has been reported in references 1, 2, 3, 4, and 5. A total of 55 sheet-covered specimens has been tested to determine the effects of the combined loading of bending plus transverse shear and to determine the influence of length on the failing load.

INVESTIGATION OF LENGTH EFFECT

Inasmuch as all the pure bending specimens had a length to diameter ratio of 2.0, it was necessary to determine whether the failing loads of these specimens were influenced by the length. The purpose of this investigation was, therefore, to determine the length to diameter ratio L/D at which the failing load becomes independent of length.

Eight specimens having L/D ratios of 1.2, 1.6, 2.0, and 2.6 were tested, four in which the frame spacing was 2.0 inches and four in which the spacing was 4.0 inches. The spacing of the longitudinal and diameters was the same for all specimens, 2.62 and 20 inches, respectively.

The curves of unit strain as a function of the applied bending moment are shown in figures 7 to 11. The test procedure and method of strain measurements have been described in detail in references 2 and 4.

The failing strain as a function of the L/D ratio is shown in figure 2. It is seen that the specimens having an L/D ratio of 2 and 2.5 failed at approximately the same strain. As the L/D ratio decreases the failing strain increases somewhat and at an L/D ratio of 1.2 the increase is 12 and 23 percent for the 4- and 2-inch frame spacings, respectively. From these test results it can be concluded that the failing loads of the pure bending specimens used in the previous experiments were not influenced by the length.

INVESTIGATION OF COMBINED BENDING AND SHEAR

Description of Testing Apparatus

The testing machine, as shown in figure 71, consists essentially of four separate elements - the bed, the fixed end plate, the movable carriage, and the loading beam. The bed is 12½ feet long, 3½ feet wide, and is constructed of heavy I-beams. To one end is rigidly bolted the fixed end plate. The loading beam is a built-up box section 6 feet long - although this can be extended by splicing on additional lengths. Bolted to one end is an end plate similar to the fixed end plate. The loading beam is freely suspended in a cage or carriage which itself is free to move along the bed. The beam is suspended from two knife edges at the top of the carriage and is balanced by counterweights. With this arrangement the weight of the loading beam and end plate will not be carried by the specimen, but will always be just balanced by the counterweights.

One end of the specimen is bolted to the fixed end plate - the other to the end plate on the loading beam. A vertical upload is then applied at any desired point on the loading beam by means of a screw jack. The load is indicated by a dial gage mounted on a dynamometer which was previously calibrated in a 150,000-pound Tinius Olsen testing machine.

Method of Strain Measurement

Resistance strain gages were used to measure the strain at desired points. Essentially, this method measures the change in resistance of a wire element cemented to the specimen. The gages were calibrated by cementing a wire element of known length to a test bar and measuring the change in resistance for a known unit strain.

The gages were mounted on the neutral axis of the stiffeners and were also mounted in pairs - one on each side of the stiffener - so that the mean reading would be unaffected by bending. The resistances were measured with a Wheatstone bridge, the voltage being supplied by two dry cells. The gages were cut in and out of the bridge circuit through a mercury switch arrangement, thus avoiding large contact resistances.

During the early part of the investigation, a 3-inch-gage length was used. However, it was found that this small gage length was apparently quite sensitive to small local deformations. Also it seemed probable that the behavior of the gage was unduly influenced by the conditions at the juncture points where the leads were soldered. It was therefore decided to adopt a 16-inch-gage length. Although this gage did not permit measuring the strain at as many different points as the 3-inch gage, it yielded more consistent results.

Test Prodedure

The specimen was mounted in the testing machine and the loading jack placed at the proper position to give the desired moment arm. The load was then applied in increments, the jack-dial reading and the strain-gage resistances being recorded for each increment. Also, over-all strain measurements were made at the top, bottom, and sides by means of dial gages. The over-all measurements on the sides were useful in detecting loading eccentricities. The behavior of the specimen was carefully noted as the test progressed; initial and subsequent buckles were observed and their positions, extent, and corresponding loads recorded.

Combined Loadings

In the pure bending tests, the problem of determining the maximum strain associated with the failing load was relatively simple because of the uniform loading over the length of the specimen. However, when the test specimen is subjected to combined bending and transverse shear the bending moment and the corresponding strains vary over the length of the specimen. Hence, the method of measuring, over the specimen length, a maximum mean strain at failure and assigning this measured strain to the failing load can no longer be applied. The variation of strain and the end conditions of the specimen make it particularly difficult to associate a particular strain with the failing load. The maximum moment occurs at the fixed end; however, this maximum moment and the corresponding strain cannot be considered as being a measure of the ultimate strength of the cylinder because of the fixed-end conditions.

There are two possible methods of presenting the test data. One method would be to measure the strain at a number

of points along the length of the specimen, which could then be presented as the existing strain condition along the length of the specimen when failure occurred. Another method would be to measure the failing strain at the point at which the first buckle appears during the loading process and to consider this to be indicative of the maximum strain to which the specimen can be subjected before failure occurs.

Both methods have certain disadvantages. The first does not lend itself readily to application in practical design. It is customary, for maximum structural efficiency, to design a reinforced cylindrical structure in such a manner as to keep the stress nearly constant even though the bending moment is variable. This is usually accomplished through the geometrical taper of the section and also by tapering the effective bending material. For this reason, the designer is mostly interested in a single value of the allowable stress or strain for calculating his margins of safety. From this point of view the second method is preferable. However, the main objection here is that the validity of associating the strain condition at a point with the failure of the cylinder may be questionable.

This latter method has also one other advantage; namely, for the pure bending failure a parameter has been derived which appears to be satisfactory. In determining the influence of the transverse shear on the maximum strain at failure, it is convenient to have a single value of the strain for combined bending and shear to compare with the strain value for pure bending.

In view of the above, the following method of presentation was adopted. It was found that the initial buckle or buckles appeared over a relatively short range, being on the average about 14 inches from the fixed end for the 32-inch-diameter cylinders and 10 inches for the 20-inch-diameter cylinders. For this reason the strain at failure was considered to be the average of the strain measurements obtained with the 3-inch-wire gage over a distance extending from about 4 to 24 inches from the fixed end, or the strain measurements of the 16-inch-wire gage which extended from 6 to 22 inches from the fixed end.

The type of failure in combined bending and shear differs particularly in one respect from that of the pure bending failure. For the combined loading the failure is, in many cases, gradual - that is, the buckles, although extending

over several frames and longitudinals, slowly increase in depth and size with increasing load until a sudden collapse occurs with a marked increase in depth and size of the buckle or buckles. In the case of the pure bending specimens, the original cross-sectional shape is closely maintained until failure occurs - the failure being particularly violent and accompanied by large and deep buckles with as much as a two-thirds drop in the applied load.

From a visual observation of the failure, it appeared, during the early part of the investigation, that failure of a specimen was either of the bending or the shear type; that is, the beginning of a failure was confined to either a region of maximum compression or a region of maximum shear. The rather definite separation of the regions of failure would indicate that when a failure occurred in the compression region the failure would be nearly independent of the applied shear and would depend primarily on the state of compression strain; whereas, when failure occurred in the region of maximum shear the failure would be independent of the state of compression strain and only depend on the applied shear.

The above observations were borne out by the experimental results. In figure 3 are plotted the ratios of the compressive strain ϵ/ϵ_o as a function of VR/M for a number of specimens of both 16-inch and 10-inch radius. In these expressions,

ϵ maximum compressive strain value (at the position previously described) for the combined loading condition

ϵ_o same strain value for pure bending

V applied shear

R radius of the specimen

M bending moment

Since $M = VL$, where L is the distance from the applied shear load to the point where the strain is measured, it is seen that the ratio VR/M is equivalent to R/L. This presentation is merely to indicate that when the failure occurs in the compression region the maximum compressive strain is independent of the applied shear. For these specimens, the applied shear load varied from 503 to 1450

pounds for the 16-inch-radius specimens and from 710 to 2310 pounds for the 10-inch specimens. The minimum moment arm L was limited by the length of the specimen.

In figure 4 are plotted ratios of ϵ/ϵ_o as a function of V/V_o for one series of tests, where V_o is the shear load which causes a pure shear failure and V is the shear load corresponding to ϵ. It is seen from this figure that specimens 116 and 117 failed at the same shear load but widely different values of ϵ (i. e., bending moment). The photographs (figs. 75 and 76) indicate that specimen 116 failed similtaneously by combined bending and shear, whereas specimen 117 failed in shear. It appears from these results that the interaction curve for this type of loading consisted essentially of two perpendicular straight lines. The photographs (figs. 72 to 74) also show types of failure.

The strains at failure, for the specimens which failed by bending, have been compared with the parameter obtained for the pure bending failures and are shown in figure 5. The test values follow the same trend as the curve for the pure bending failures; however, the experimental scatter is somewhat greater than for the pure bending specimens. It is felt that this scatter is primarily due to the following:

(a) The accuracy of the strain measurements depends on the reliability of the electric strain gages and an accuracy greater than 5.0 percent is not to be expected.

(b) Adopting a strain value at a constant distance from the fixed end as being indicative of strain at failure also leads to some inaccuracies.

A number of curves showing the variation of strain in the longitudinal direction are shown in figures 67 to 69. The distribution of normal stresses is shown for a number of specimens in figures 64 to 66. In table I the experimental and computed data are given for each specimen. The curves of strain against applied shear are shown in figures 12 to 63. The strain-gage position and the stringer number is indicated in tabular form for each set of curves. The method of numbering the stringers is indicated in figure 6.

In all specimens the longitudinal stringers were S_1 and the frames F_5 with the exception of specimens 126 and 127 in which the longitudinals were S_2 and the frames F_5. The cross-sectional dimensions of the longitudinals

and frames are shown in figure 1. The variations of the sectional parameters, ρ and I_s, for the longitudinal S_2 with a variation in effective width are shown in figure 70.

CONCULUSIONS

The experimental results indicate that for combined bending and transverse shear, the failure is characterized by two distinct types of failure, namely, a bending or a shear failure. If the failure is bending, the failing strain can be predicted with sufficient accuracy by means of the pure bending parameter. However, it is still necessary to determine a shear failure parameter in order to ascertain when a bending or a shear failure will occur. Because of the limitation imposed by the specimen length on the maximum shear which could be applied, it was quite difficult to obtain shear failures. In the combined bending and torsion experiments additional data on shear failures will be obtained which may be sufficient to lead to a suitable shear parameter.

Guggenheim Aeronautical Laboratory,
California Institute of Technology,
Pasadena, Calif., June 10, 1942

REFERENCES

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TABLE I

Test	Long Spacing b	Frame Spacing d	Radius	Ultimate Applied Shear	Moment arm - to fixed end.	Strain at Failure	$\sqrt[4]{bd}$	P_x	P_y	$\sqrt[4]{(P_x P_y)^3}$	$\frac{\sqrt[4]{bd}}{\sqrt[4]{(P_x P_y)^3}}$	$\sqrt[4]{\frac{(P_x P_y)^3}{bd}} \frac{1}{R}$	Type of Failure
75	5.06	8"	16"	1067#	128"	.00161	2.52	.1167	.02335	.0119	3390	2.95x10-4	Gen. Instability
76	"	"	"	1050	116"	.00153	"	.1167	.02335	.0119	3390	2.95 "	"
77	"	"	"	1450	92"	.001516	"	.1167	.02335	.0119	3390	2.95 "	"
78	"	"	"	995	116	.00151	"	.1167	.02335	.0119	3390	2.95 "	"
79	"	2"	"	2000	128	.00270	1.785	.1172	.02836	.0138	2070	4.83 "	"
80	"	"	"	1850	107	.00211	"	.1170	.02836	.0138	2070	4.83 "	"
81	"	"	"	2175	86	.00242	"	.1170	.02836	.0138	2070	4.83 "	"
82	"	"	"	2070	64	.00189	"	.1170	.02836	.0138	2070	4.83 "	"
83	10.12	4	"	563	128	.00154	2.52	.1117	.02677	.0128	3150	3.18 "	"
84	"	2	"	825	128	.00212	2.12	.1129	.02836	.0135	2510	3.98 "	"
85	"	"	"	915	104	.00196	"	.1127	.02856	.0134	2550	3.95 "	"
86	"	"	"	915	80	.00218	"	.1130	.02836	.0135	2510	3.98 "	"
87	"	"	"	929	66.5	.00213	"	.1130	.02836	.0135	2510	3.98 "	"
88	5.06	4	"	2115	80	.00216	"	.1170	.02677	.0132	2570	3.89 "	"
89	"	"	"	1910	66.5	.00187	"	.1169	.02677	.0132	2570	3.89 "	"
90	"	"	"	1212	128	.00209	"	.1170	.02677	.0132	2570	3.89 "	"

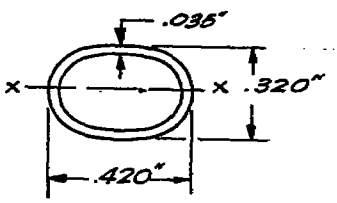
Note: Sheet thickness for all specimens = .010"

TABLE I (CONT'D)

Test	Long Spacing b	Frame Spacing d	Radius	Ultimate Applied Shear	Moment arm - to fixed end	Strain at Failure	$\sqrt[4]{bd}$	P_x	P_y	$\sqrt[4]{P_x P_y^3}$	$\frac{\sqrt[4]{bd}}{\sqrt[4]{P_x P_y^3}} R$	$\sqrt[4]{\frac{P_x P_y^3}{bd}} \frac{1}{E}$	Type of Failure
91	5.06"	4"	16"	1331#	116"	.00194	2.12	.1169	.02677	.0132	2570	3.89x10-4	Gen. Instability
92	"	8	"	503	194	.00152	2.52	.1167	.02335	.0119	3390	2.95 "	"
93	"	"	"	1353	74	.00135	"	.1166	"	.0119	3390	2.95 "	"
94	2.62	"	10"	980	114	.00174	-	-	-	-	-	- -	Panel instability
95	"	"	"	1283	88.4	.00290	2.14	.1165	.02335	.0119	1800	5.55 "	Gen. Instability
96	"	"	"	1711	66	.00265	"	.1166	"	.0119	1800	5.55 "	"
97	"	"	"	2900	40.5	.00207	-	-	-	-	-	-	Shear
98	"	4	"	1310	113	.00340	1.80	.1165	.02677	.0132	1360	7.35 "	Gen. Instability
99	"	"	"	1660	88.4	.00330	"	.1164	"	.0132	1360	7.35 "	"
100	"	"	"	2250	65	.00270	-	-	-	-	-	-	Bending-Shear
101	"	"	"	3300	40.5	.00260	-	-	-	-	-	-	Shear
102	2.53	2	16	2480	137	.00282	1.50	.1160	.02836	.0137	1750	5.71 "	Gen. Instability
103	"	"	"	2860	112.4	.00277	"	.1160	"	.0137	1750	5.71 "	"
104	"	"	"	4200	89	.00343	"	.1157	"	.0137	1760	5.71 "	"
105	"	"	"	4500	65.1	.00218	-	-	-	-	-	-	Shear
106	2.62	"	10	1490	113	.00396	1.51	.1161	.02836	.0138	1090	9.17 "	Gen. Instability
107	"	"	"	2760	65	.00460	"	.1160	"	.0137	1100	9.10 "	"
108	"	"	"	4270	40.5	.00398	"	.1161	"	.0138	1090	9.17 "	"
109	5.24	8	"	450	113	.00186	2.54	.1162	.02335	.0119	2130	4.69 "	"

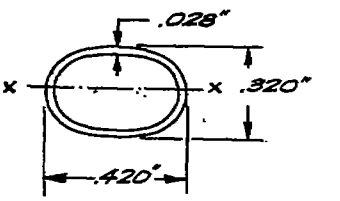
TABLE I (CONT'D)

Test	Long Spacing b	Frame Spacing d	Radius	Ultimate Applied Shear	Moment arm - to fixed end.	Strain at Failure	$\sqrt[4]{\frac{b}{t}}$	ρ_x	ρ_y	$\sqrt[4]{\rho_x \rho_y^3}$	$\sqrt[4]{\frac{b}{t}} \frac{R}{\sqrt{\rho_x \rho_y^3}}$	$\sqrt[4]{\frac{\rho_x \rho_y^3}{b}} R$	Type of Failure
110.	5.24"	8"	10"	1470	40.6	.00242	2.54	.1166	.02356	.0119	2130	4.69×10^{-4}	Gen. Instability
111	"	4	"	660	113	.00251	2.14	.1170	.02677	.0132	1620	6.17 "	"
112	"	"	"	1780	40.6	.00290	"	.1169	"	.0132	1620	6.17 "	"
113	"	"	"	930	88.4	.00265	"	.1169	"	.0132	1620	6.17 "	"
114	2.63	"	16"	1900	137	.00216	1.785	.1162	"	.0132	2160	4.63 "	"
115	"	"	"	2010	112.4	.00194	"	.1163	"	.0132	2160	4.63 "	"
116	"	"	"	2400	89	.00217	"	.1162	"	.0132	2160	4.63 "	"
117	"	"	"	2450	65.3	.00136	-	-	-	-	-	-	Shear
118	5.24	2	10	710	113	.00360	1.80	.1170	.02856	.0138	1300	7.69 "	Gen. Instability
119	"	"	"	990	88.4	.00364	"	.1170	"	.0138	1300	7.69 "	"
120	"	"	"	1460	65	.00353	"	.1169	"	.0138	1300	7.69 "	"
121	"	"	"	2310	40.6	.00374	"	.1170	"	.0138	1300	7.69 "	"
122	2.62	1	"	1665	113	.00525	1.27	.1169	.02801	.0136	930	10.76 "	"
123	"	"	"	2140	88.4	.00492	"	.1160	"	.0136	930	10.76 "	"
124	"	"	"	3220	65	.00481	"	.1160	"	.0136	930	10.76 "	"
125	"	"	"	5280	40.6	.00479	"	.1160	"	.0136	930	10.75 "	"
126	5.06	4	16	1600	109	.00207	2.12	.1150	.02677	.0131	2590	3.86 "	"
127	"	"	"	1940	89	.00192	"	.1150	"	.0131	2590	3.86 "	"
128	"	"	"	1445	137	.00208	"	.1150	"	.0131	2590	3.86 "	"
129	"	"	"	2340	68	.00206	"	.1150	"	.0131	2590	3.86 "	"



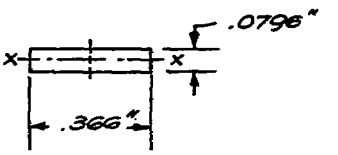
STIFFENER-S_2

AREA = .0368 I_{xx} = .000407



STIFFENER-S_1

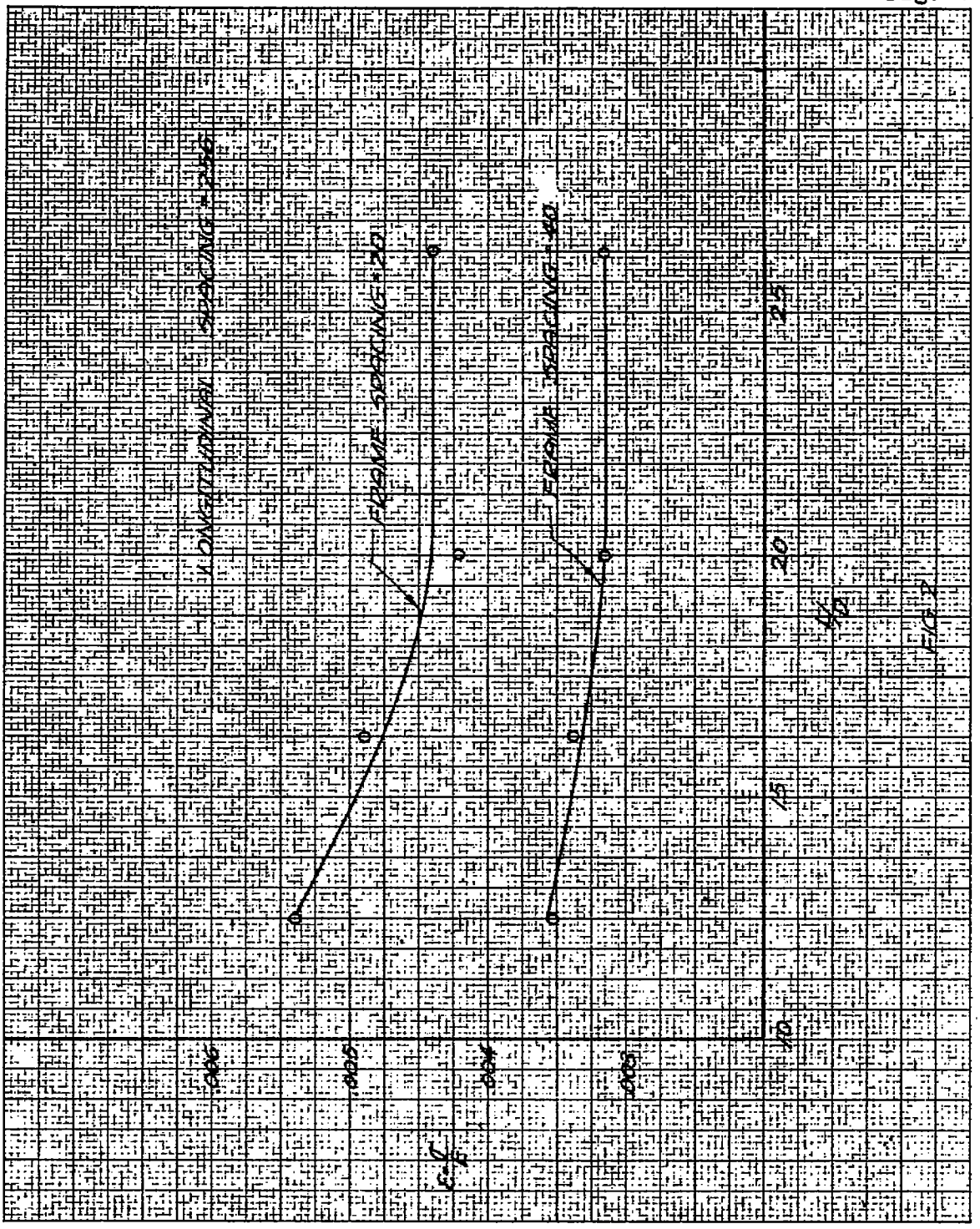
AREA = .0324 I_{xx} = .000374



FRAME-F_3

AREA = .0291 $I_{xx} = 1.537 \times 10^{-3}$

FIG. 1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.005
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.003
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765
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780
785
790
795
800
805
810
815
820
825
830
835
840
845
850
855
860
865
870
875
880
885
890
895
900
905
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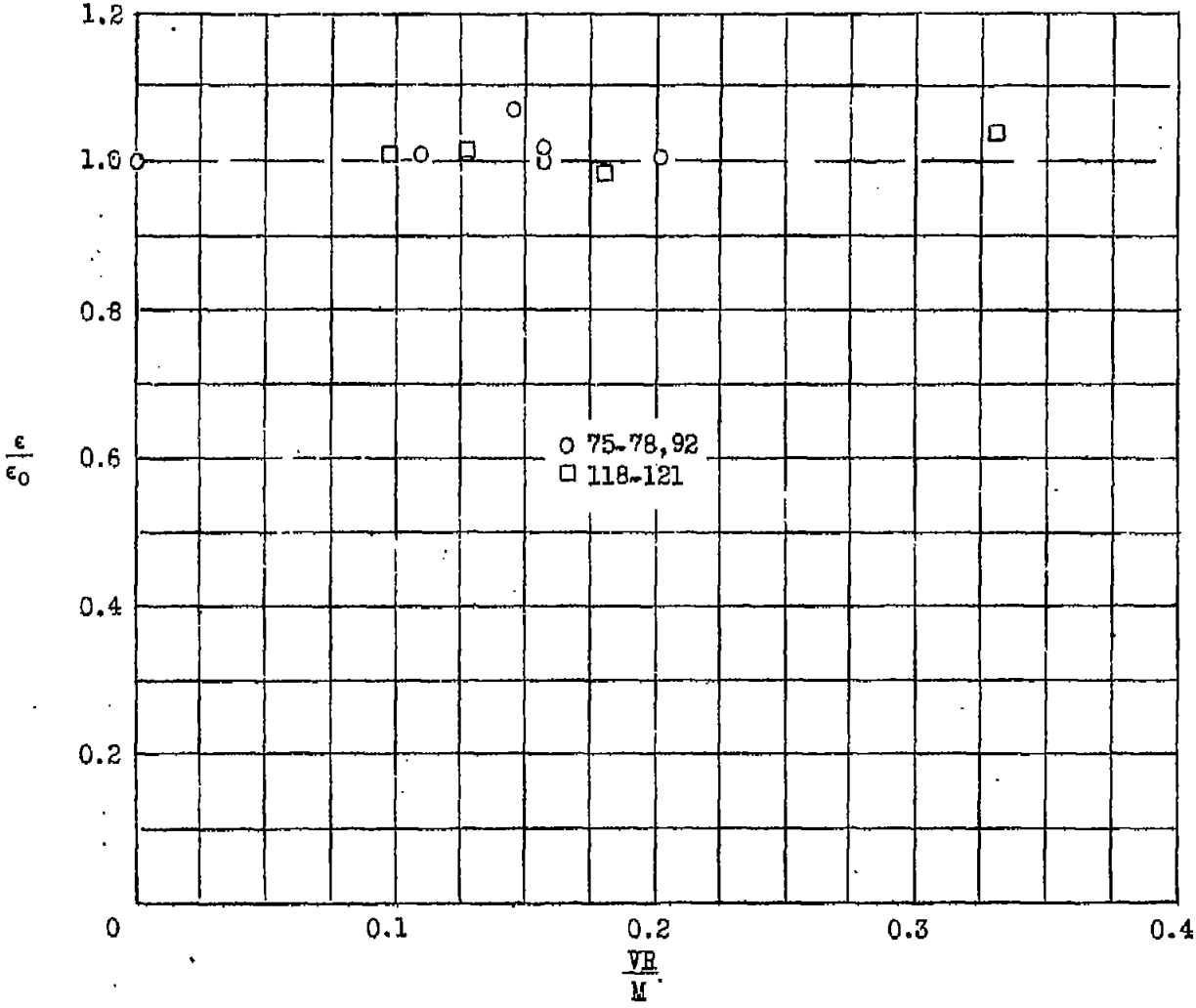
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1.0
0.8
0.6
0.4
0.2
0
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0.3
0.4
$\frac{\epsilon}{\epsilon_0}$
$\frac{VR}{M}$
o 75-78, 92
□ 118-121

Figure 3.

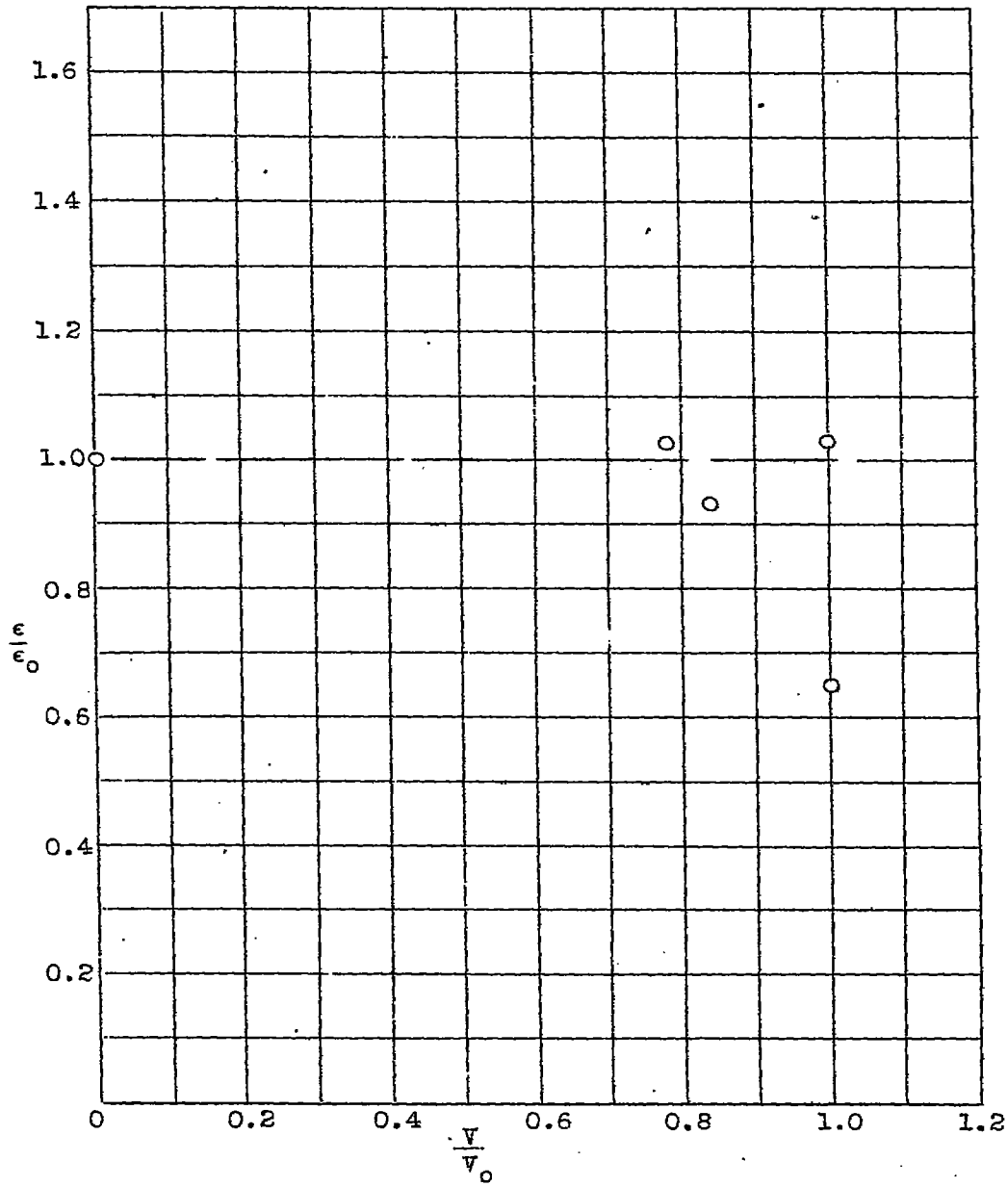

1.6
1.4
1.2
1.0
0.8
0.6
0.4
0.2
$\frac{\epsilon}{\epsilon_o}$
0
0.2
0.4
0.6
0.8
1.0
1.2
$\frac{V}{V_o}$

Figure 4

Fig. 5

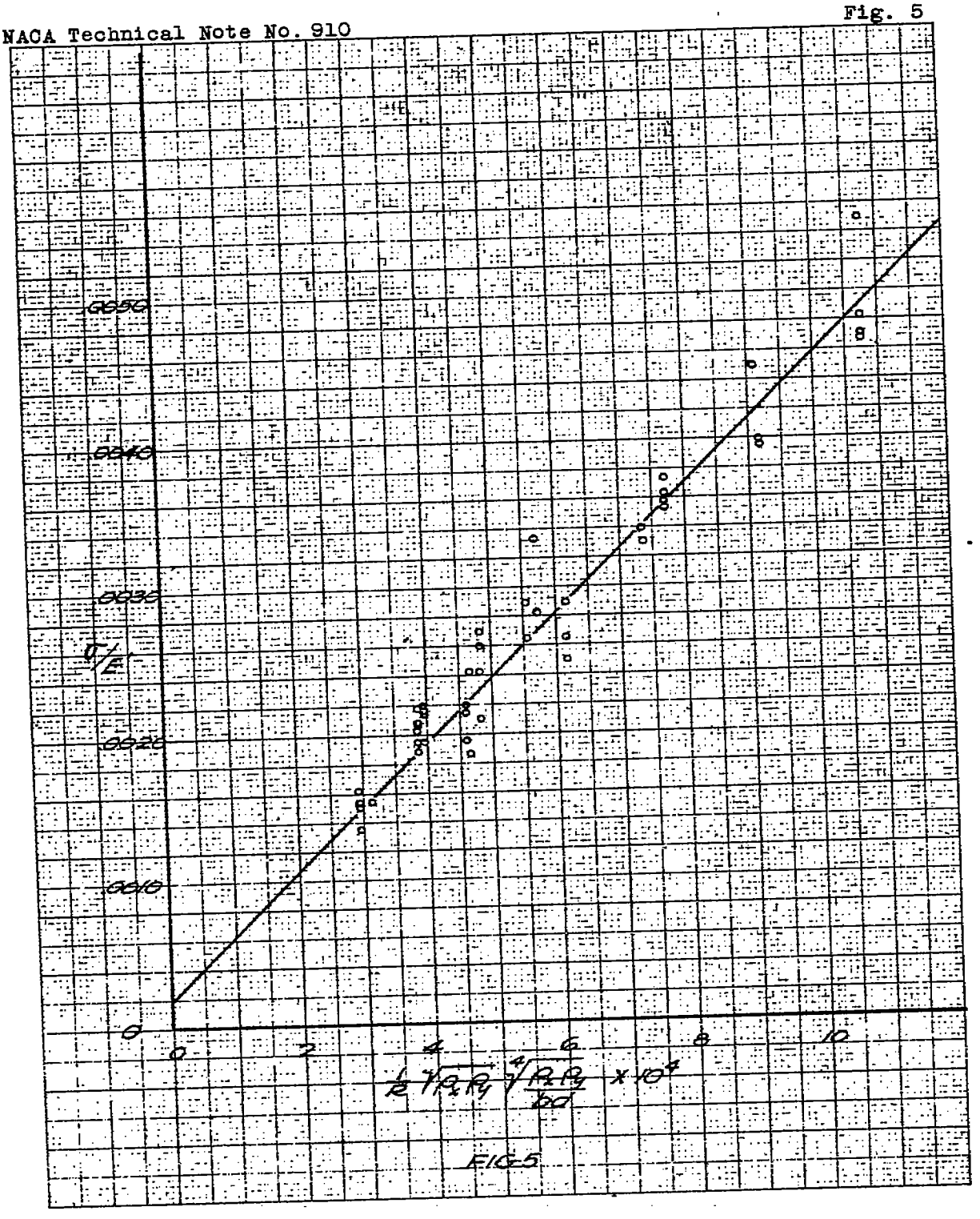

.0050
.0040
.0030
.0020
.0010
0
T/E
0
2
4
6
8
10
$\frac{1}{2}\sqrt{R_1 R_2}$
$\frac{1}{50}\sqrt{R_1 R_2}$
$\times 10^4$

FIG. 5

METHOD OF NUMBERING STIFFENERS

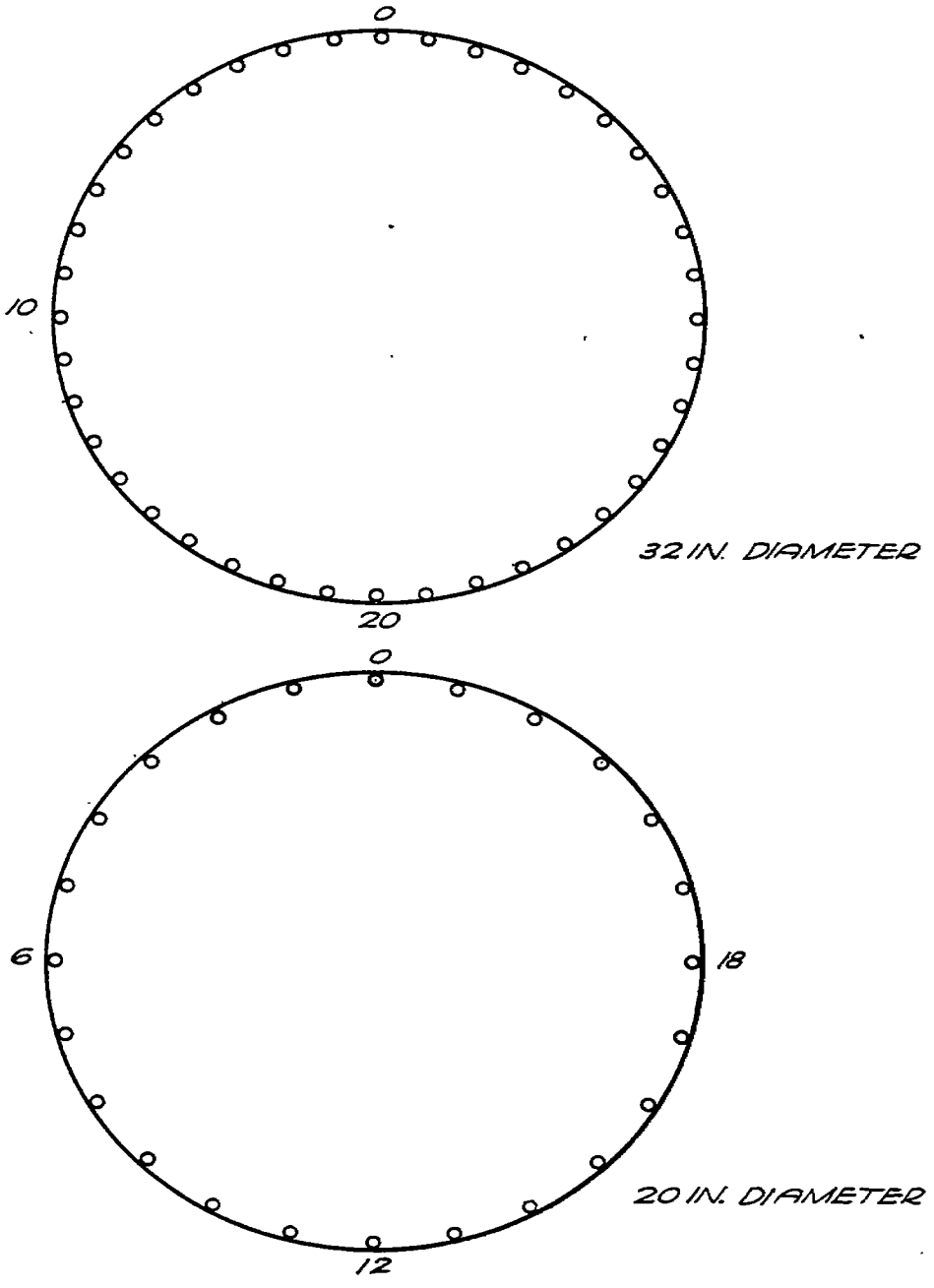


FIG 6

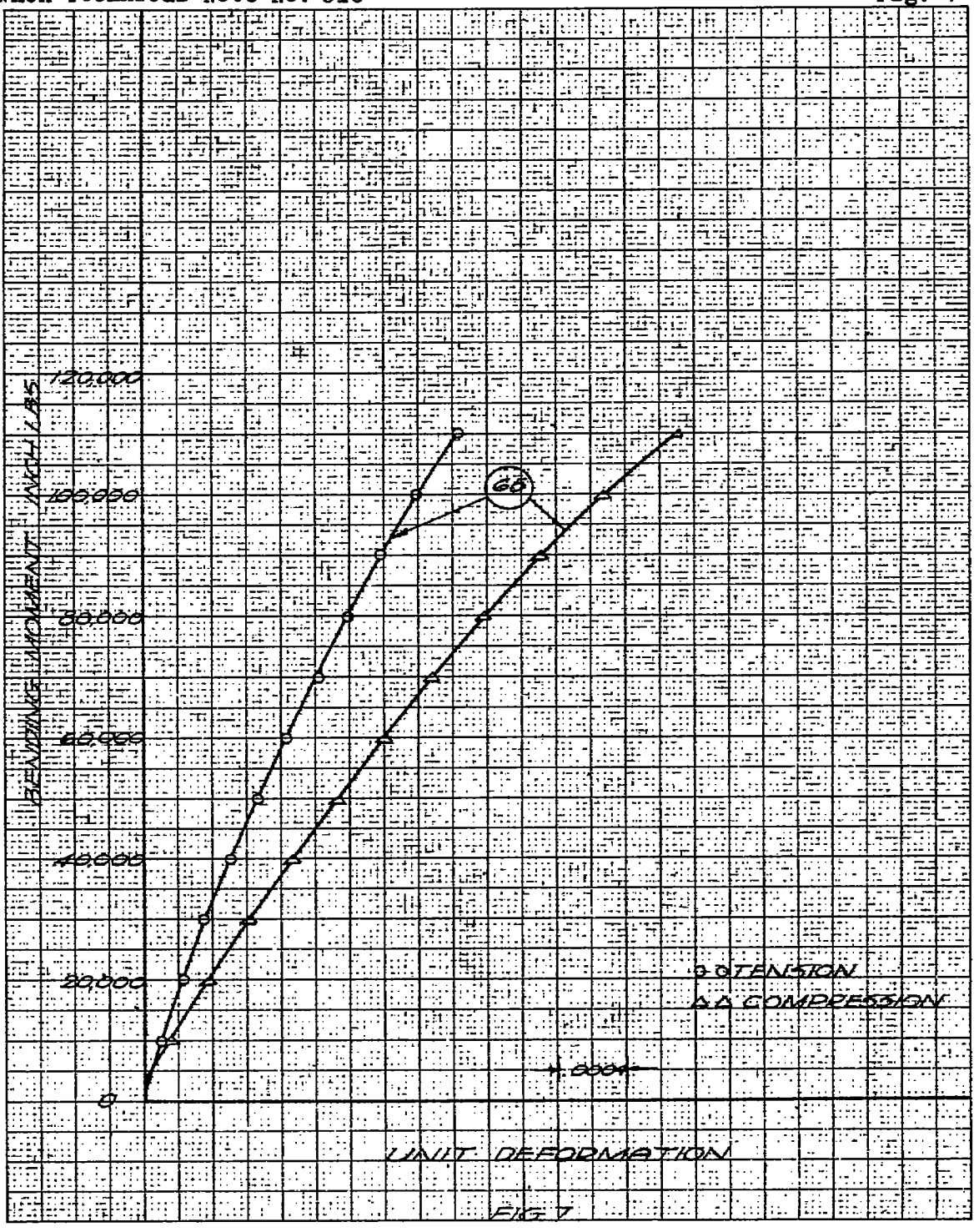

BENDING MOMENT INCH LBS.
120,000
100,000
80,000
60,000
40,000
20,000
0
68
TENSION
COMPRESSION
+.000-
UNIT DEFORMATION

FIG 7

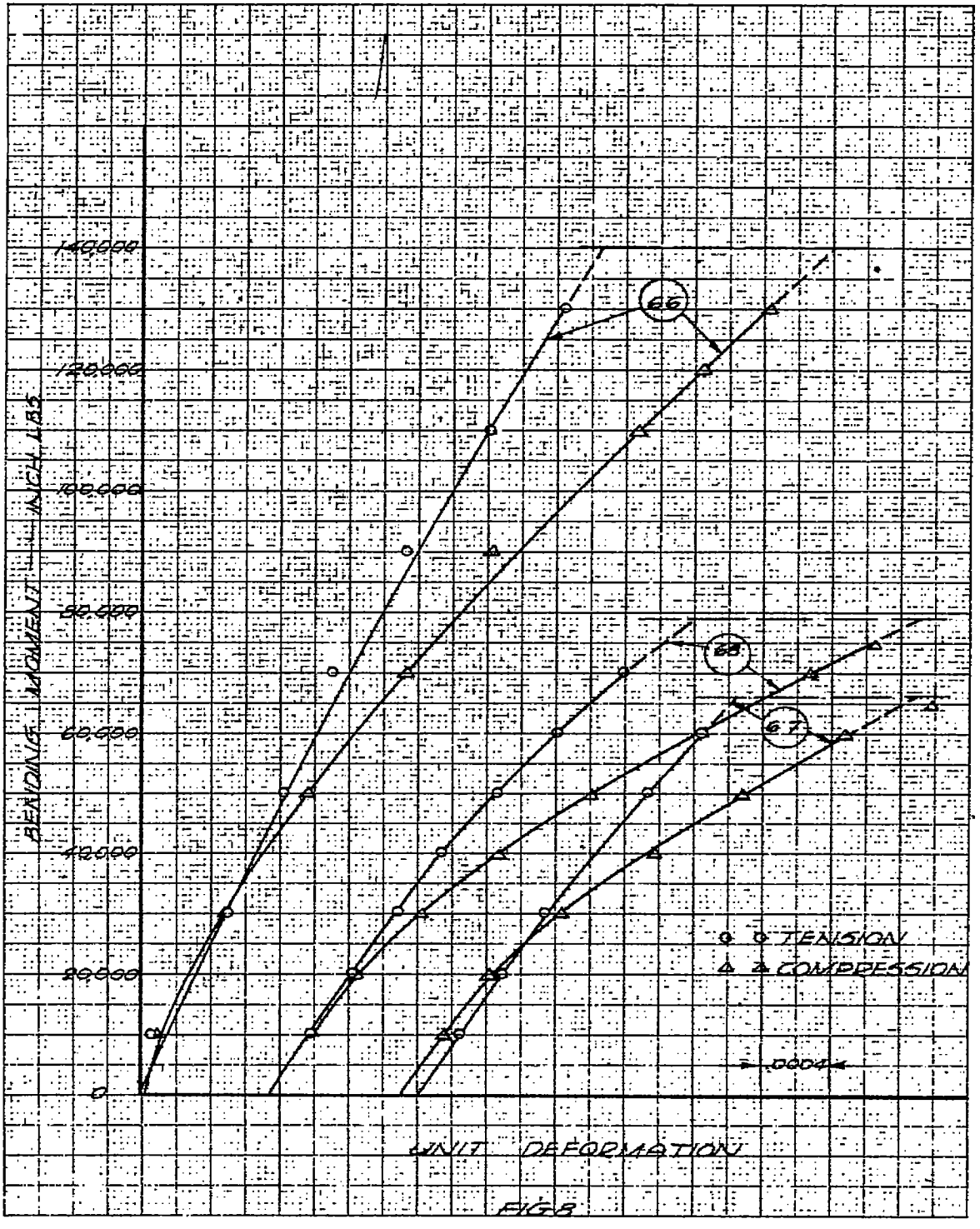

BENDING MOMENT INCH LBS
140000
120000
100000
80,000
60,000
40000
20,000
0
66
68
67
TENSION
COMPRESSION
.0004
UNIT DEFORMATION

FIG 8

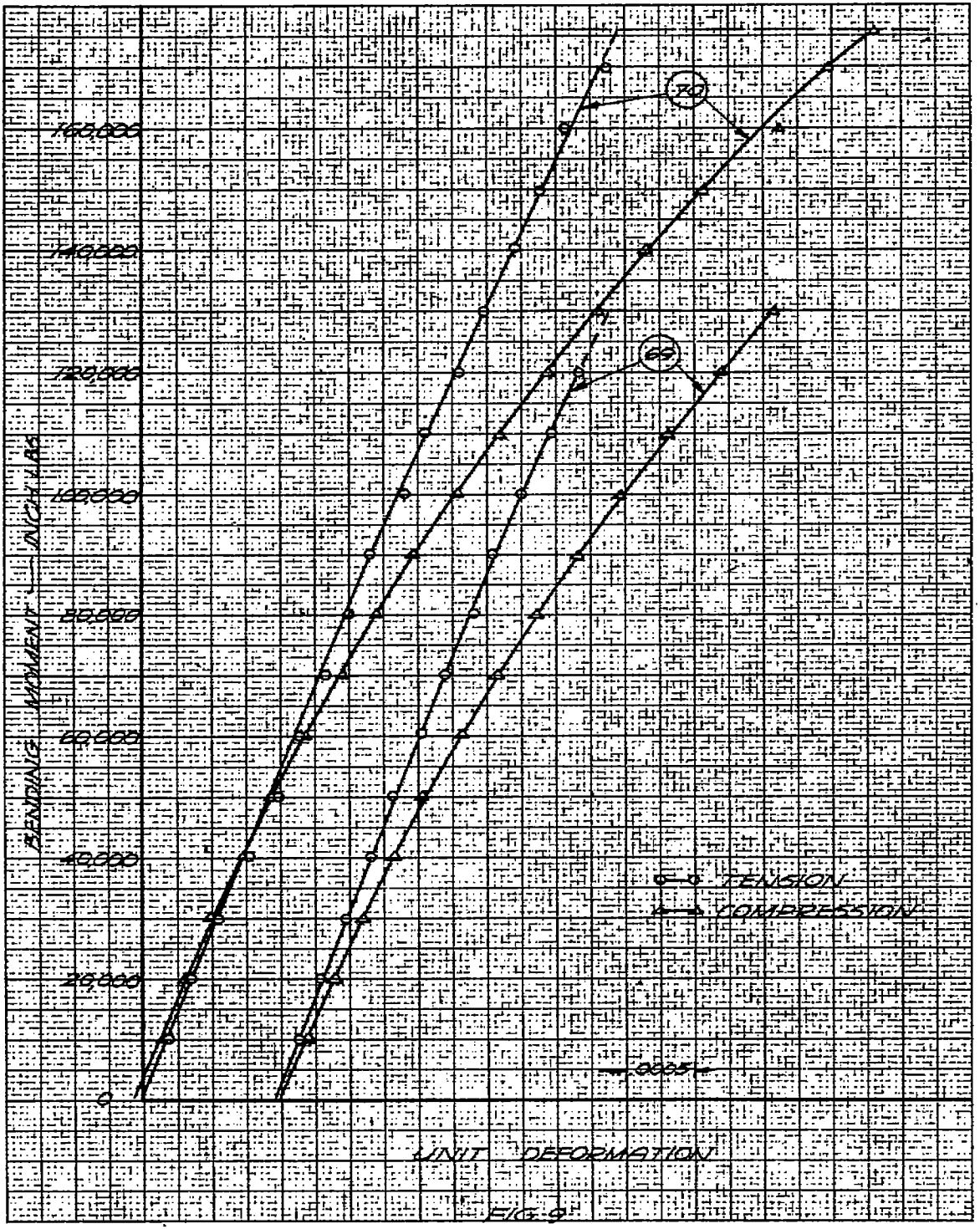

160,000
140,000
120,000
100,000
80,000
60,000
40,000
20,000
0
BENDING MOMENT - INCH-LBS
70
65
TENSION
COMPRESSION
.0005
UNIT DEFORMATION

FIG. 9

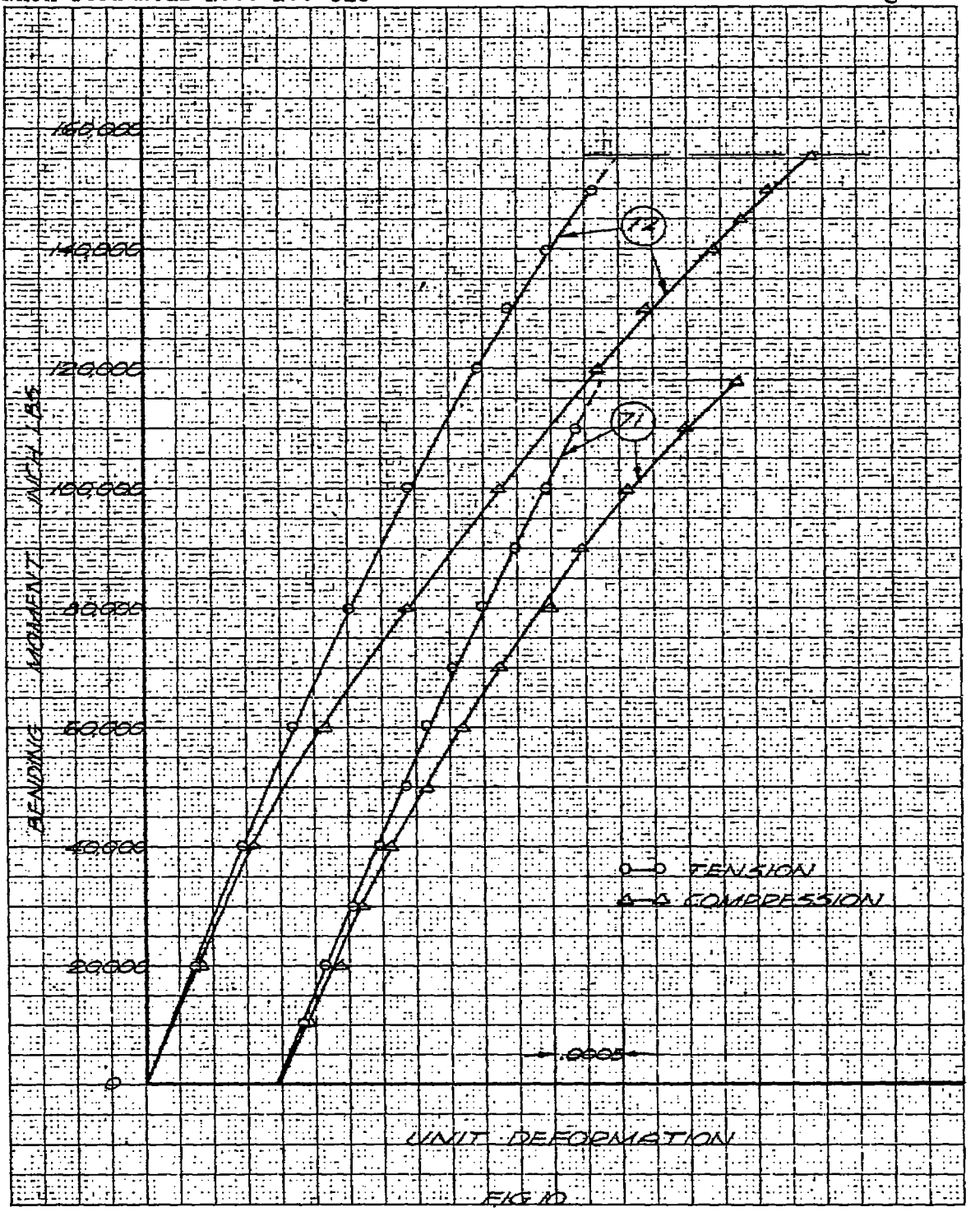

160,000
140,000
120,000
100,000
80,000
60,000
40,000
20,000
0
BENDING MOMENT IN LBS
12
71
0—0 TENSION
Δ—Δ COMPRESSION
.0005
UNIT DEFORMATION

FIG 10

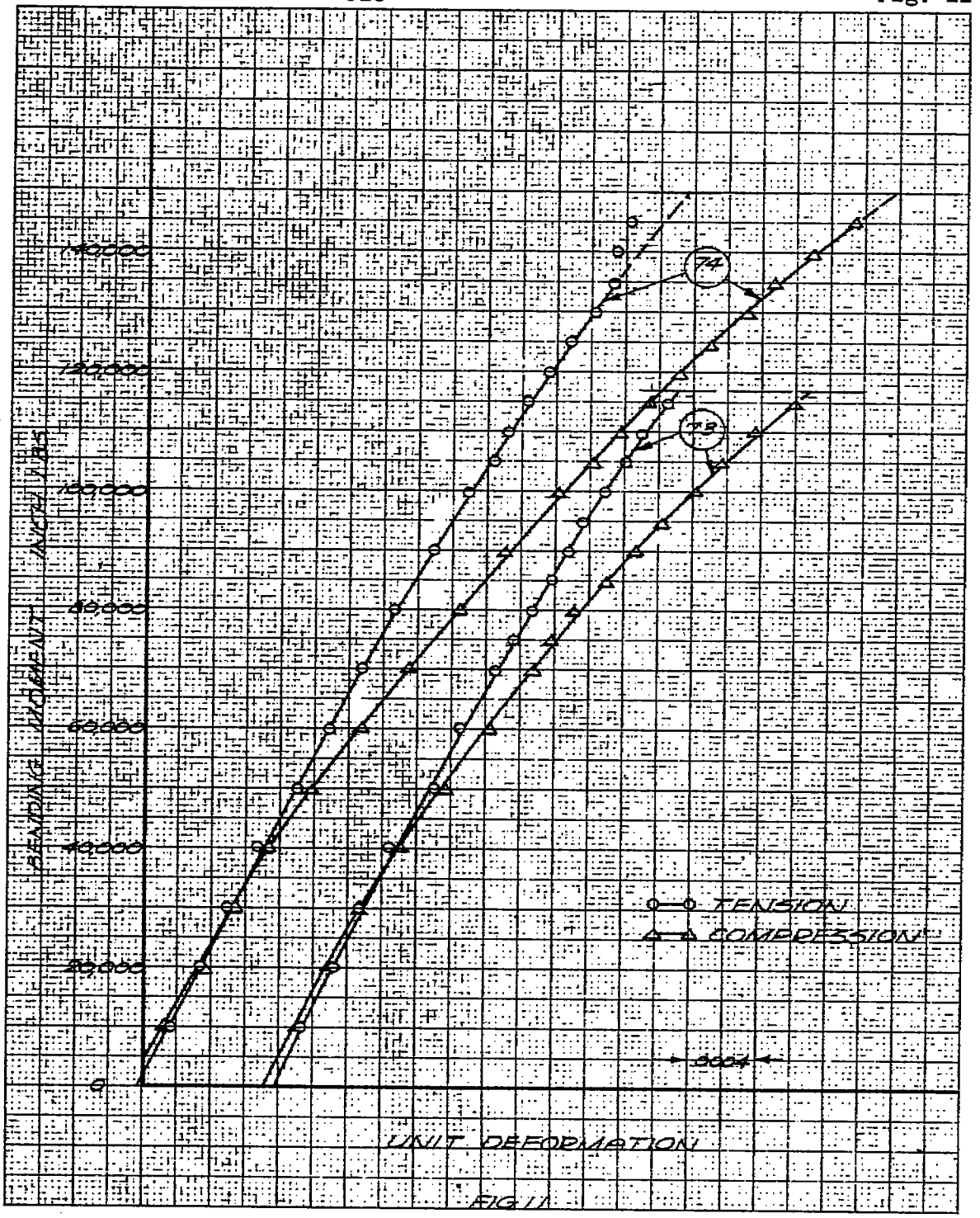
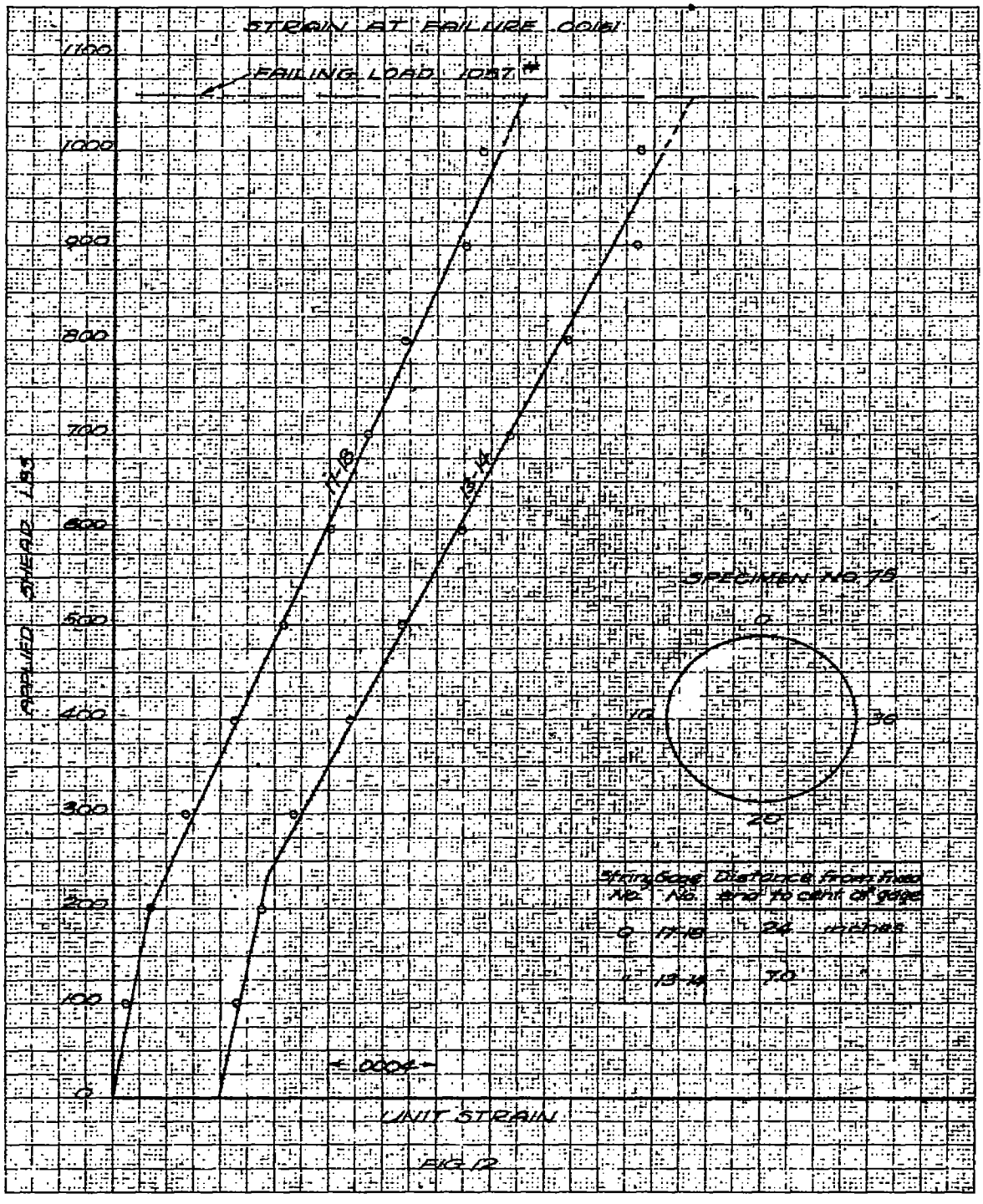

BENDING MOMENT INCH LBS
120000
90000
60000
30000
0
74
78
TENSION
COMPRESSION
.0004
UNIT DEFORMATION

FIG 11


STRAIN AT FAILURE .0015
FAILING LOAD 1097 #
1100
1000
900
800
700
600
500
400
300
200
100
0
LOAD IN POUNDS
SPECIMEN NO 75
0
10
30
20
String Gage No.
Distance from end to cent of gage
0 1716 24 inches
13 14 7.0
.0005
UNIT STRAIN
FIG 12

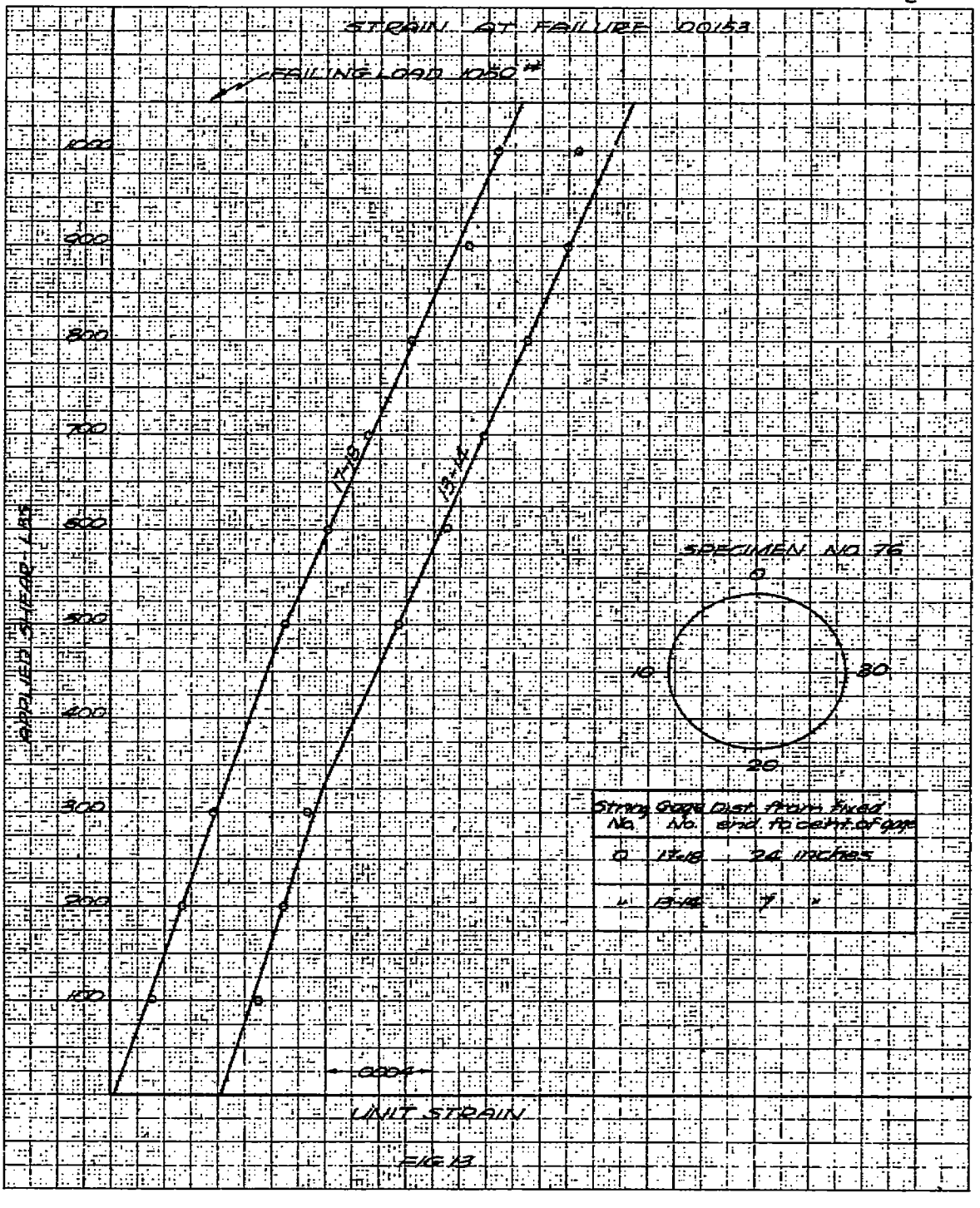
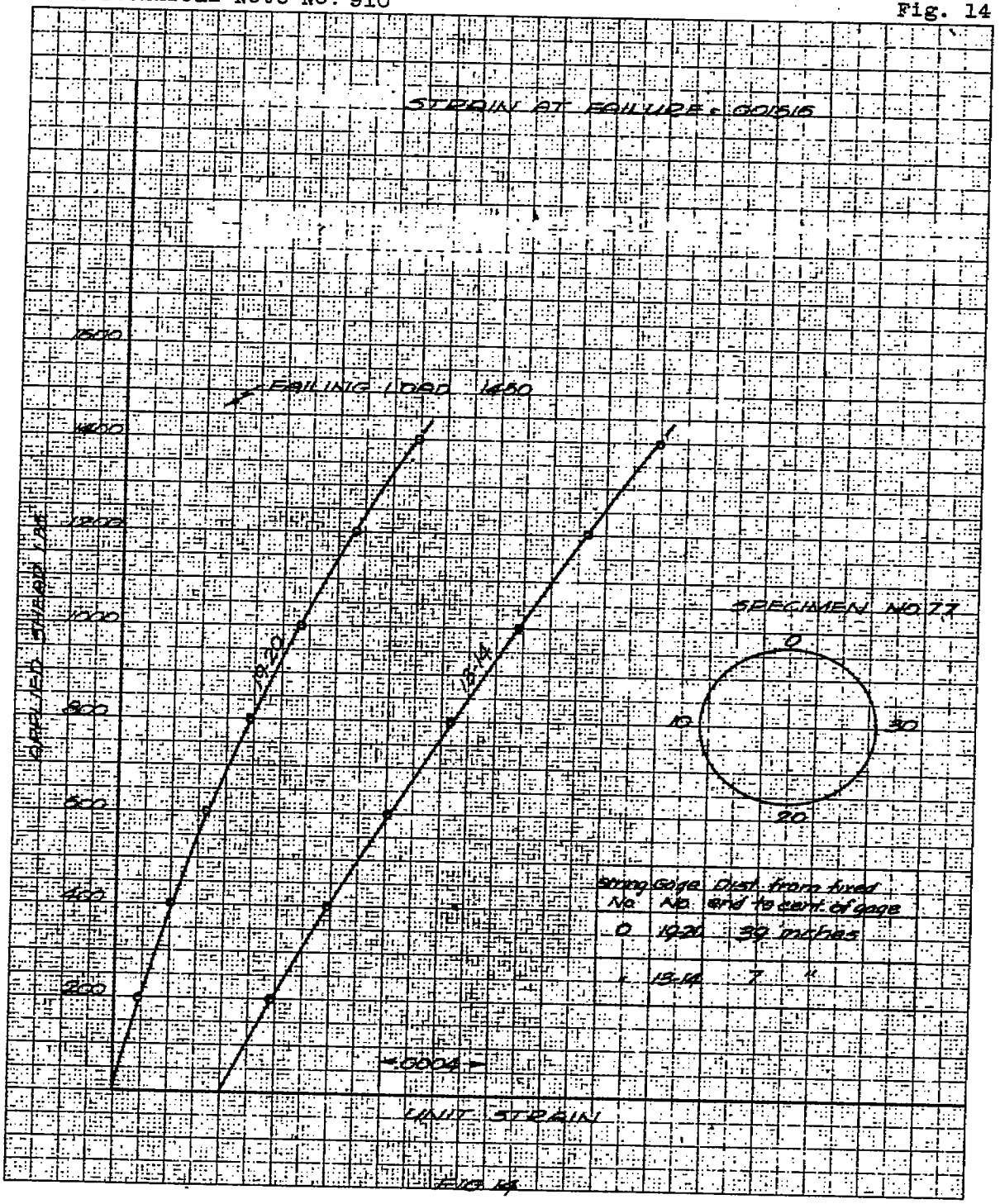
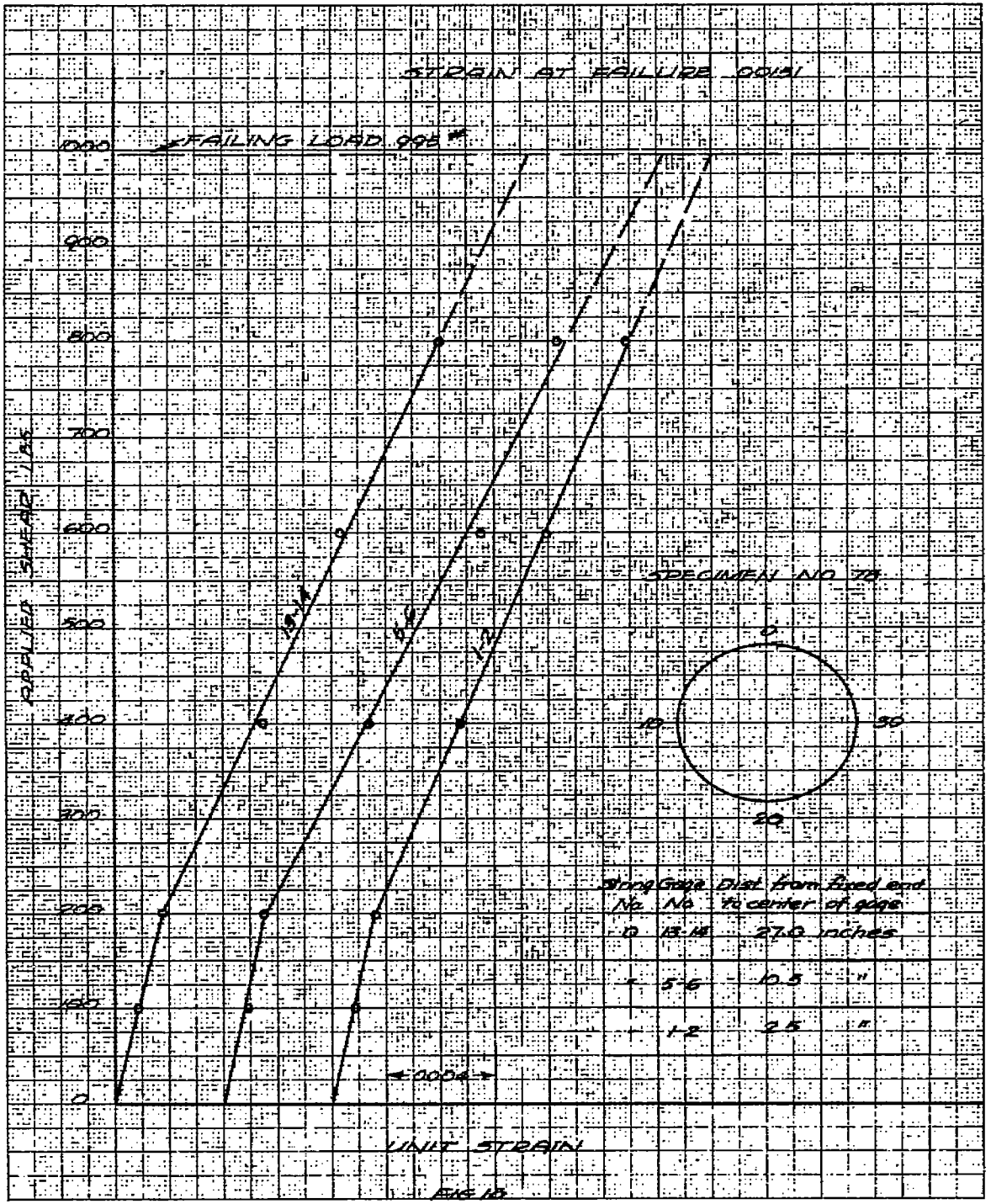

STRAIN AT FAILURE .00153
FAILING LOAD 1050 #
1000
900
800
700
600
500
400
300
200
100
APPLIED STRESS - LBS
.0004
UNIT STRAIN
FIG 13
SPECIMEN NO 76
0
10
30
20
String No.
Gage No.
Dist. from fixed end to center of gage
0
17-18
24 inches
4
3-4
7 "

Fig. 14


STRAIN AT FAILURE: .00516
FAILING LOAD 1650
1600
1400
1200
1000
800
600
400
200
APPLIED STRESS LBS
SPECIMEN NO 77
0
10
30
20
Strain Gage No
Gage No.
Dist. from fixed end to cent. of gage
0
19.20
39 inches
19.14
7
.0004
UNIT STRAIN


STRAIN AT FAILURE 0018
FAILING LOAD 998 #
1000
900
800
700
600
500
400
300
200
100
0
APPLIED SHEAR LBS
SPECIMEN NO 78
0
10
30
20
Strain Gage No
Dist from fixed end to center of gage
0 13-14 27.0 inches
1 5-6 10.5 "
2 1-2 2.5 "
.0004
UNIT STRAIN
AVG 10

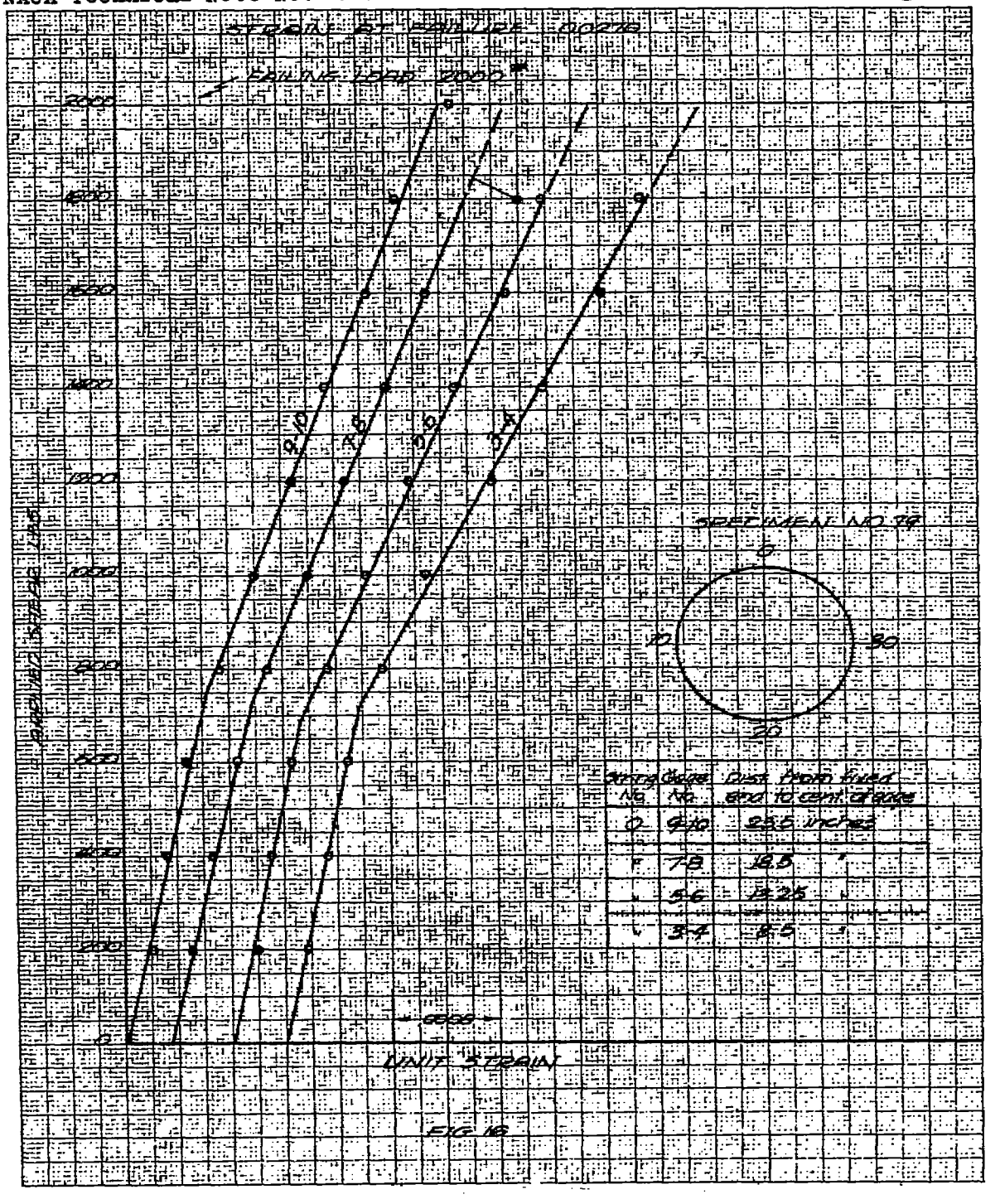

STRAIN AT FAILURE 0.0026
FAILING LOAD 2000 lb
2000
1600
1200
800
400
0
STRAIN
UNIT STRAIN
FIG 16
SPECIMEN NO 78
Gage No
Dist from fixed end to cent of gage
Q 9.10 25.5 inches
P 7.8 18.5 "
L 5.6 12.25 "
L' 3.4 6.5 "

Fig. 17

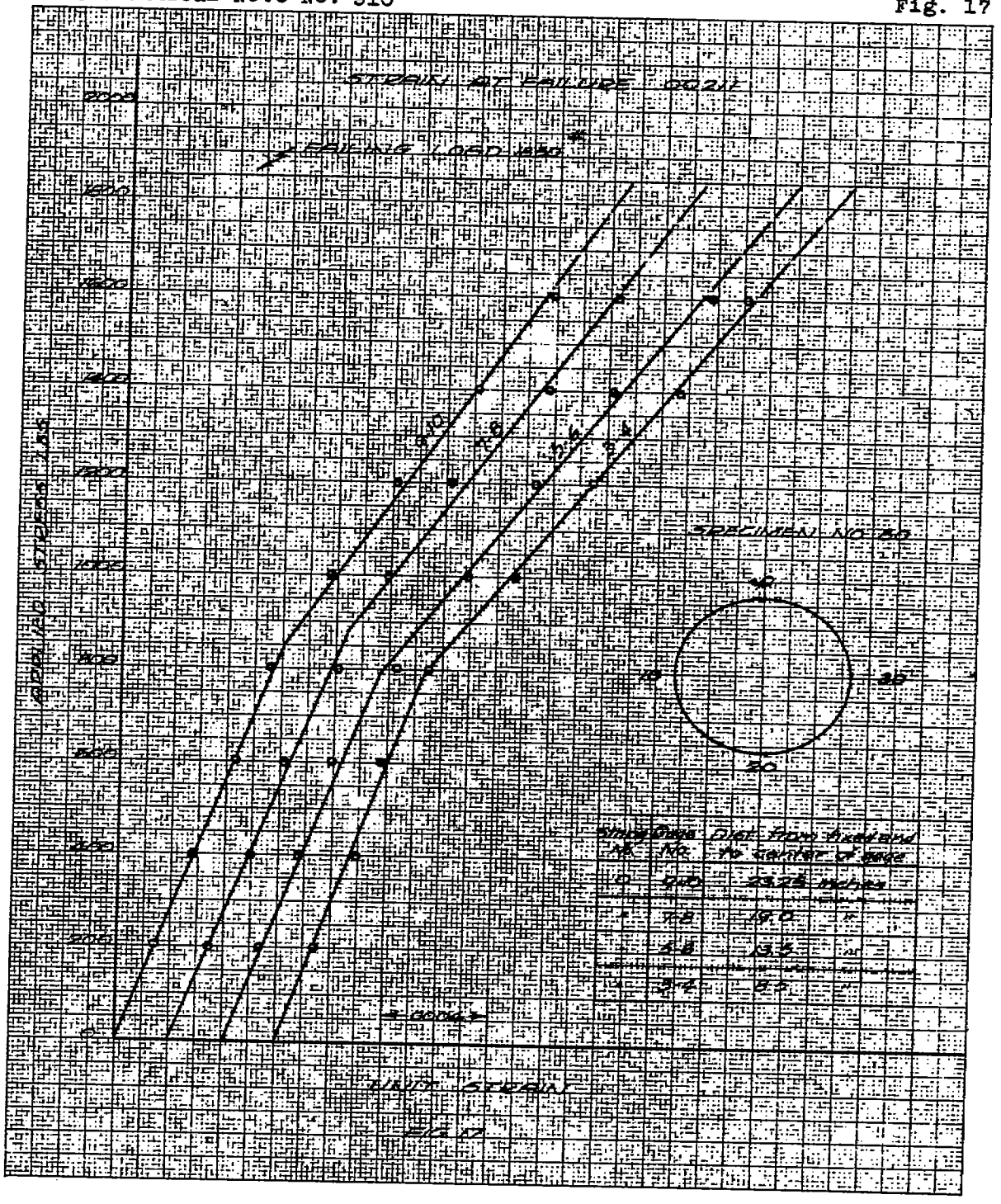
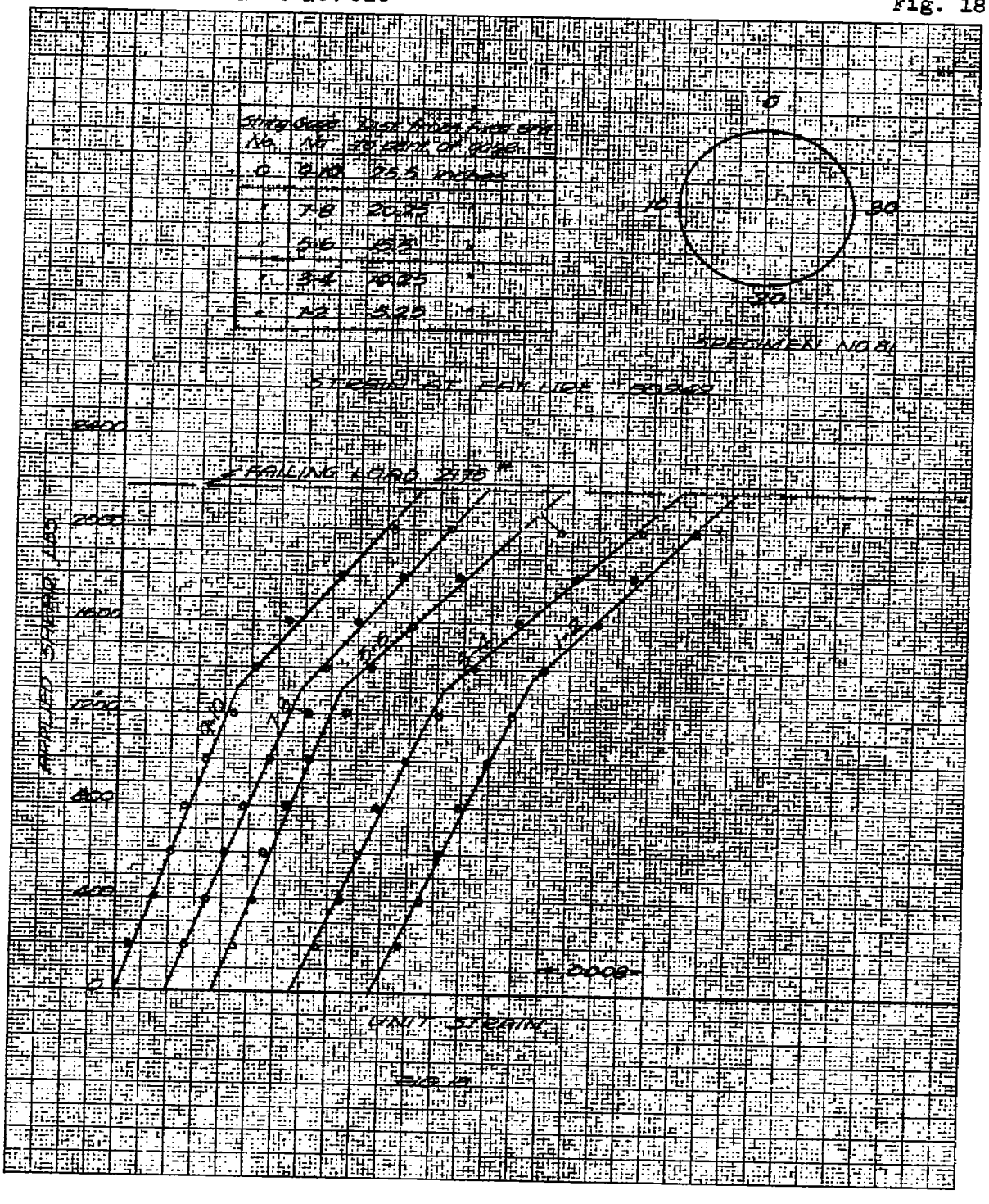

STRAIN AT FAILURE .0021
SPECIMEN NO 60
Strain Gage No.
Dist. from Bulkhead to center of gage
6 910 23.25 inches
7 8 19.0 "
4 13.5 "
3 4 9.5 "
2000
1800
1600
1400
1200
1000
800
600
400
200
0
.0005
.0010
.0015
.0020
.0025

Fig. 18


0
30
60
SPECIMEN NO. 9
STRESS-STRAIN CURVE
FAILING LOAD 2175 lb
2400
2000
1600
1200
800
400
0
2000
UNIT STRAIN

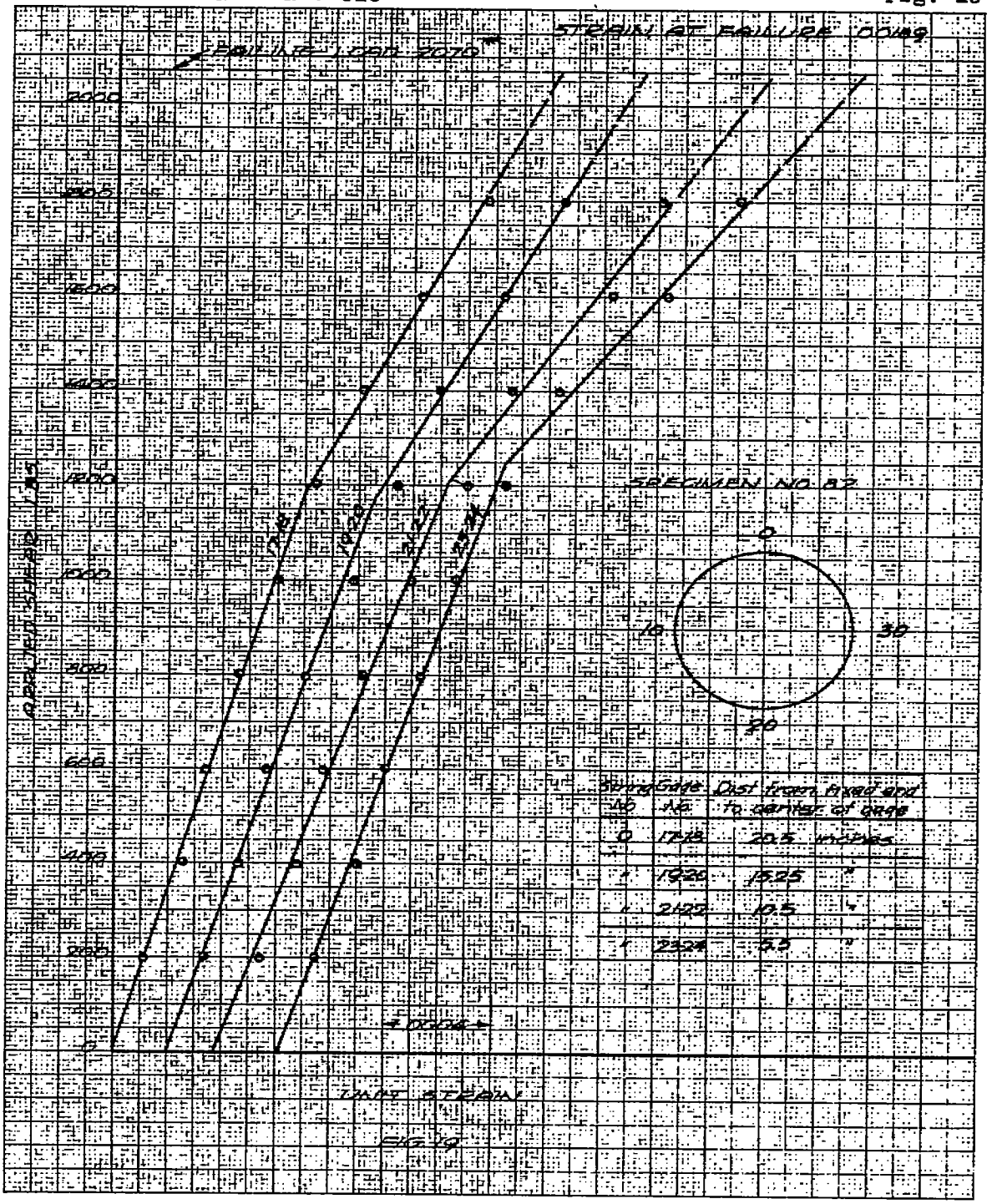
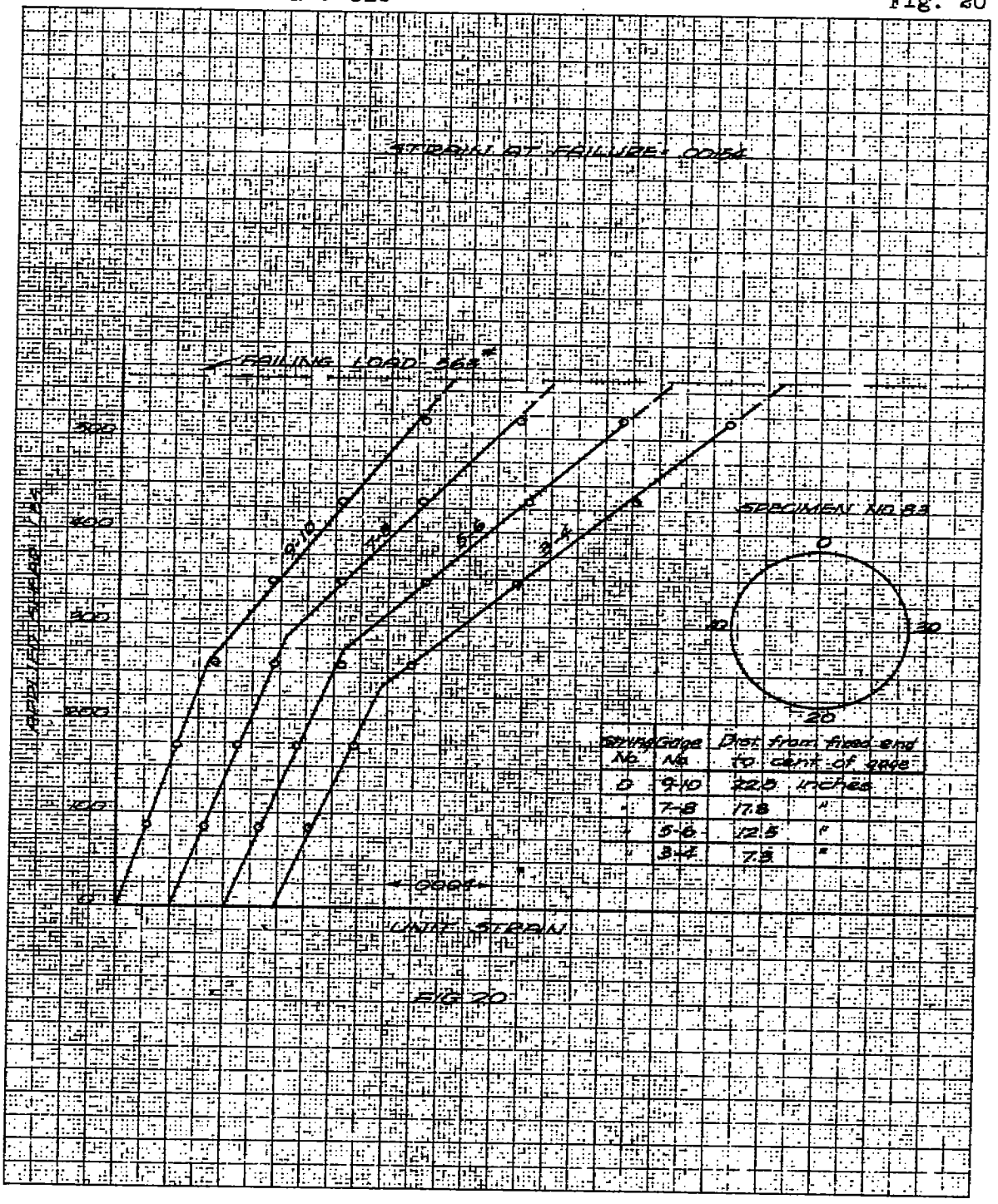

STRAIN AT FAILURE 0.0049
FAILURE 2070
SPECIMEN NO. A2
0
10
30
20
Strain Gage No.
Dist from fixed end to center of gage
0 1.25 2.25 inches
1 1.20 1.75 "
2 1.25 1.25 "
3 1.24 .75 "
STRESS, POUNDS PER SQUARE INCH
STRAIN
2000
1800
1600
1400
1200
1000
800
600
400
200
0
.0010
.0020
.0030
.0040
.0050
UNIT STRAIN
FIG. 19

Fig. 20


STRAIN AT FAILURE: .0054
FAILING LOAD 364#
SPECIMEN NO 93
0
10
20
String No.
Gage No.
Dist from fixed end to cent of gage
0 9-10 22.5 inches
1 7-8 17.5 "
2 5-6 12.5 "
3 3-4 7.5 "
.0054
UNIT STRAIN
FIG 20

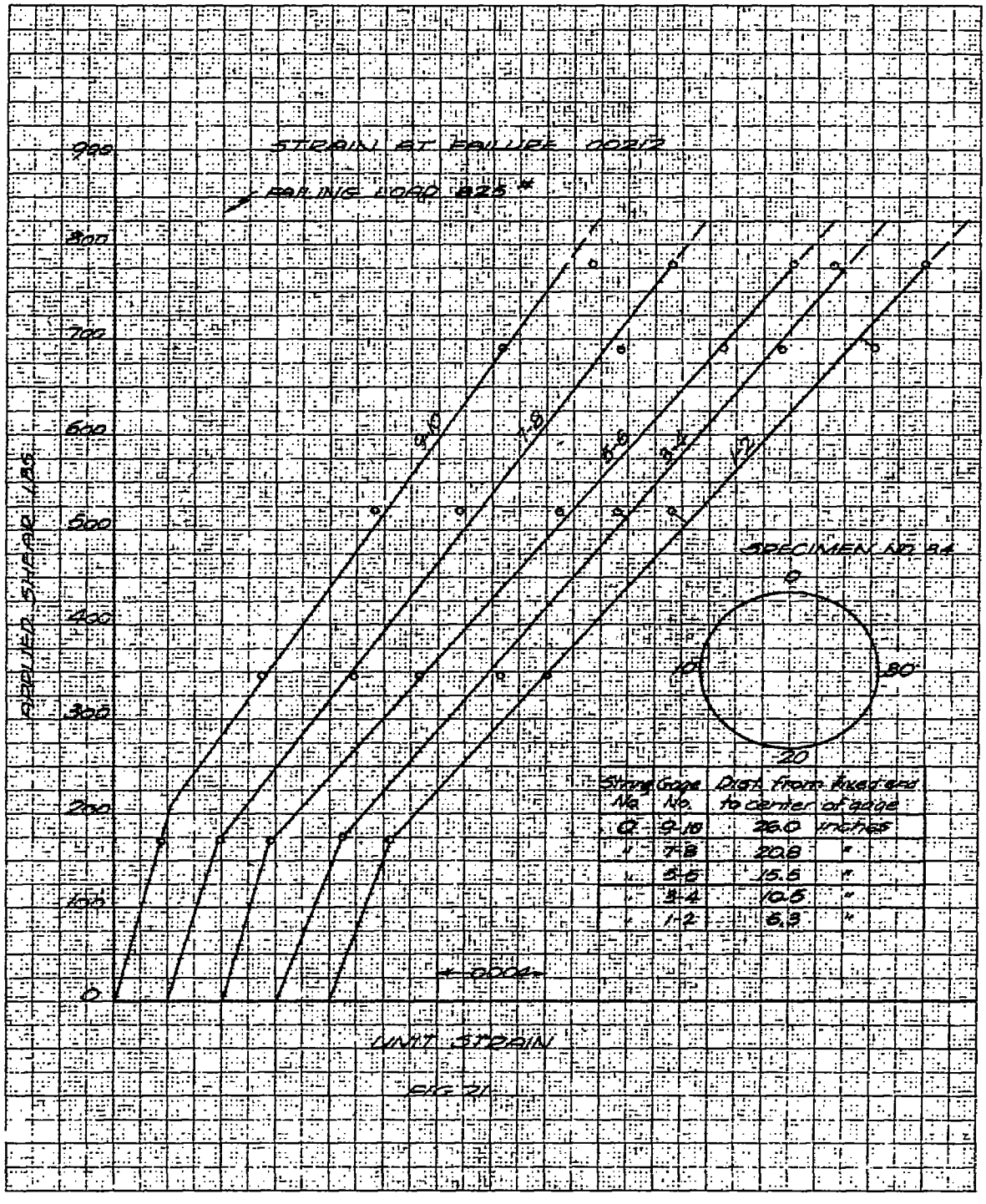
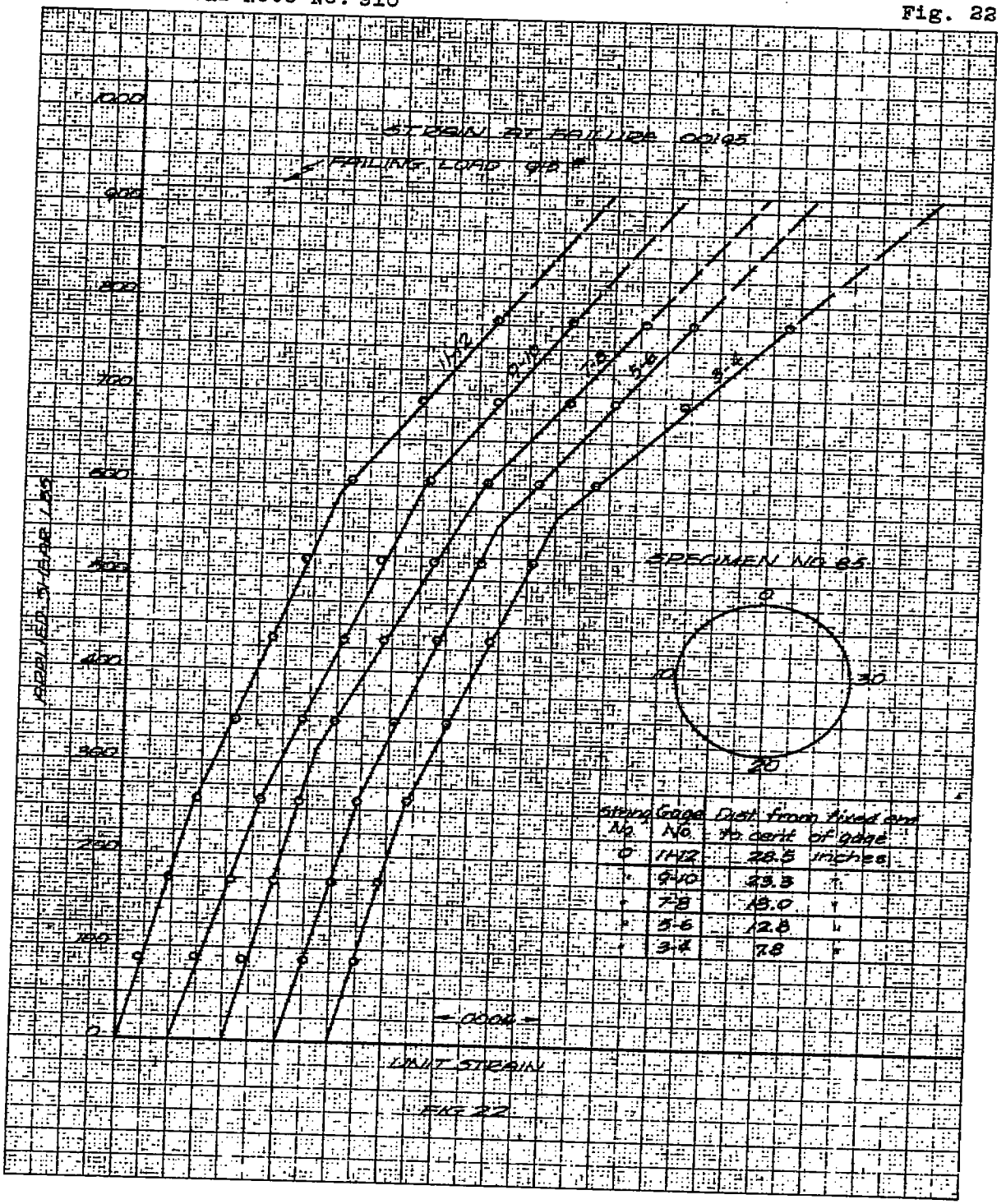

STRAIN AT FAILURE .0022
FAILING LOAD 825 #
SPECIMEN NO 94
FAILING SHEAR LBS
UNIT STRAIN
.0004
0
200
300
400
500
600
700
800
900
10
20
30
String No
Gage No
Dist. from fixed end to center of gage
9-10 26.0 inches
7-8 20.8
5-6 15.5
3-4 10.5
1-2 5.3

FIG 21

Fig. 22


STRAIN AT FAILURE .0095
FAILING LOAD 960 #
1142
9-10
7-8
5-6
3-4
SPECIMEN NO. 85
0
10
30
20
String No. | Gage No. | Dist. from fixed end to center of gage
0 | 1142 | 28.5 inches
1 | 9-10 | 23.3 "
2 | 7-8 | 18.0 "
3 | 5-6 | 12.8 "
4 | 3-4 | 7.8 "
1000
900
800
700
600
500
400
300
200
100
0
.0004
UNIT STRAIN
FIG 22

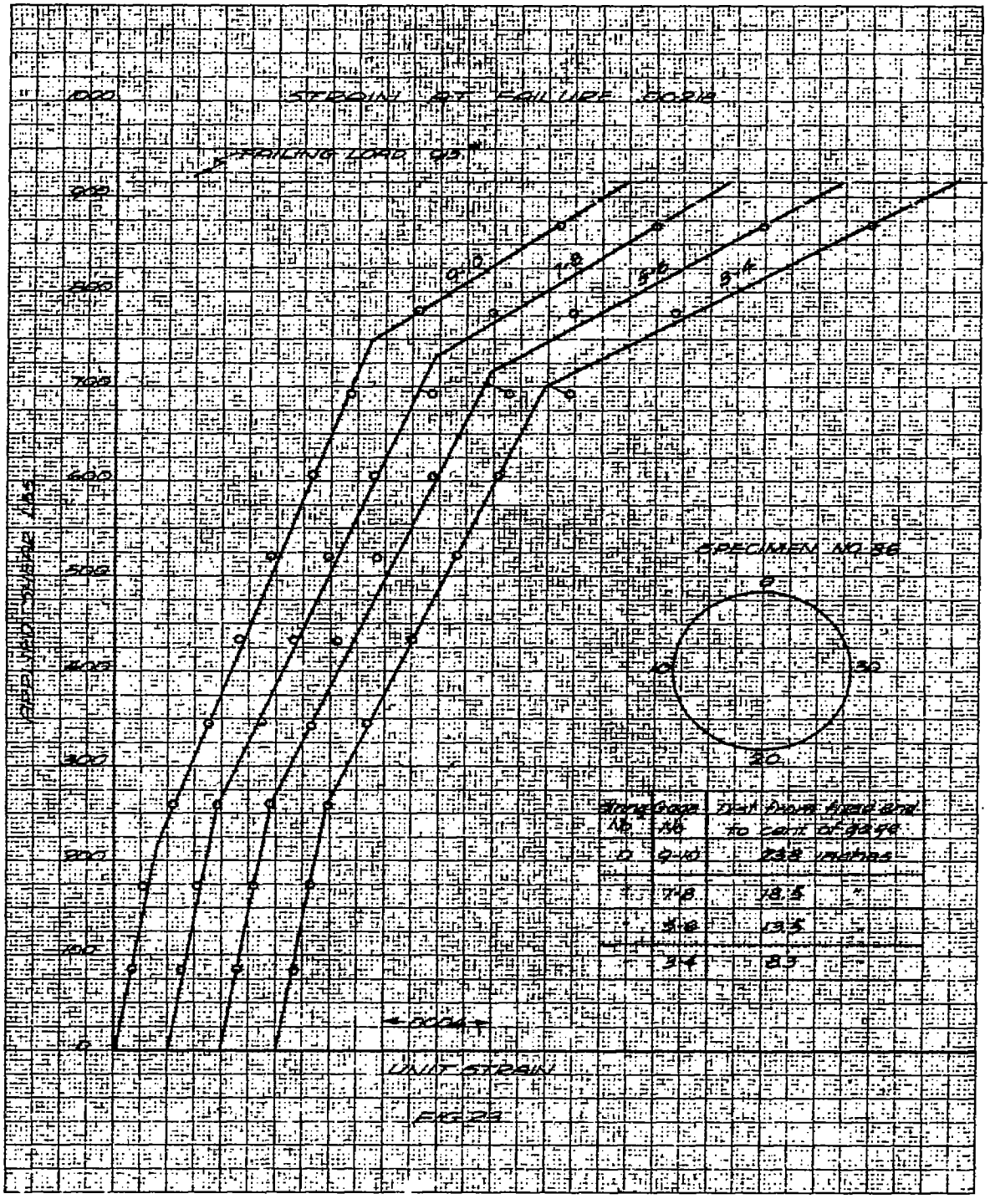
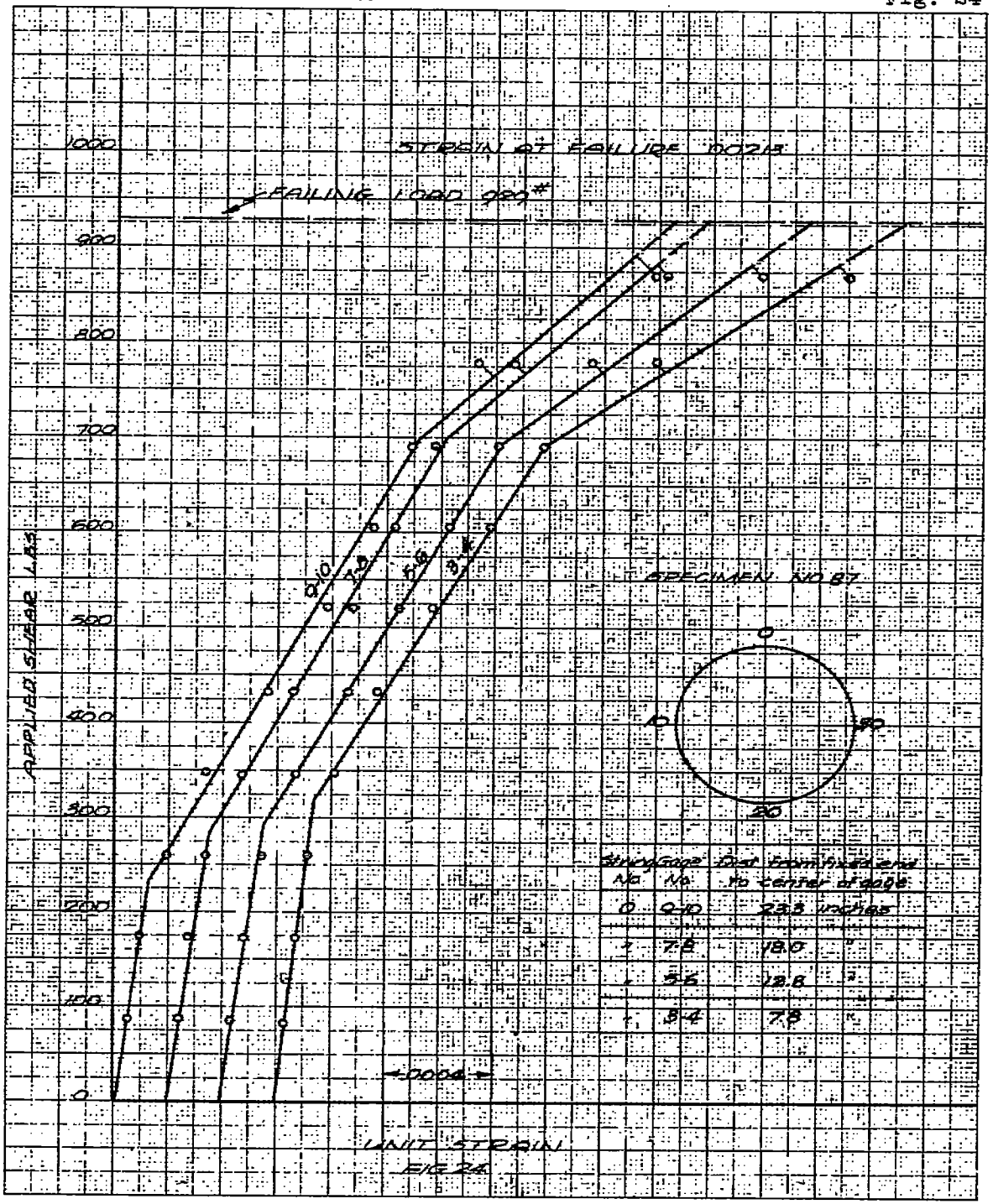
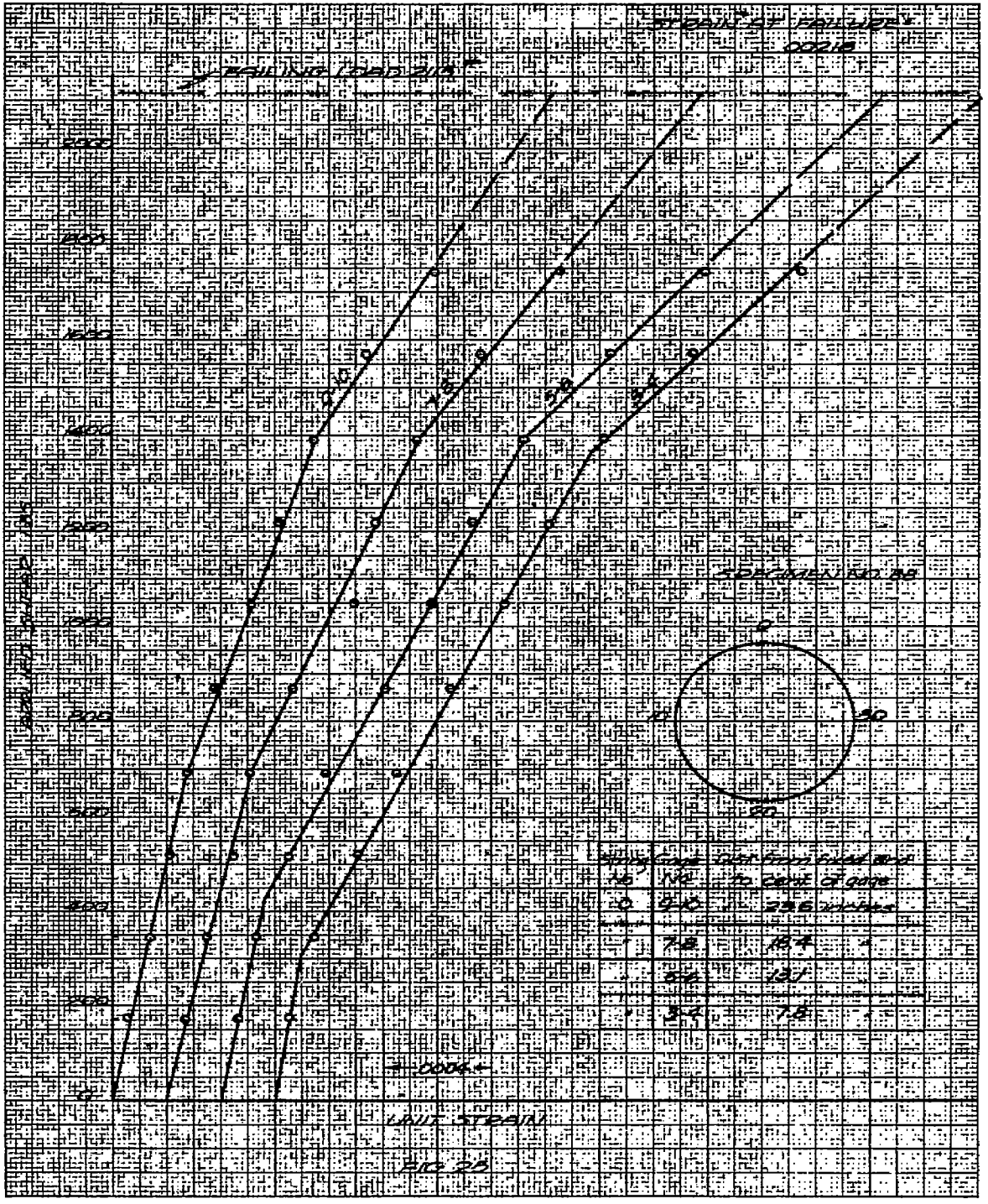
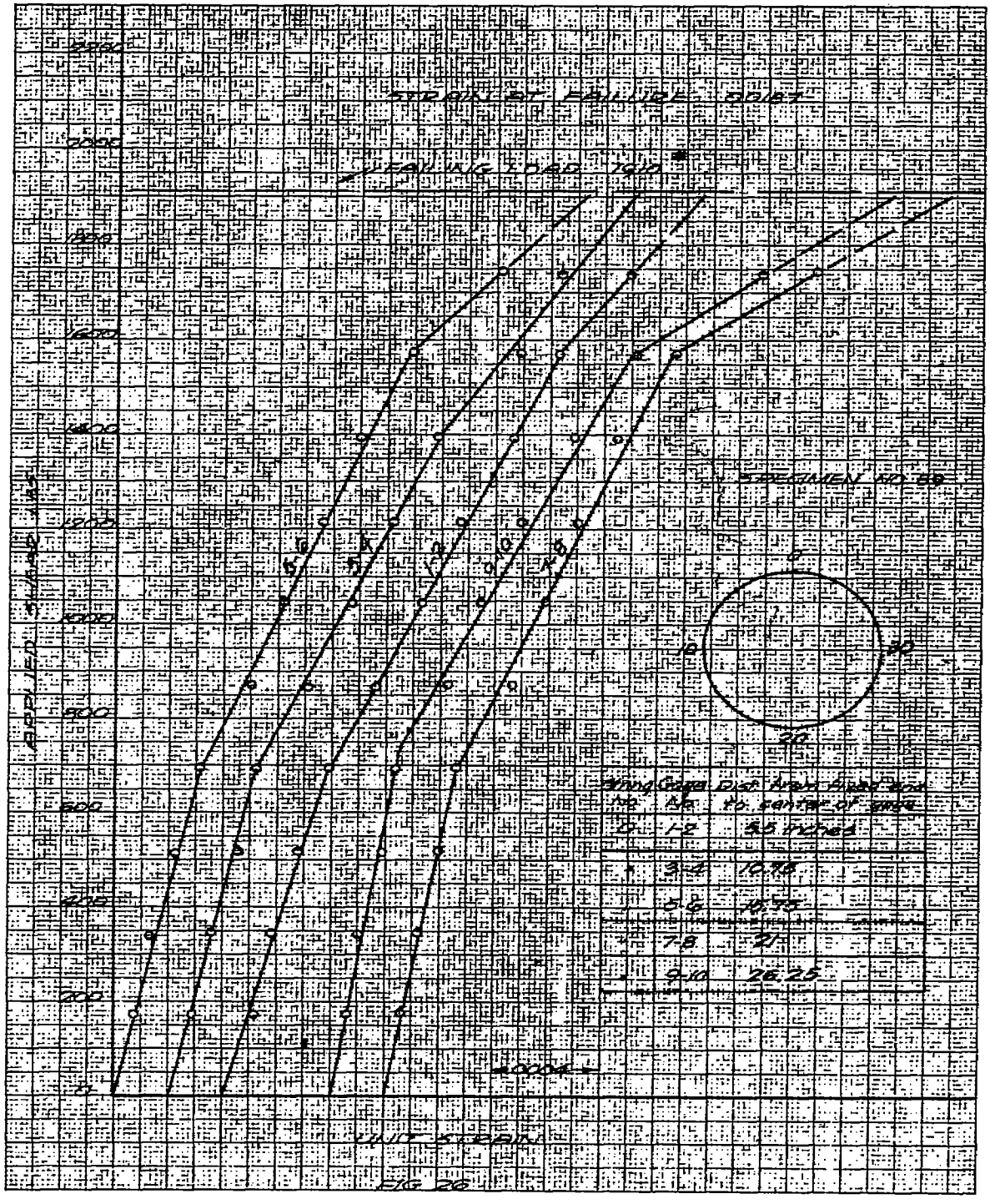
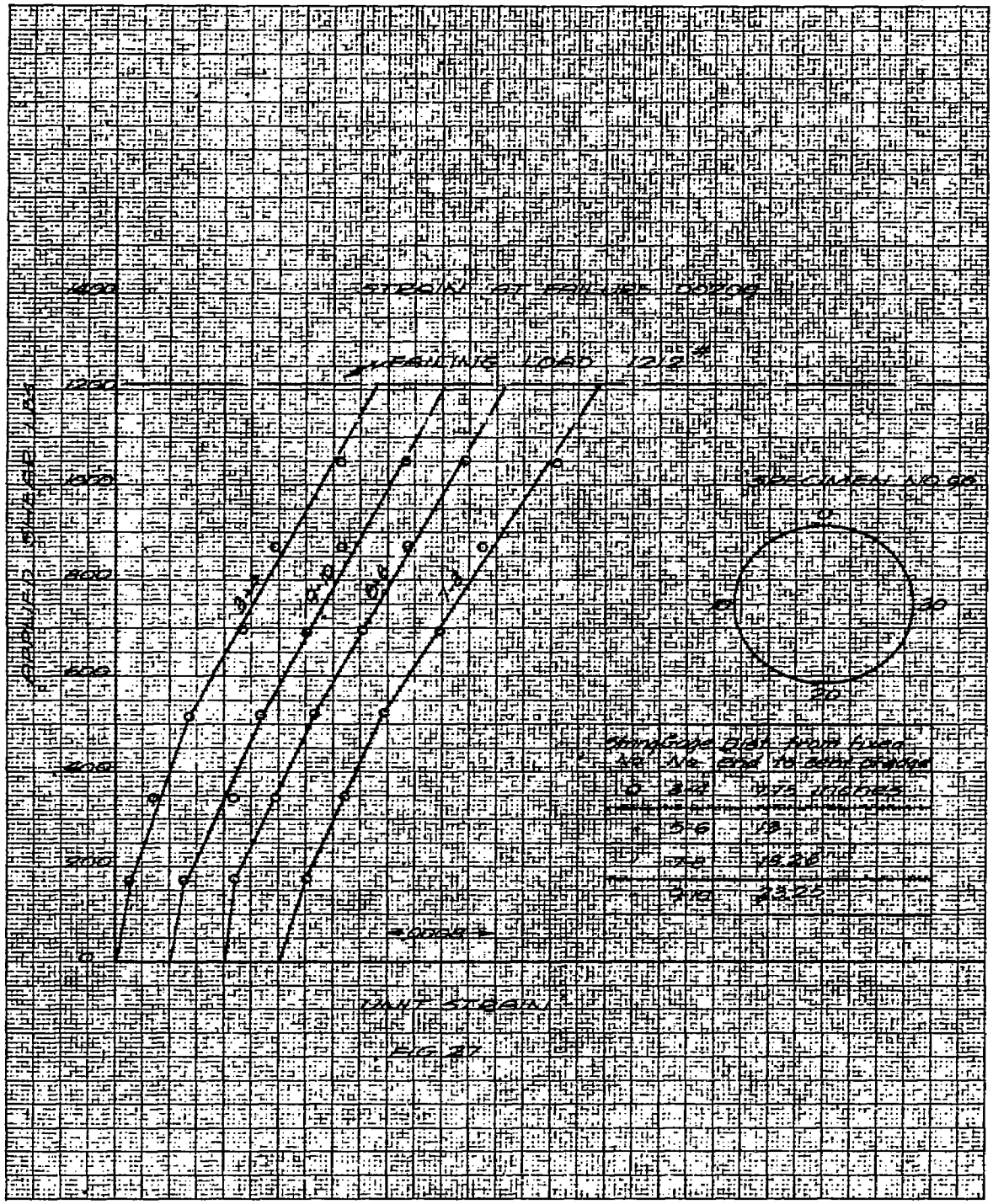

STRAIN AT FAILURE .00218
FAILING LOAD 93.
SPECIMEN NO. 56
UNIT STRAIN
FIG. 23

Fig. 24


STRAIN AT FAILURE 0024
FAILING LOAD 980#
APPLIED SHEAR LBS
1000
900
800
700
600
500
400
300
200
100
0
-.0004
UNIT STRAIN
FIG 24
SPECIMEN NO 87
0
10
20
30
Gage No
Distance from Edge to center of gage
0
0.10
2.35 inches
1
7.2
18.0
2
5.6
12.8
3
5.4
7.8


STRAIN AT FAILURE
FAILING LOAD
2000
1500
1000
500
0
.004
UNIT STRAIN
FIG. 25


STRAIN AT FAILURE .0087
FAILURE LOAD 7910
SPECIMEN NO 69
Strain Gage Dist from End and No. No. to center of gage
1-2 .50 inches
3-4 10.75
5-6 10.75
7-8 21
9-10 26.25
Fig 26


Airspeed, mph
Engine speed, rpm
Propeller No. 6868-6
Diameter 11 feet 10 inches
Blade angle at 0.75 radius
42.2°
Engine No. 1232
Fuel grade 91/96
Specification No. 90
Test stand
Fig. 27

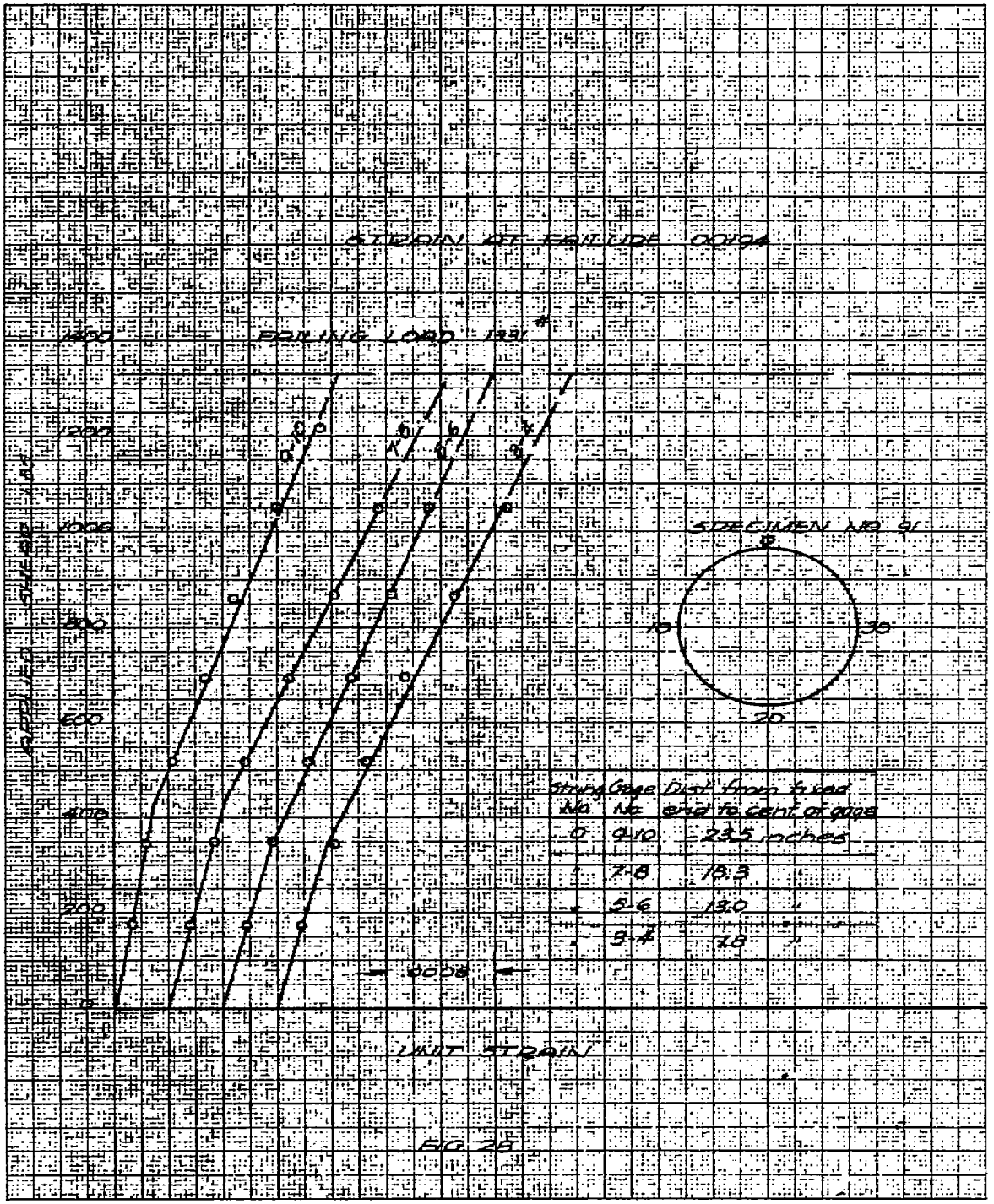
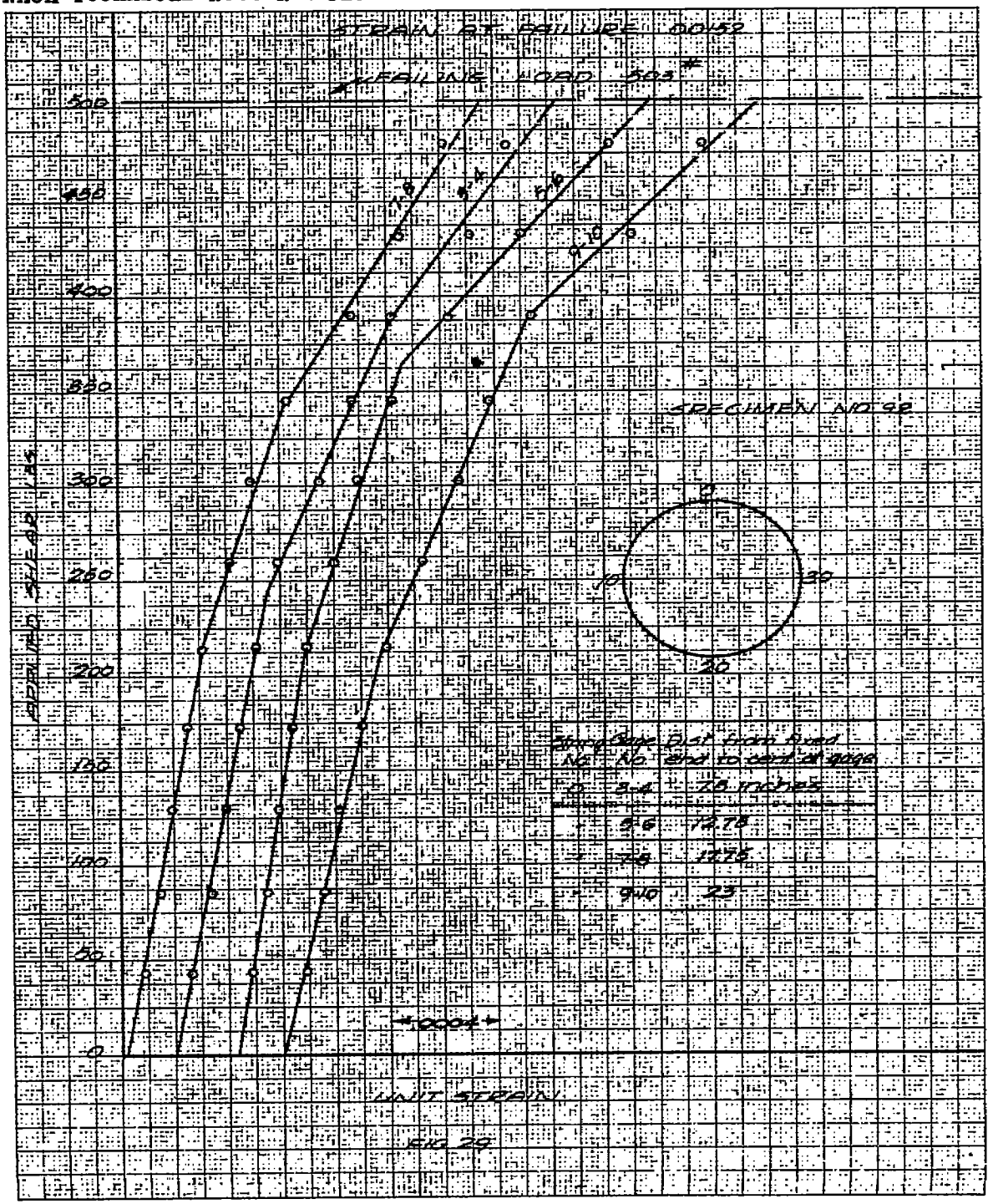
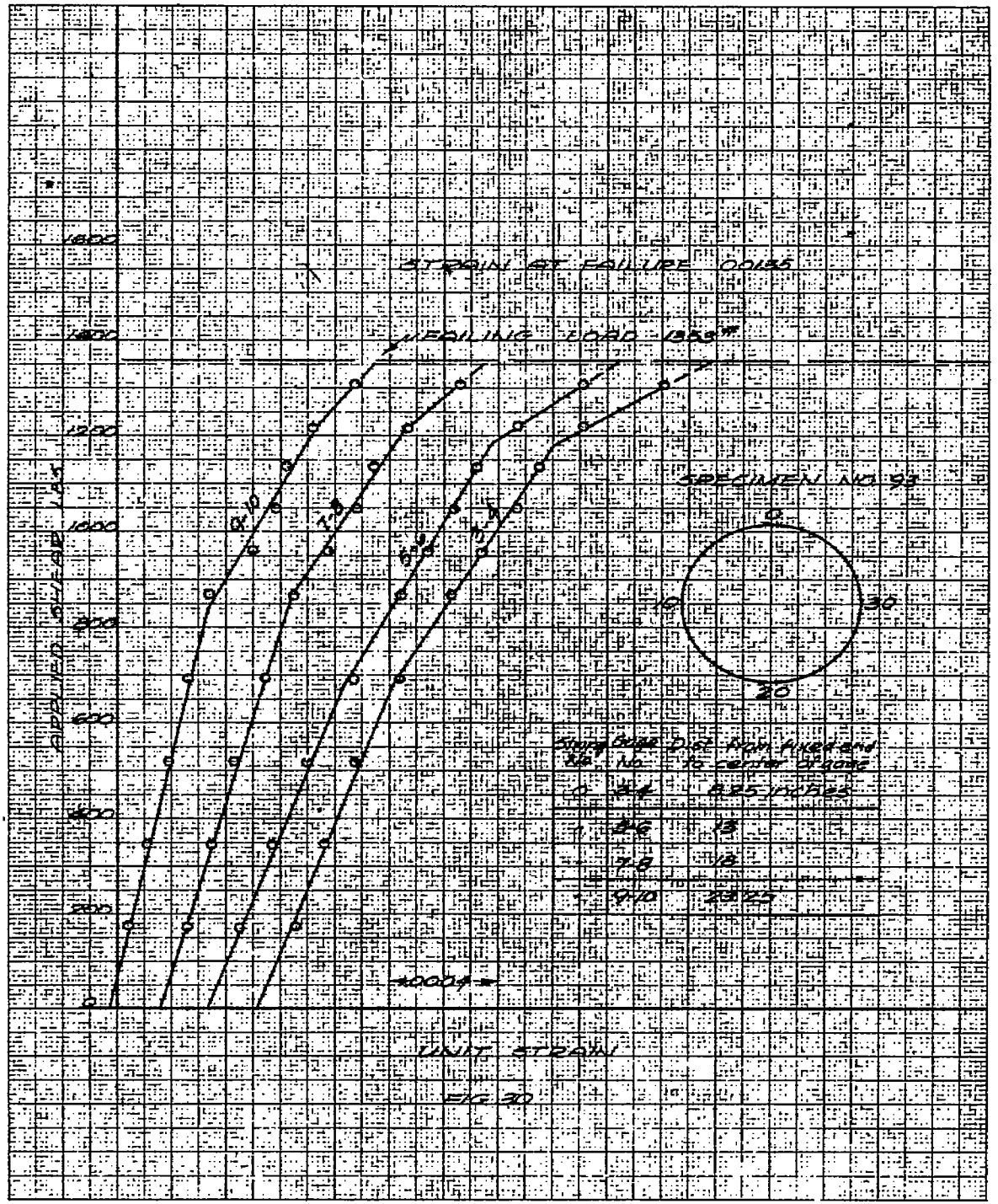
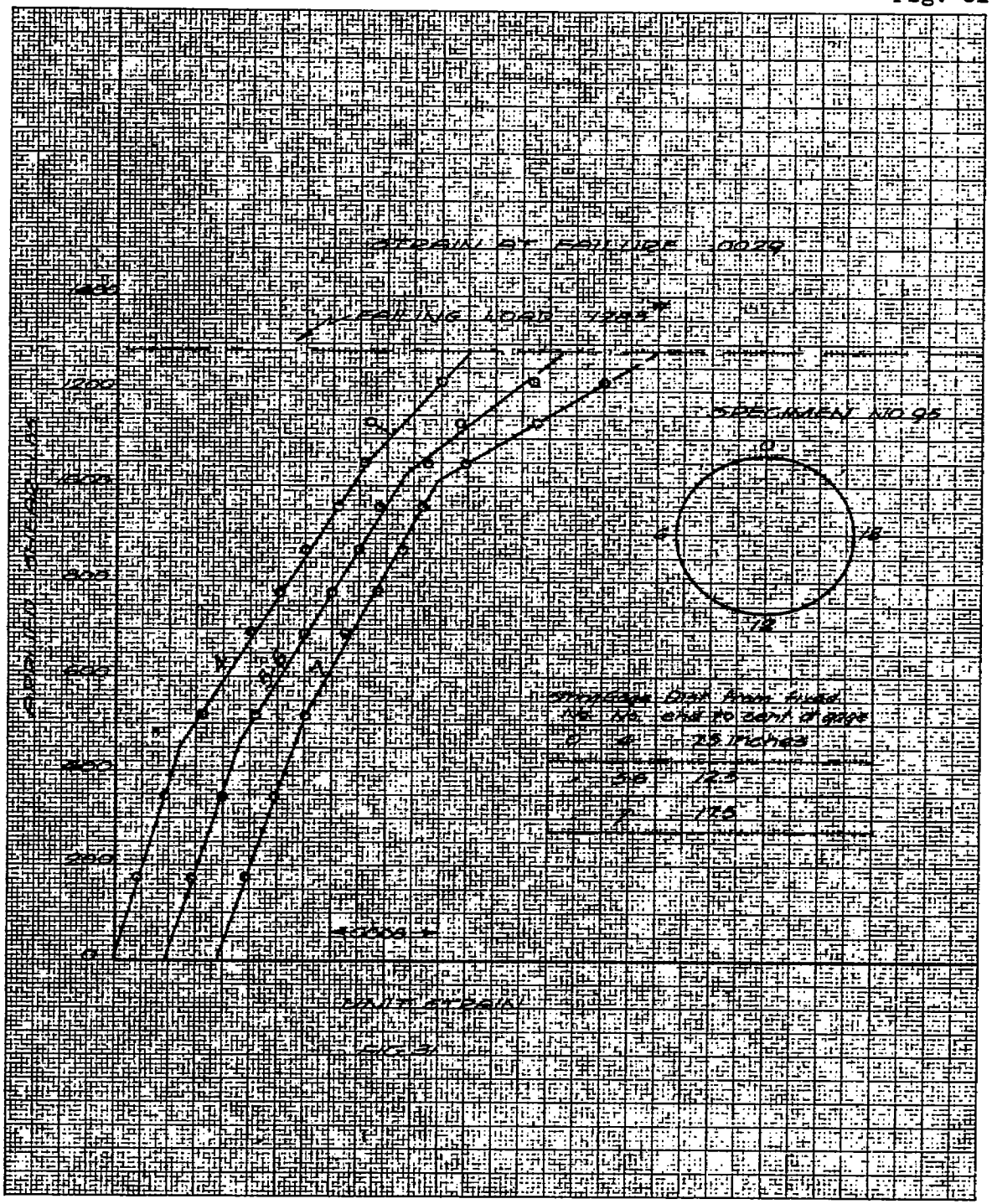
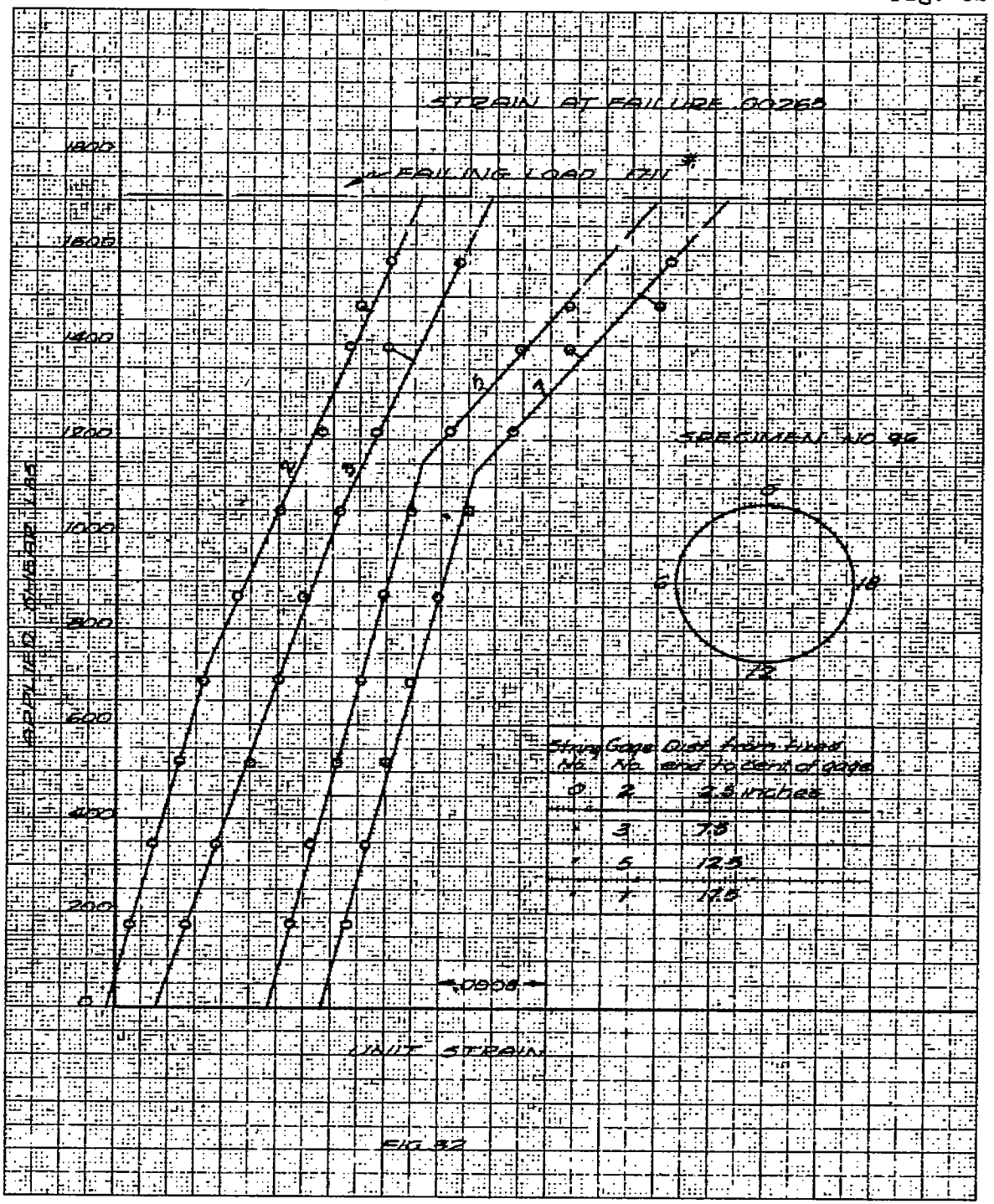

STRAIN AT FAILURE .0094
FAILING LOAD 1891 #
1600
1200
800
400
200
0
.0052
UNIT STRAIN
SPECIMEN No 91
10
30
20
Strain Gage No
Gage No
Dist from fixed end to cent of gage
0
9-10
2.35 inches
1
7-8
1.93
2
5-6
1.90
3
3-4
1.88

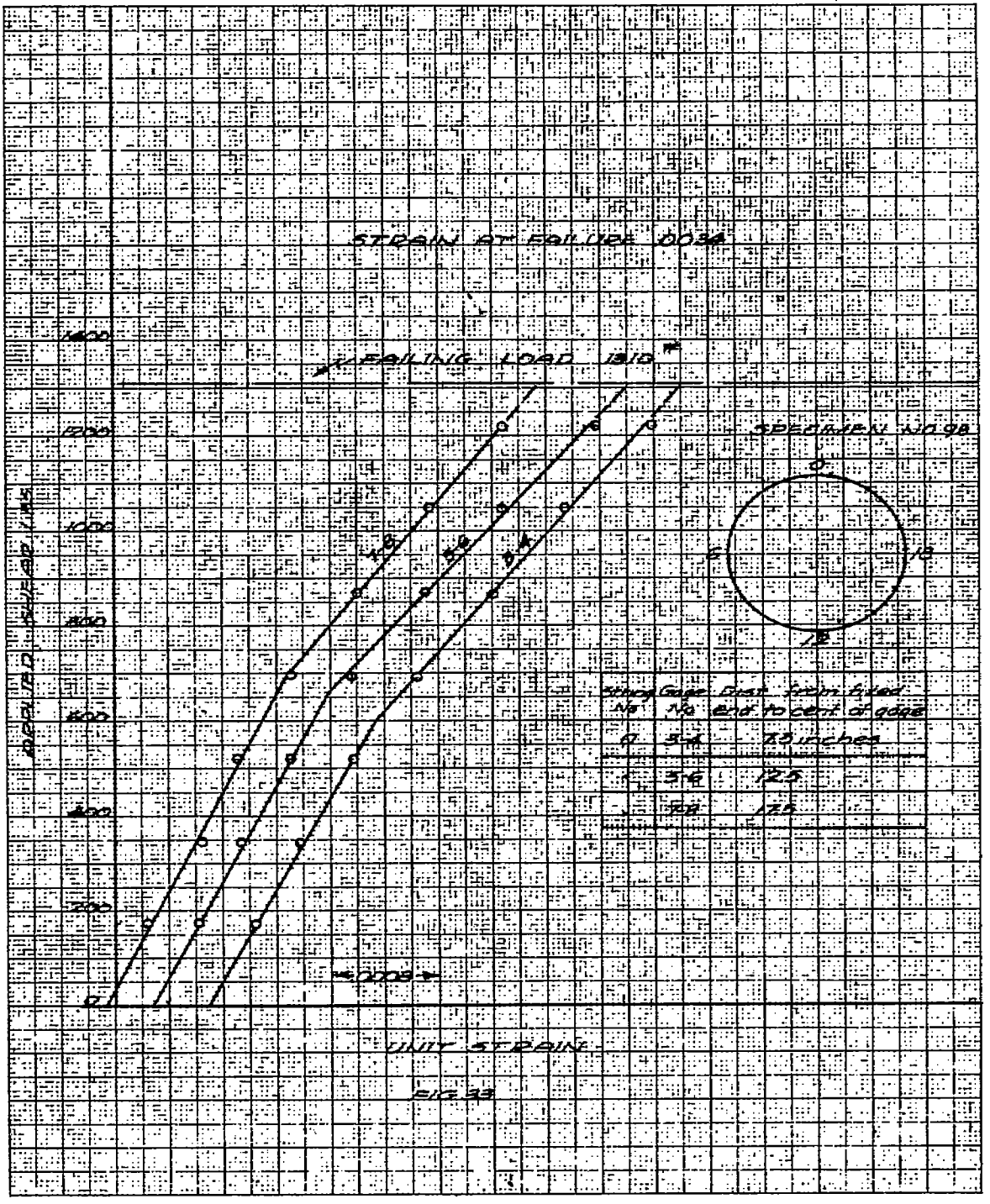
FIG 28

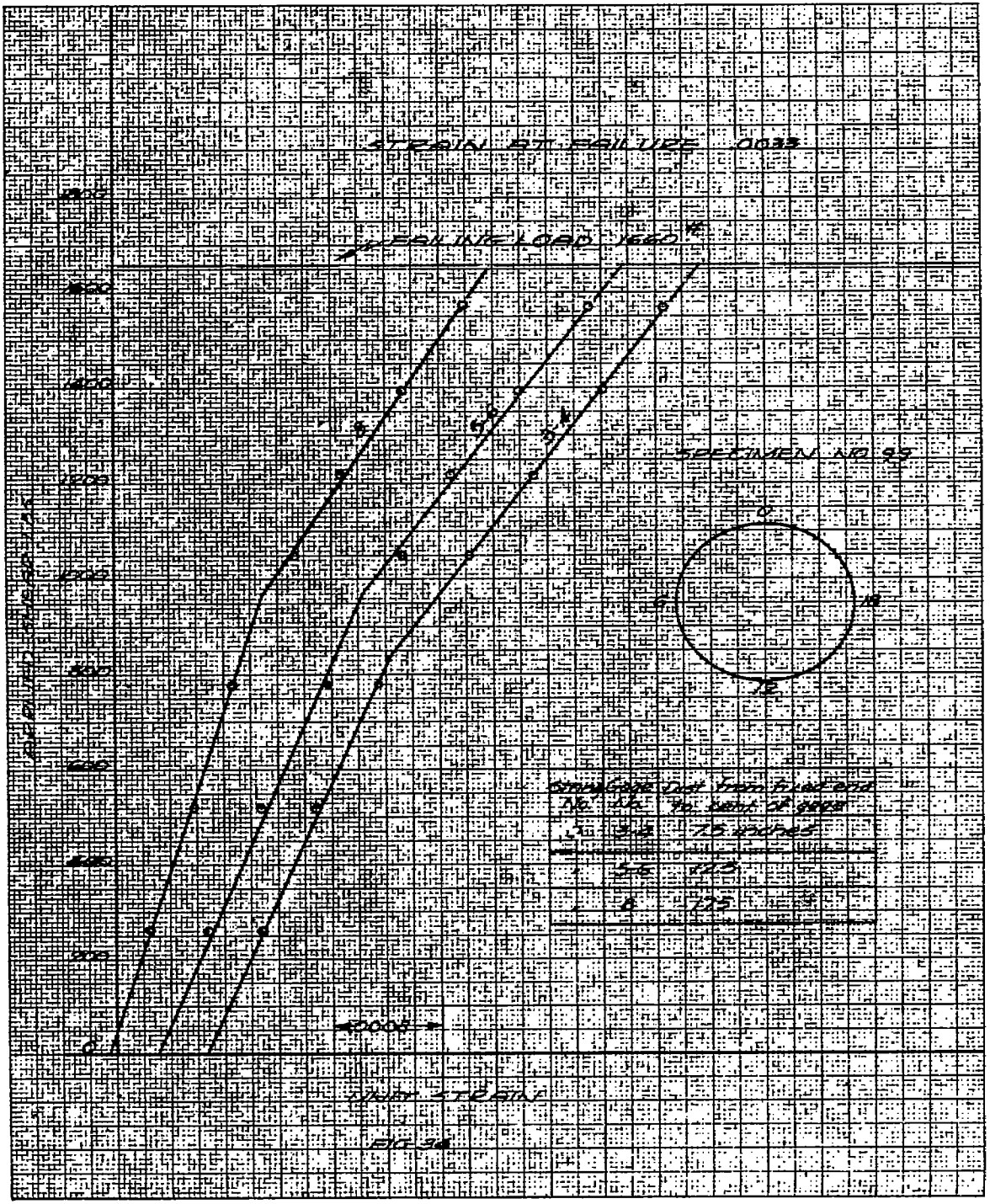

STRAIN AT FAILURE .00552
FAILURE LOAD 503 #
500
400
300
200
100
0
SPECIMEN NO 98
10
30
20
Strain Gage Dist from Rivet
No. No. and to center of gage
3-4 .25 inches
5-6 .75 "
7-8 1.25 "
9-10 2.5 "
.0004
UNIT STRAIN
FIG 29


STRAIN AT FAILURE .0035
FAILING LOAD 1955#
SPECIMEN NO 93
UNIT STRAIN
EXT. 30


SPECIMEN NO 95
12
12
0
4
28 inches
4.3
120
1200
1000
800
600
400
200
0


STRAIN AT FAILURE .00265
FAILING LOAD 1711#
SPECIMEN NO 96
String No.
Gage No.
Dist. from Load end in Inches and Percent of Gage
0 2 2.5 INCHES
3 7.5
5 12.5
7 17.5
STRESS LBS/SQ IN
UNIT STRAIN
FIG 32


STRAIN AT FAILURE .0034
FAILING LOAD 1310 #
SPECIMEN NO 98
Strain Gage Dist. from fixed end to cent of gage
No 1 3/4 .75 inches
No 2 3/4 1.25
No 3 1/2 1.75
1400
1200
1000
800
600
400
200
.0034
UNIT STRAIN
FIG 33



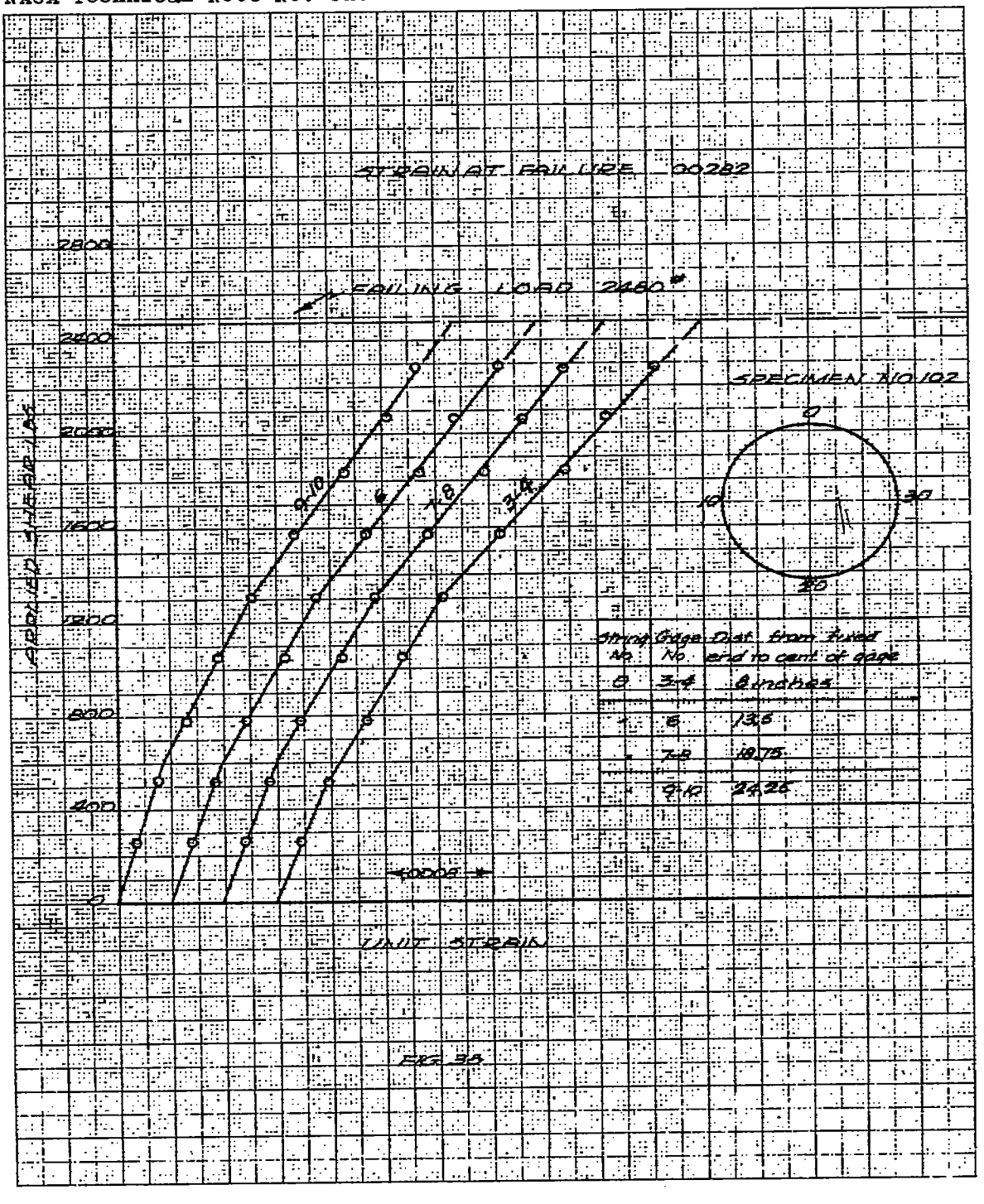
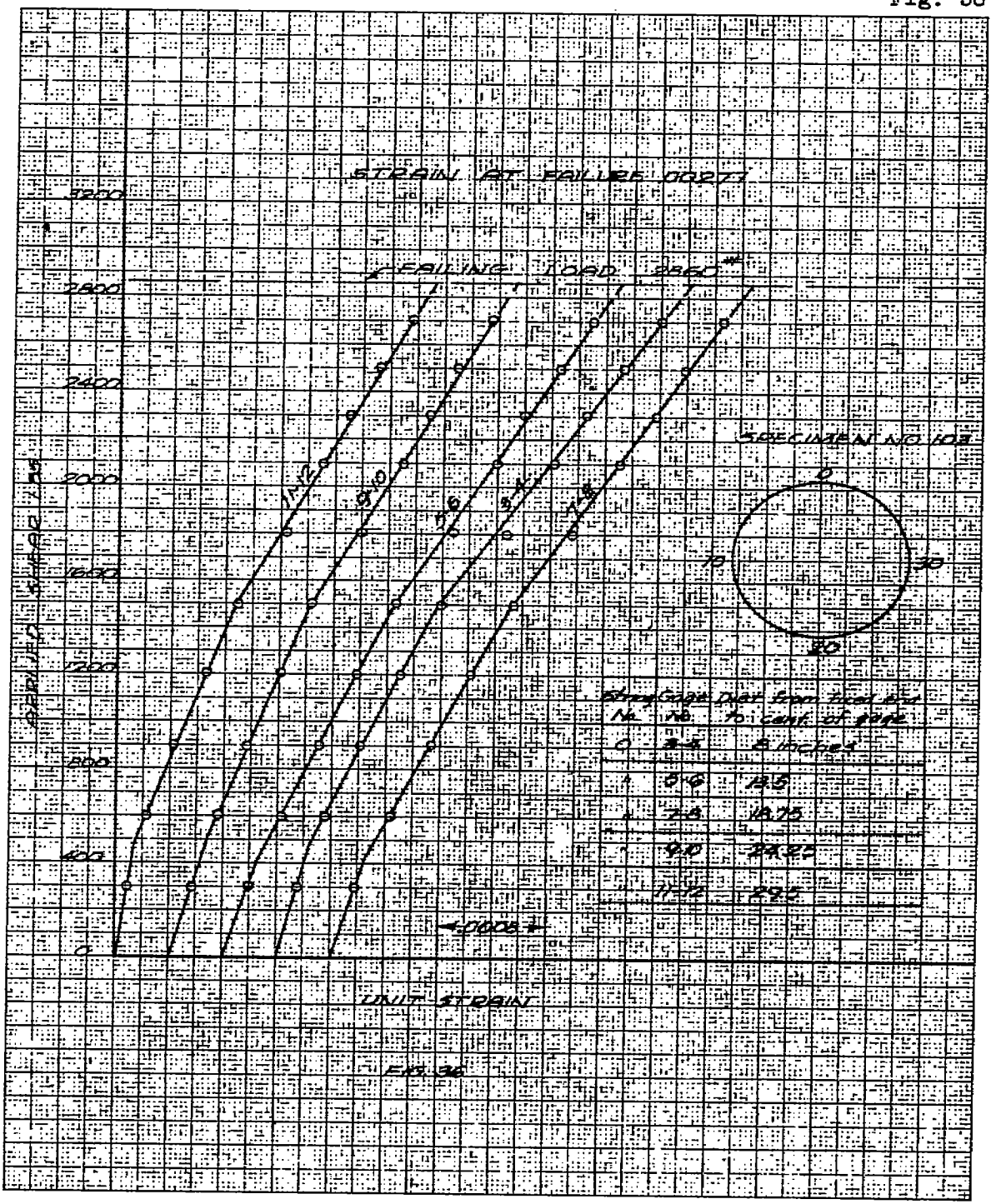
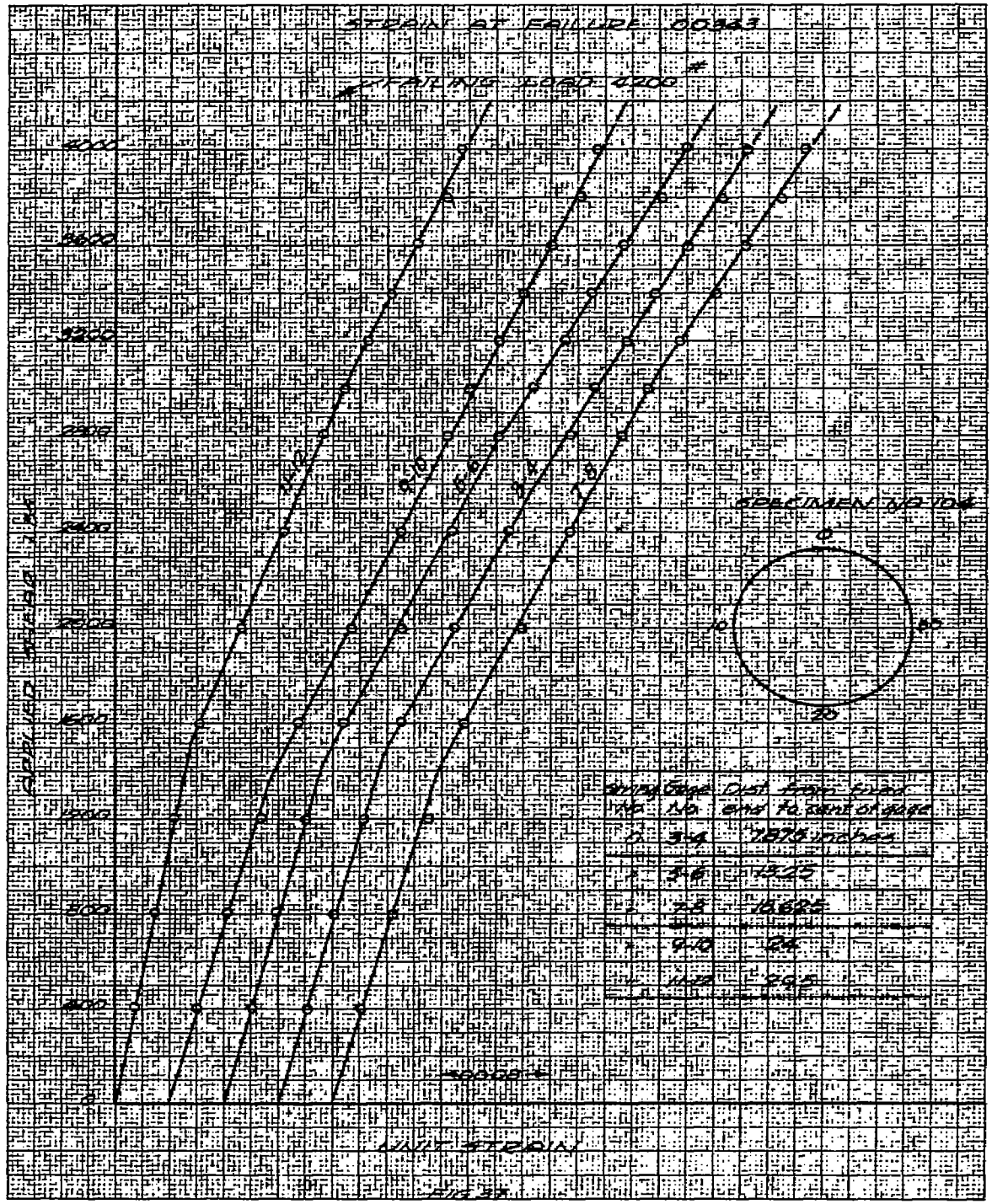

STRAIN AT FAILURE .00282
FAILING LOAD 2460#
SPECIMEN NO 102
String No
Gage No
Dist. from fixed end to cent of gage
Inches
0 3-4
1 6 13.5
2 7-8 18.75
3 9-10 24.25
.0005 in
UNIT STRAIN
FIG 35

Fig. 36


STRAIN AT FAILURE .00277
FAILING LOAD 2960 #
SPECIMEN NO 103
0
10
20
Strain Gage No.
Dist from load end
% Dist of Gage
0
2-3
4 inches
5-6
12.5
7-8
18.75
9-10
24.25
11-12
29.5
UNIT STRAIN
FIG 36


STRAIN at FAILURE .0033
STRAINS LOAD 4200 #
4000
3800
3600
3400
3200
3000
2800
2600
2400
2200
2000
1800
1600
1400
1200
1000
800
600
400
200
0
-200
-400
-600
-800
-1000
-1200
-1400
-1600
-1800
-2000
-2200
-2400
-2600
-2800
-3000
-3200
-3400
-3600
-3800
-4000
SPECIMEN NO 104
0
10
20
30
Strain Gage Dist from Load
No No one Percent of Gage
1 3.4 7.375 inches
2 3.6 4.25
3 7.8 1.625
4 9.0 .24
5 11.2 .295
-.00200
-.00400
-.00600
-.00800
-.01000
-.01200
-.01400
-.01600
-.01800
-.02000
-.02200
-.02400
-.02600
-.02800
-.03000
-.03200
-.03400
-.03600
-.03800
-.04000
AVG STRAIN
.0033

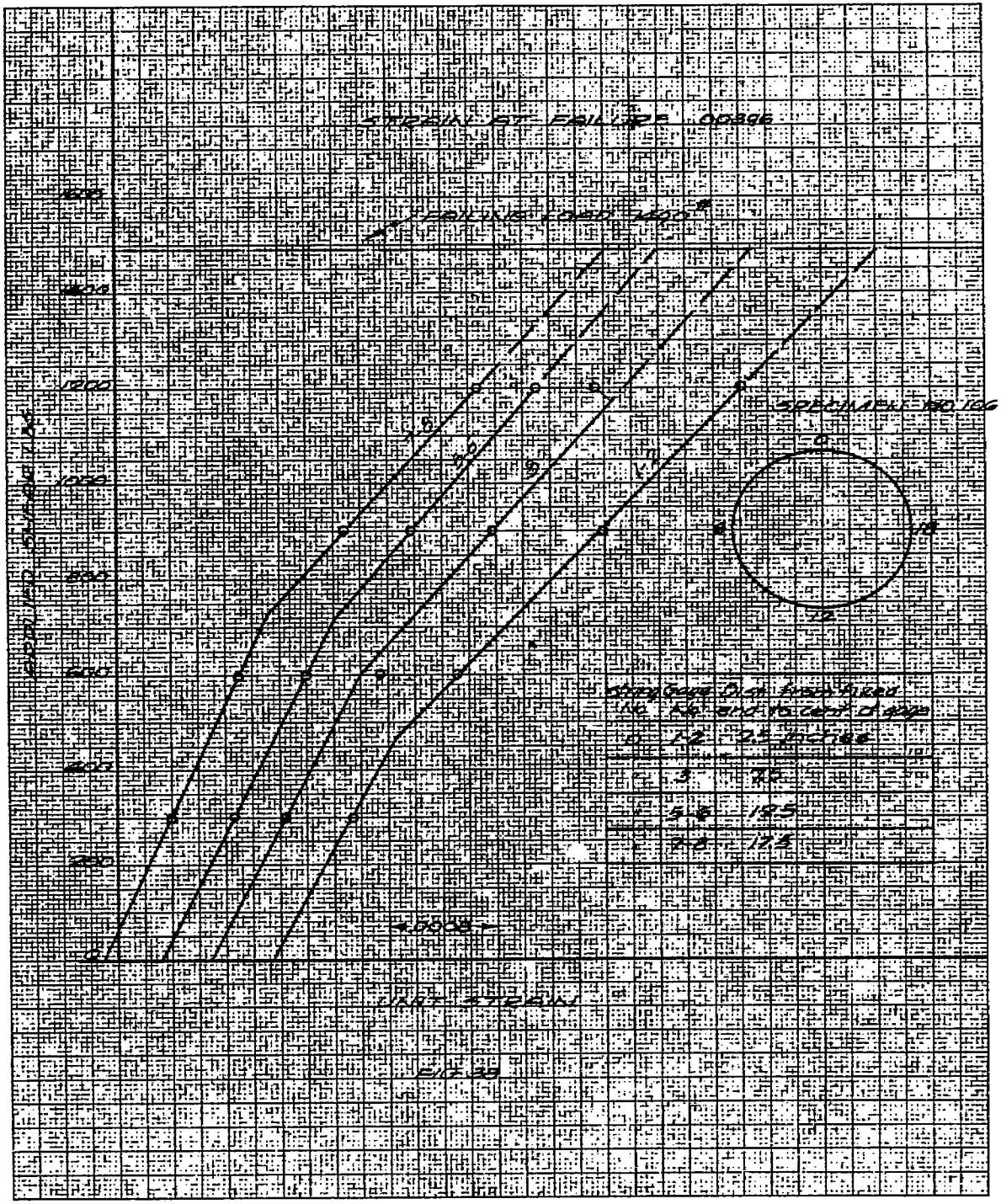
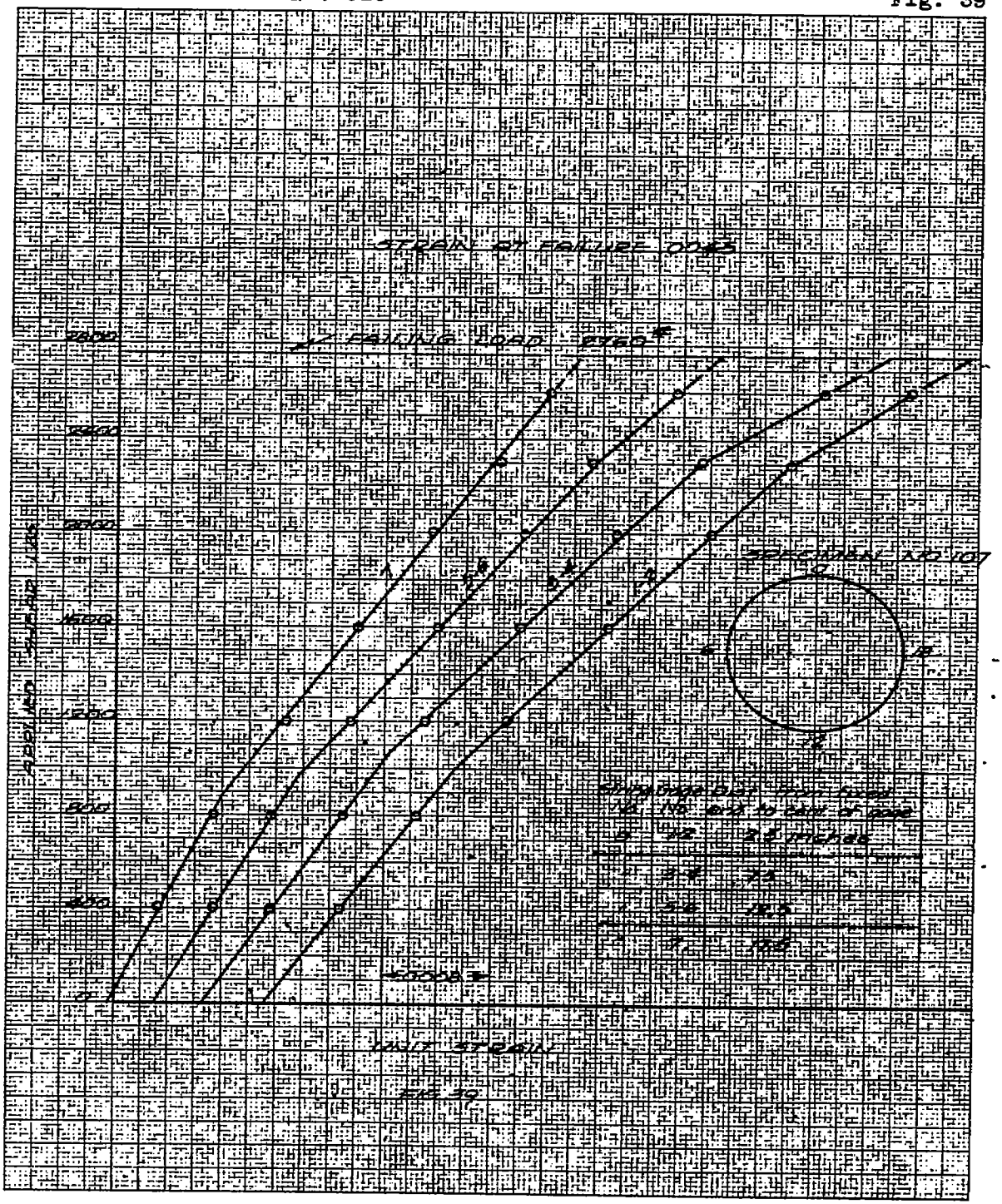
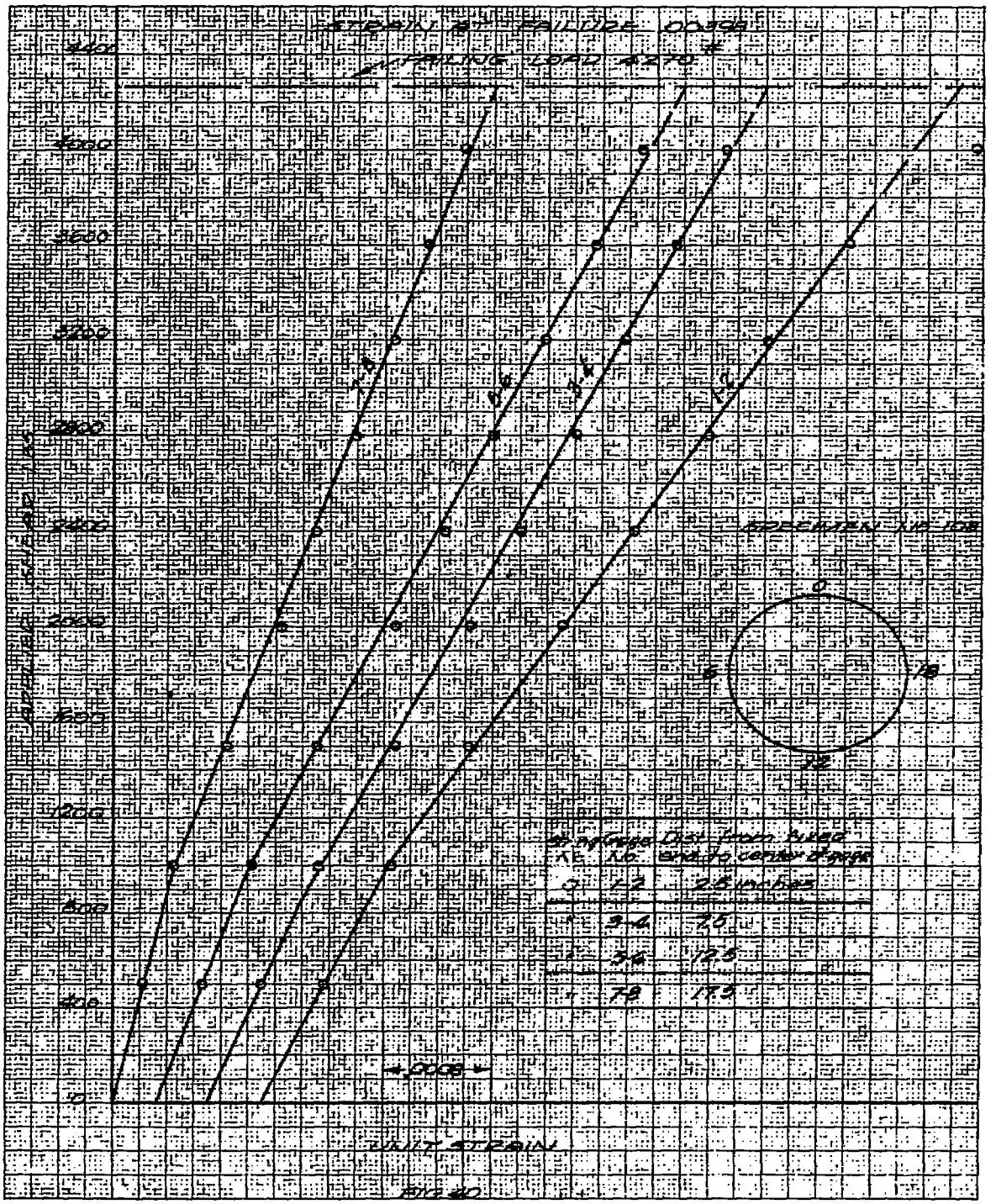
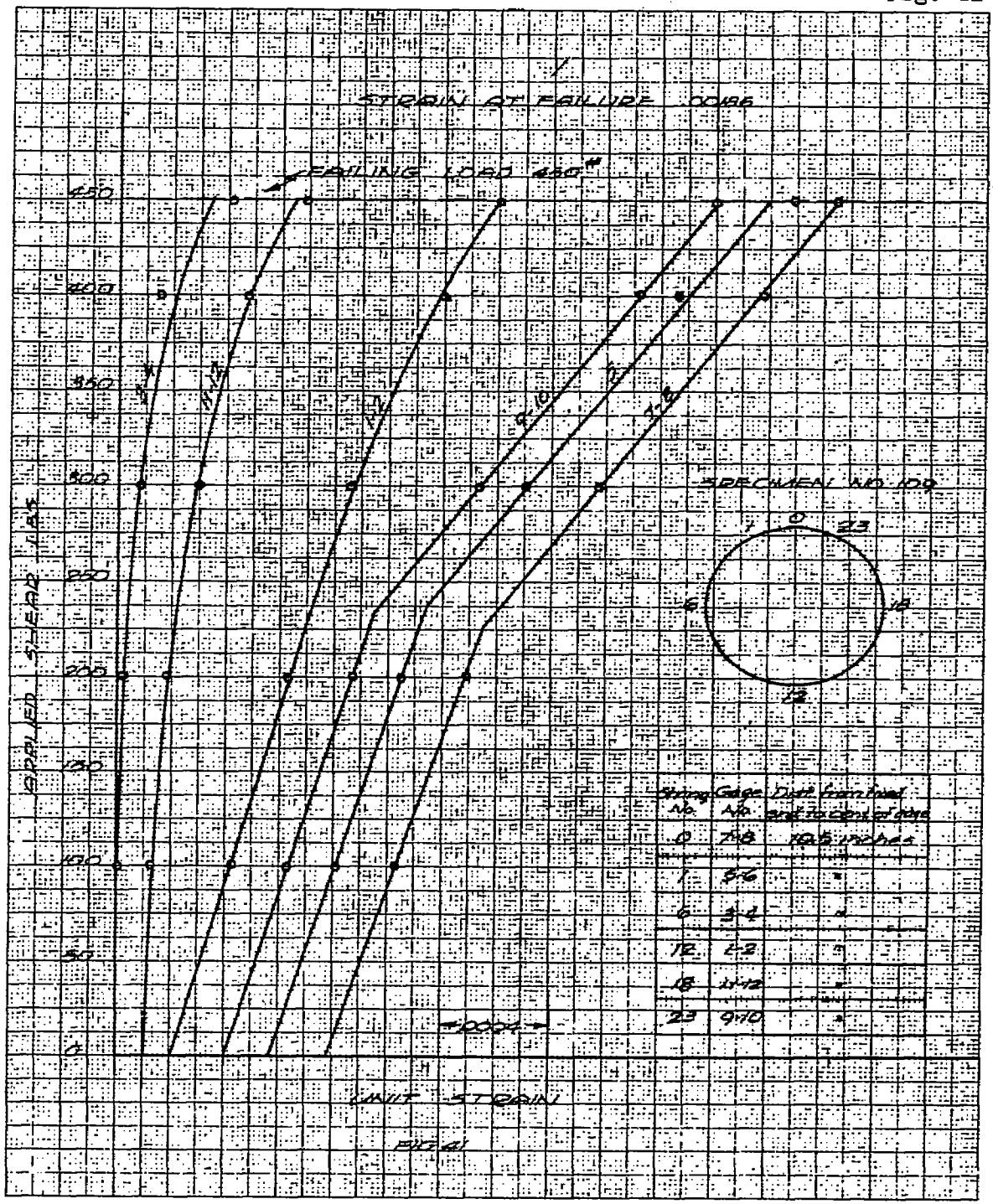
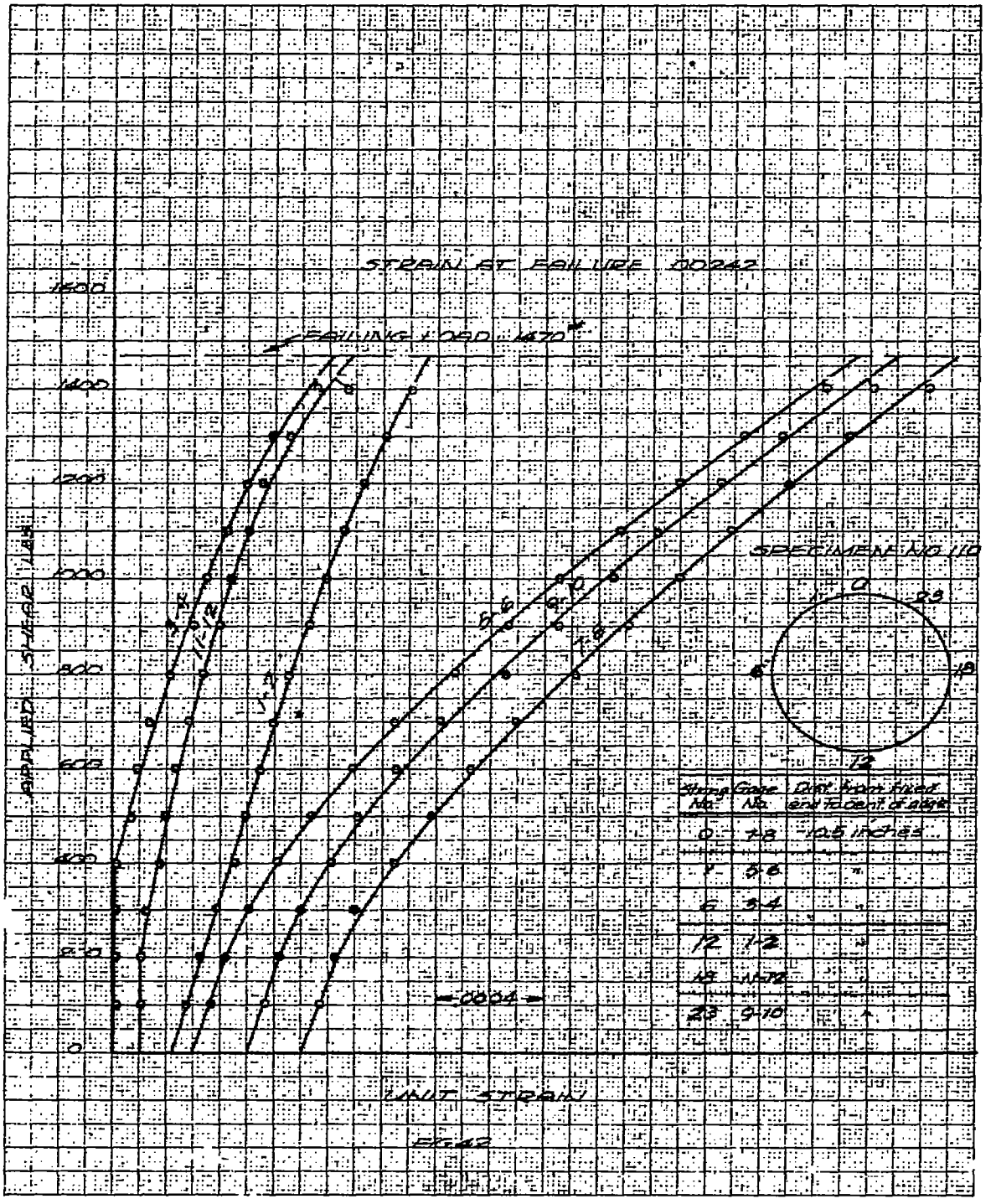


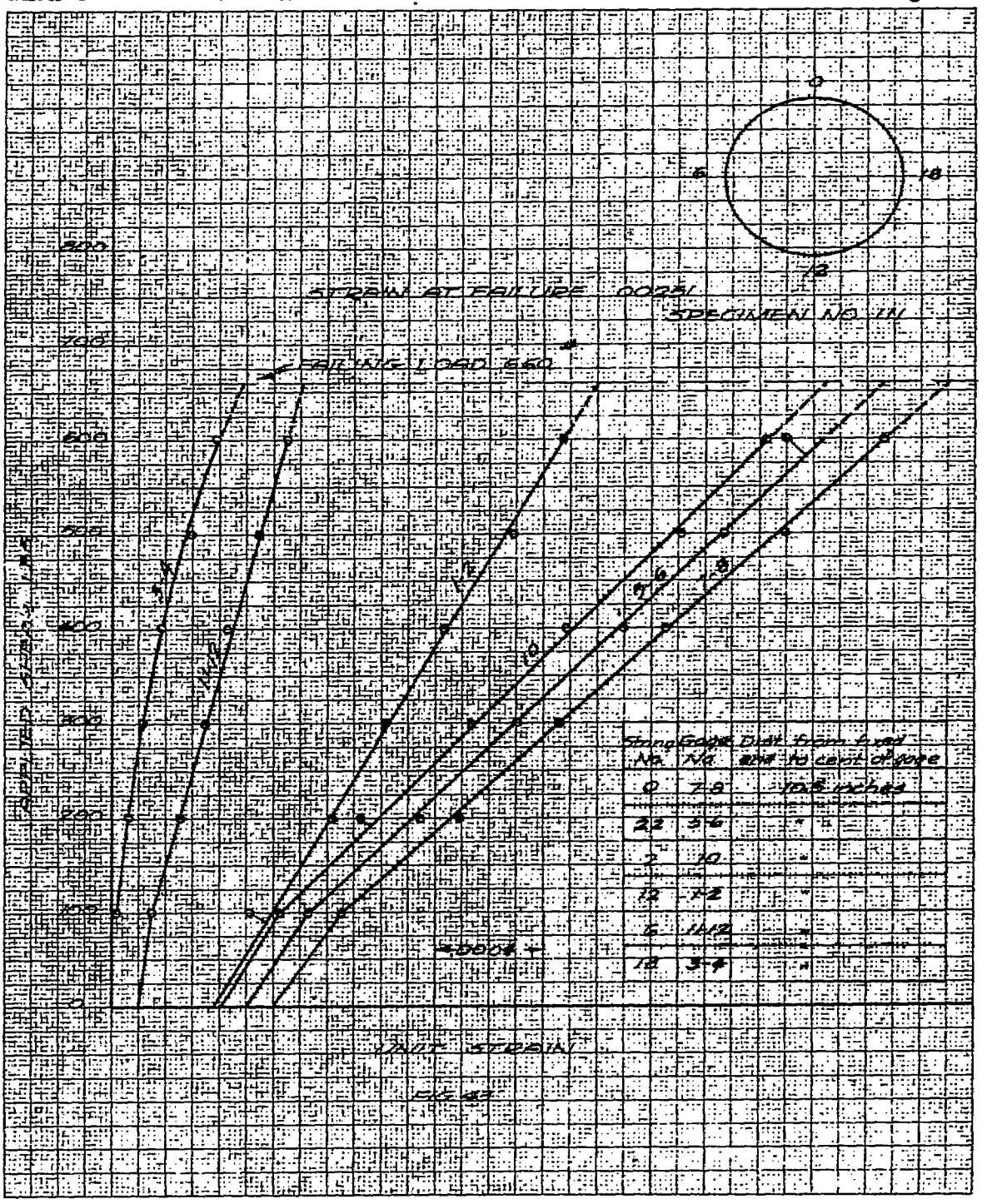
Fig. 39

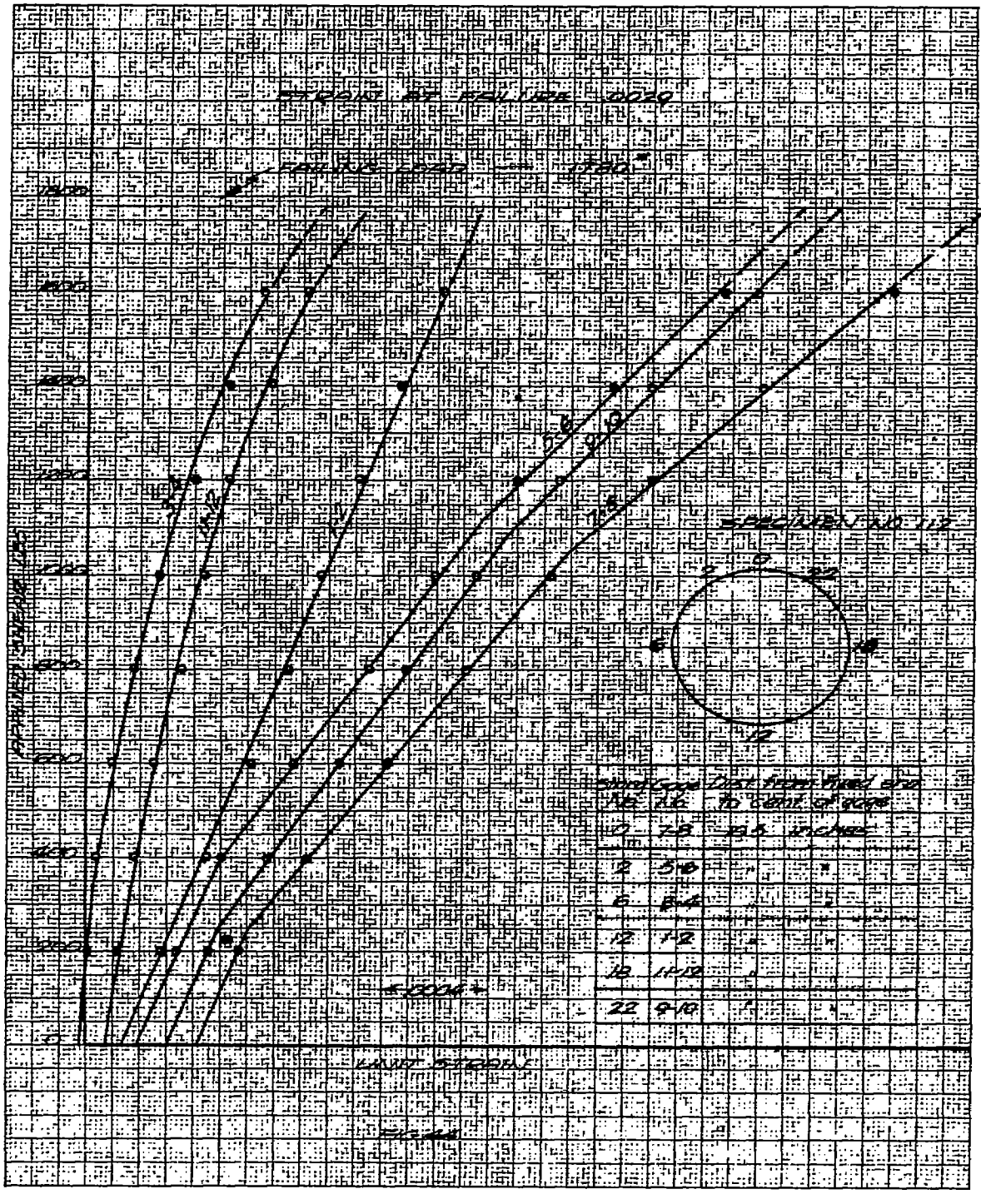


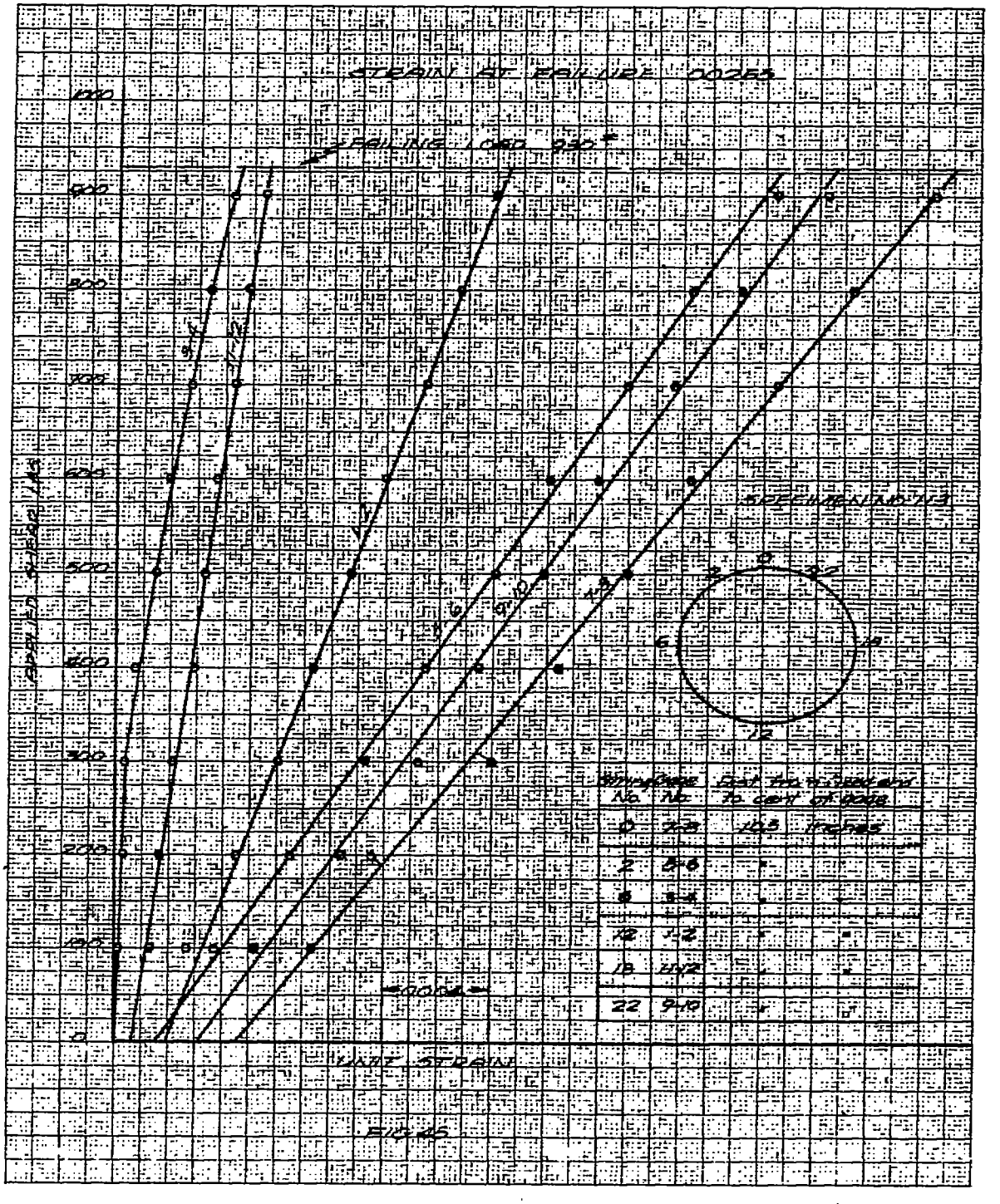

Strain, at failure 0.0394
Breaking load 4270 #
4400
4200
4000
3800
3600
3400
3200
3000
2800
2600
2400
2200
2000
1800
1600
1400
1200
1000
800
600
400
200
0
1,000 #
Unit stress in lbs.
Specimen No. 108
0
12
No. of gage Dist. from edge
No. to end to center of gage
0 1.2 2.5 inches
1 3.4 7.5
2 5.6 12.5
3 7.8 17.5


STRAIN AT FAILURE .00196
FAILING LOAD 450#
SPECIMEN NO 109
APPLIED SHEAR LBS
UNIT STRAIN
.0004
String No.
Gage No.
Dist from load
0 7/8 1/2 inches
1 5/16
6 3/4
12 4.2
18 4.12
23 9/10


STRAIN AT FAILURE .00242
FAILING LOAD 1470 lbs
SPECIMEN NO 110
Wire Gage No. Dist from fixed end to Center of gage
0 18 .125 inches
6 5.6 "
6 3.4 "
12 1.2 "
18 .444 "
23 .310 "
.0024
LIMIT STRAIN
STRESS
APPLIED LBS
1500
1400
1200
1000
800
600
400
200
0


STRAIN AT FAILURE .00751
SPECIMEN NO. III
FAILING LOAD 660 #
0
6
12
18
900
700
600
500
400
300
200
100
0
STRAIN IN MICRO-INCHES PER INCH
Strain Gage No.
Dist. from Load
and % of Gage
0 7.9 105 inches
22 3.6 " "
2 10 " "
12 2.2 " "
6 1.412 " "
18 3.4 " "
+ 500# +
UNIT STRESS IN
PSI




STRAIN AT FAILURE .0025
FAILING LOAD 930 #
1000
900
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500
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-.0004
.0004
.0008
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1.4496
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1.4508
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1.4596
1.4600
1.4604
1.4608
1.4612
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1.4624
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1.4640
1.4644
1.4648
1.4652
1.4656
1.4660
1.4664
1.4668
1.4672
1.4676

Fig. 46

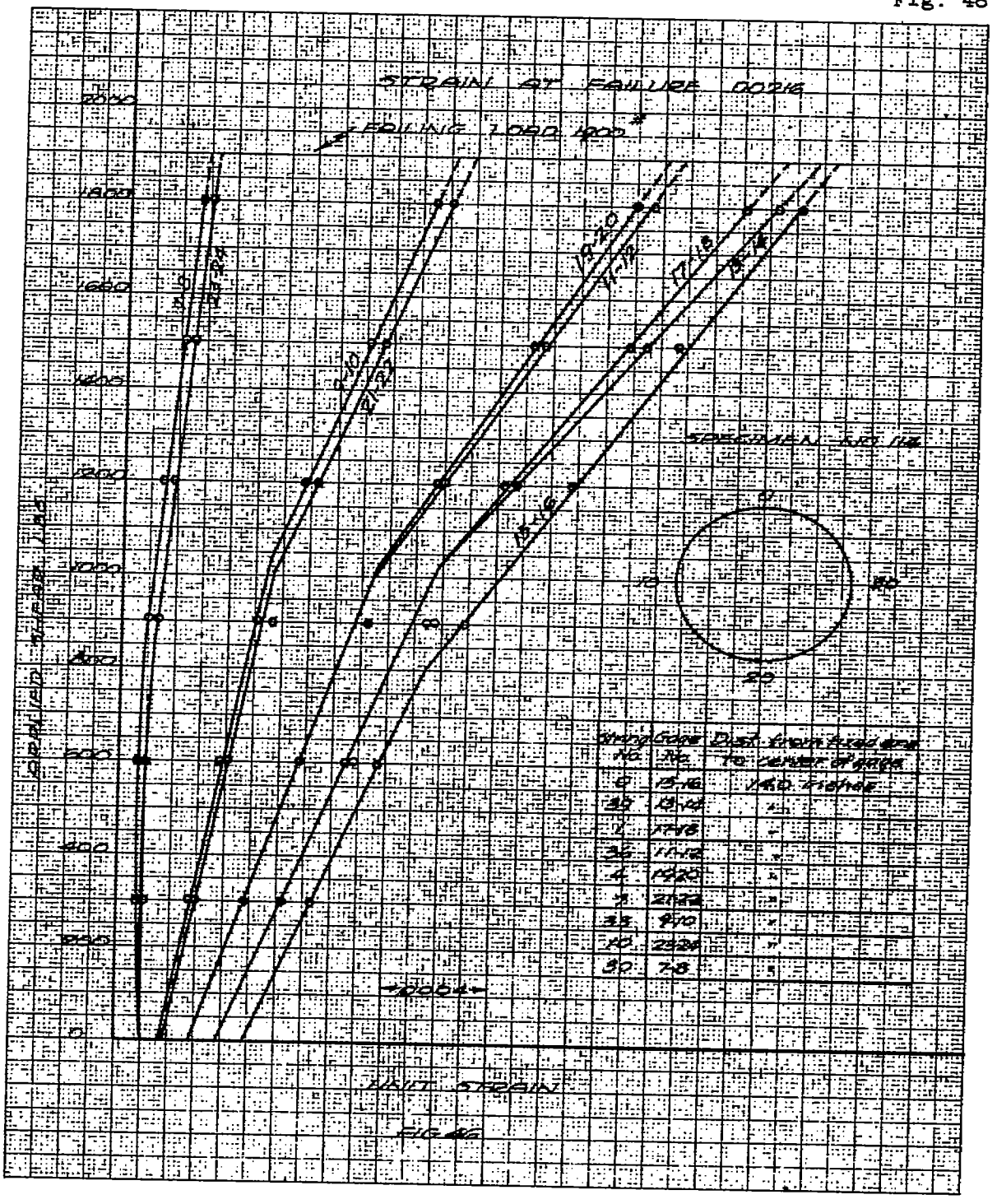

STRAIN AT FAILURE .0026
FAILING LOAD 1000 lbs
SPECIMEN AND ITS
Gage No.
Dist. from end of spec. to center of gage
Unit Stress
Unit Strain

Fig. 47

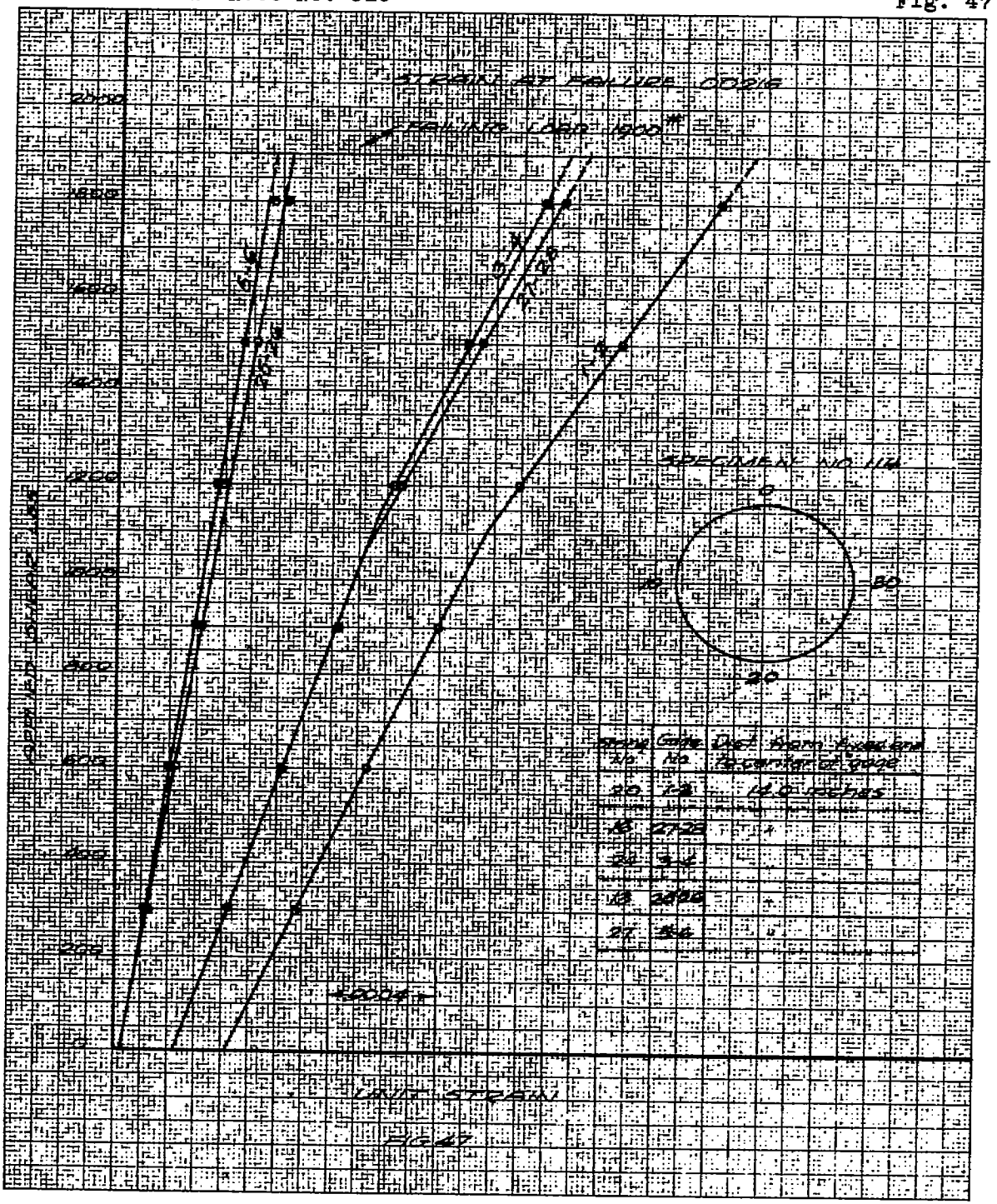
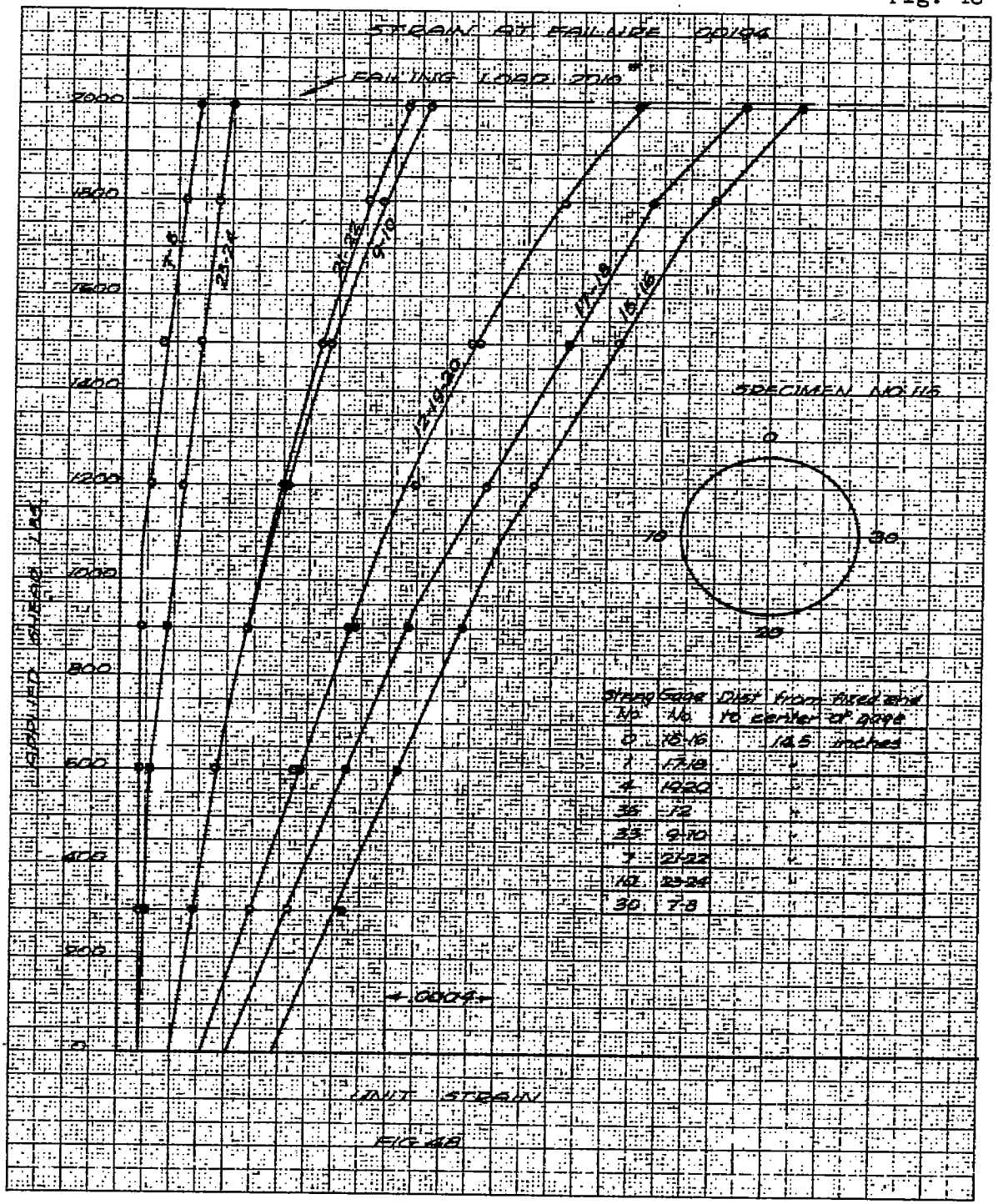
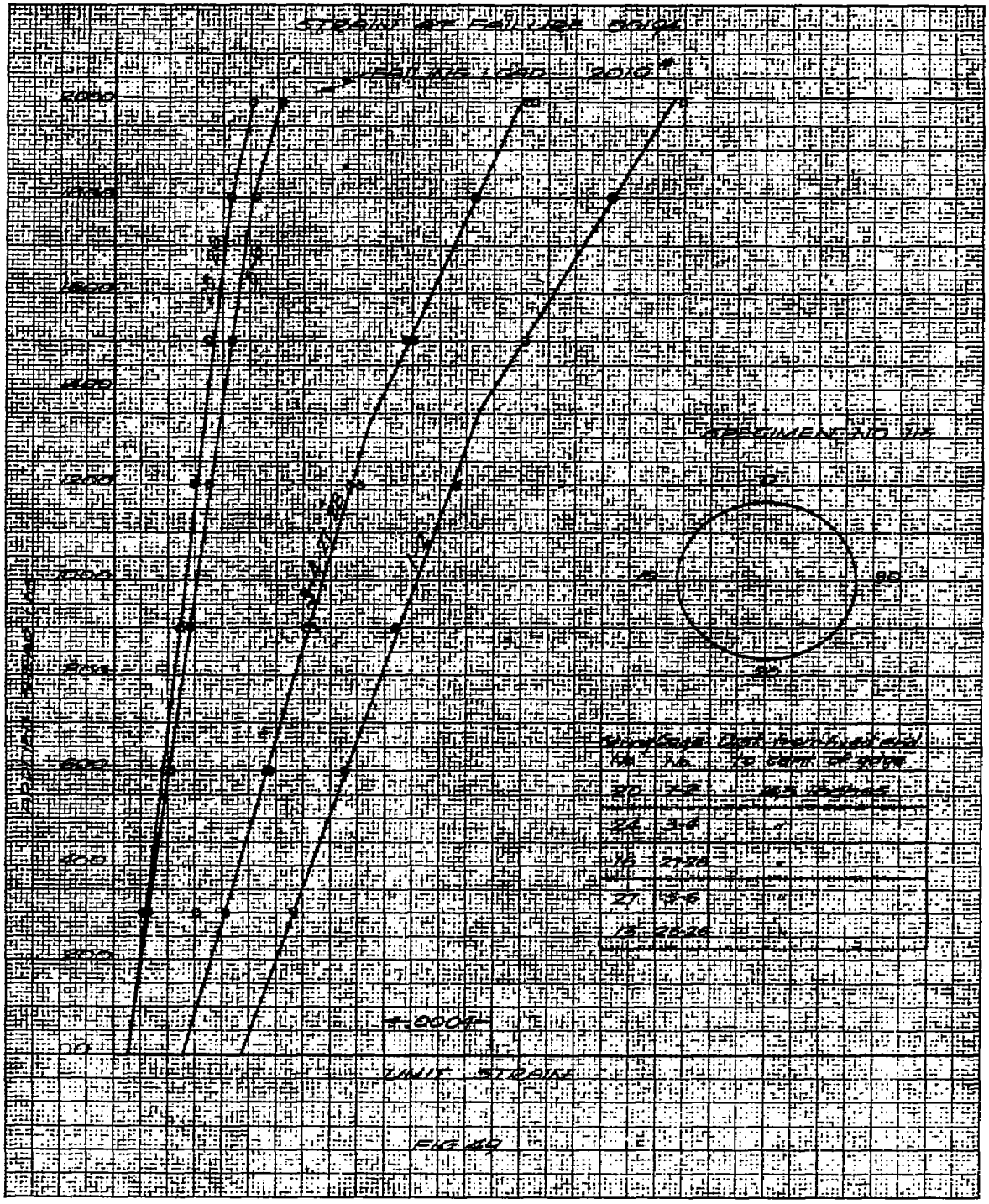
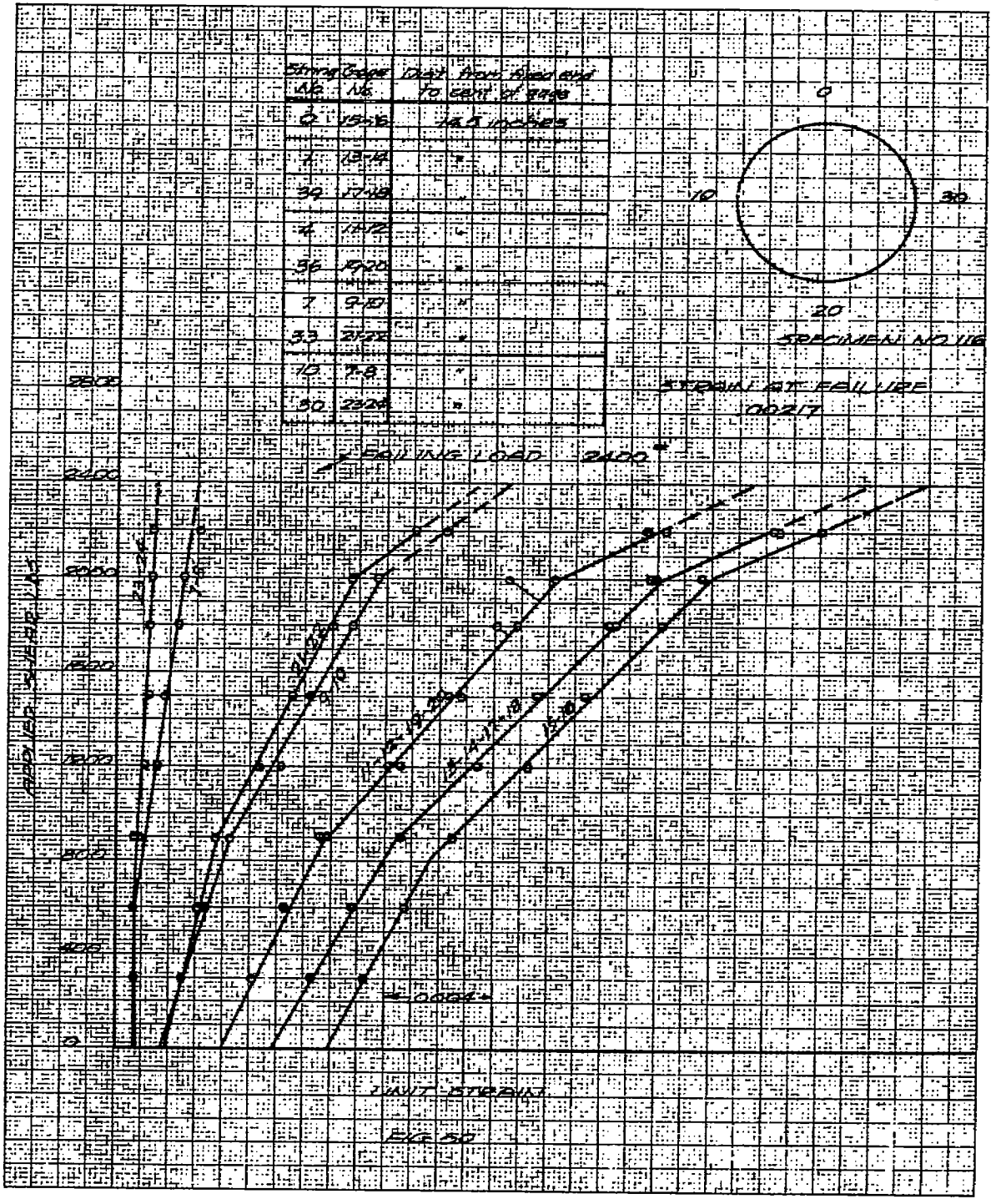
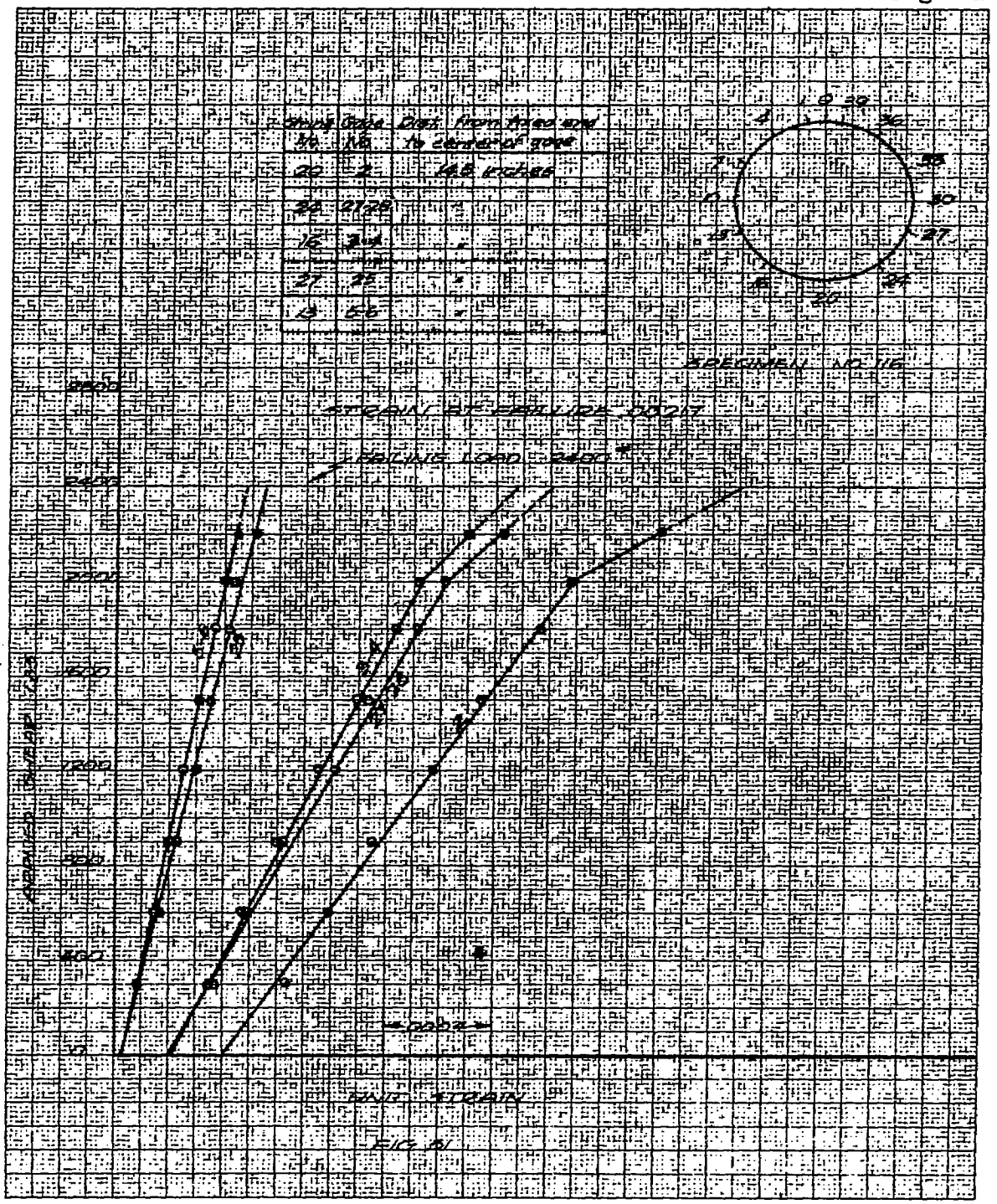

STRAIN AT FAILURE 0.0016
FAILURE LOAD 1900 lb
SPECIMEN NO. 118
0
20
40
60
80
Strain Gage No.
Dist. from Fixed end
Remarks
20
1.2
40
2.2
2.3
60
3.0
3.2
80
3.8
3.6
100
4.4
4.6
120
5.2
5.4
140
6.0
6.2
160
6.8
7.0
180
7.6
7.8
200
8.4
8.6
220
9.2
9.4
240
10.0
10.2
260
10.8
11.0
280
11.6
11.8
300
12.4
12.6
320
13.2
13.4
340
14.0
14.2

Fig. 48


STRAIN AT FAILURE .0094
FAILING LOAD 2110 lb
2000
1800
1600
1400
1200
1000
800
600
400
200
0
STRESS lb/sq in
+.0004
UNIT STRAIN
FIG 48
SPECIMEN NO 116
0
10
20
30
Strain Gage No
Dist from fixed end to center of gage
0 10,16 14.5 inches
1 17,18
4 19,20
35 12
33 9,10
7 21,22
10 23,24
30 7,8




SPECIMEN NO 116
STRAIN AT FAILURE
.002 IN/IN
2400 #
0
10
20
30



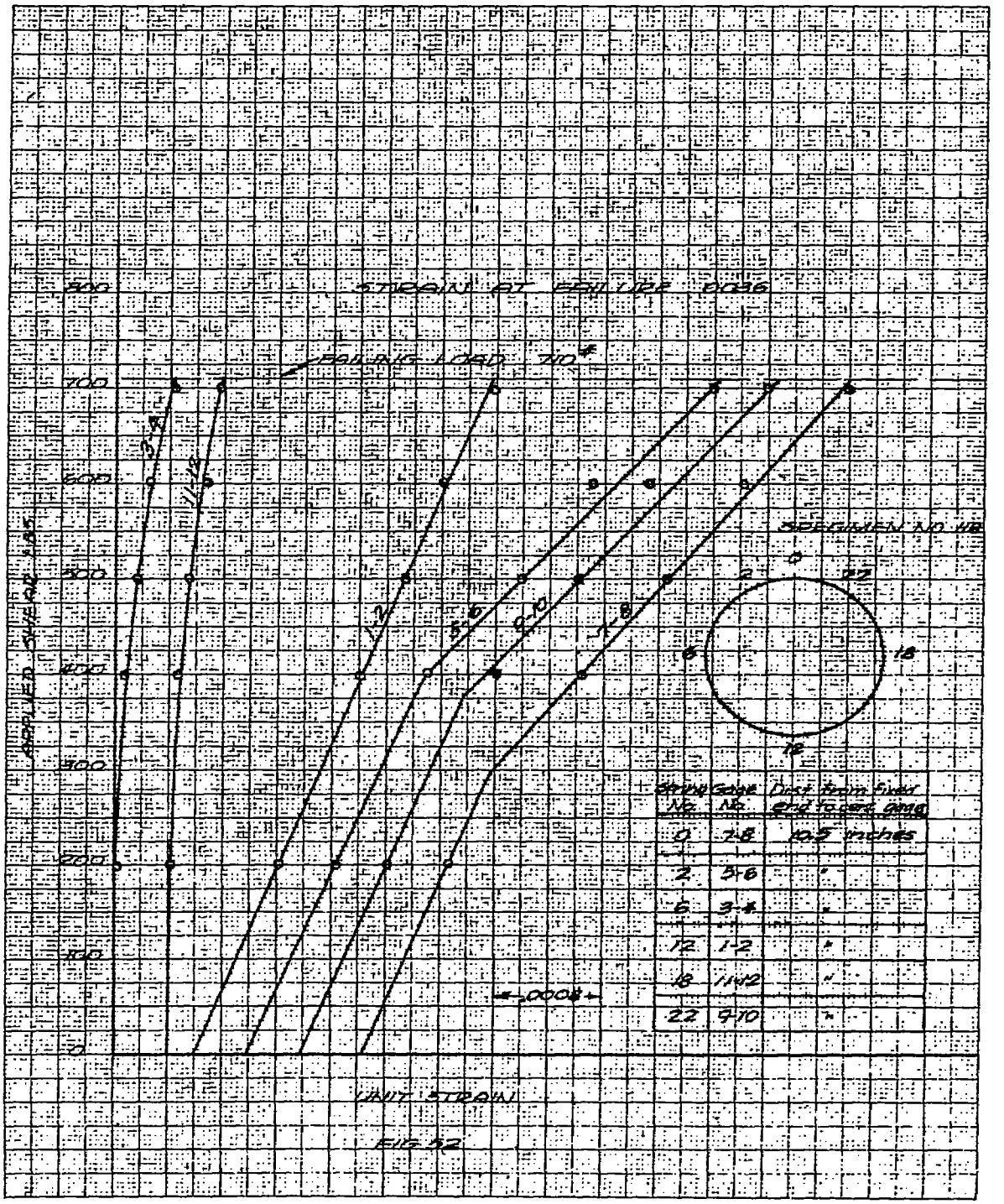
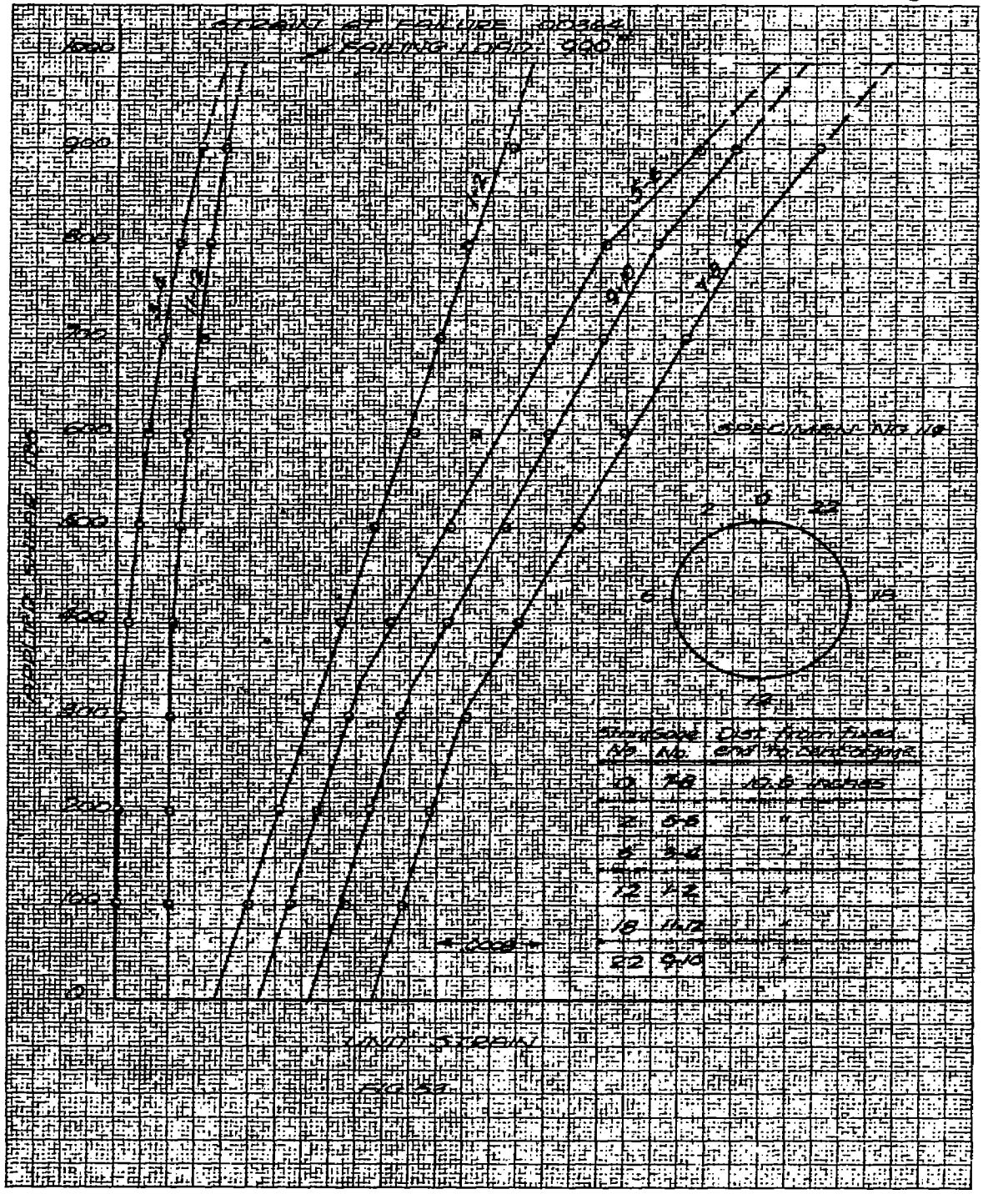
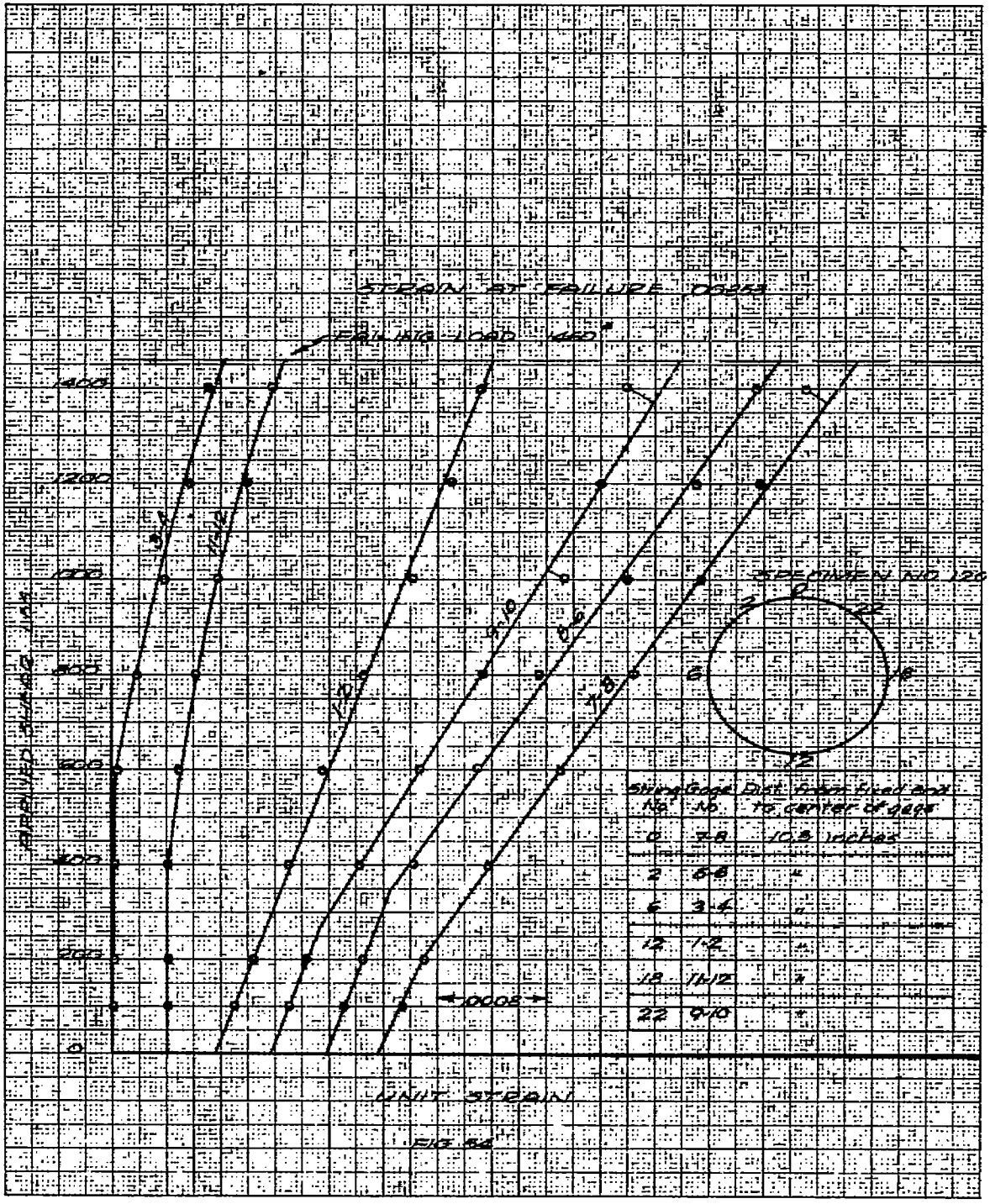

STRAIN AT ECCENTRICITY 0.236
POLAR LOAD 710#
APPLIED LOAD
500
400
300
200
100
0
UNIT STRAIN
SPECIMEN NO 118
0
2
6
12
18
22
Spring No
Gage No
Dist from Inner end to Gage Line
0 2-8 10.5 inches
2 3-6 "
6 3-4 "
12 1-2 "
18 11-12 "
22 9-10 "

FIG 52




STRAIN AT FAILURE 0.0033
FAILING LOAD 1460
1400
1200
1000
800
600
400
200
0
APPLIED LOAD
.0002
UNIT STRAIN
FIG 54
SPECIMEN NO 120
Strain Gage No.
Dist. from fixed end No. to center of gage
0 2-8 10.3 inches
2 6-8 "
6 3-4 "
12 1-2 "
18 11-12 "
22 9-10 "

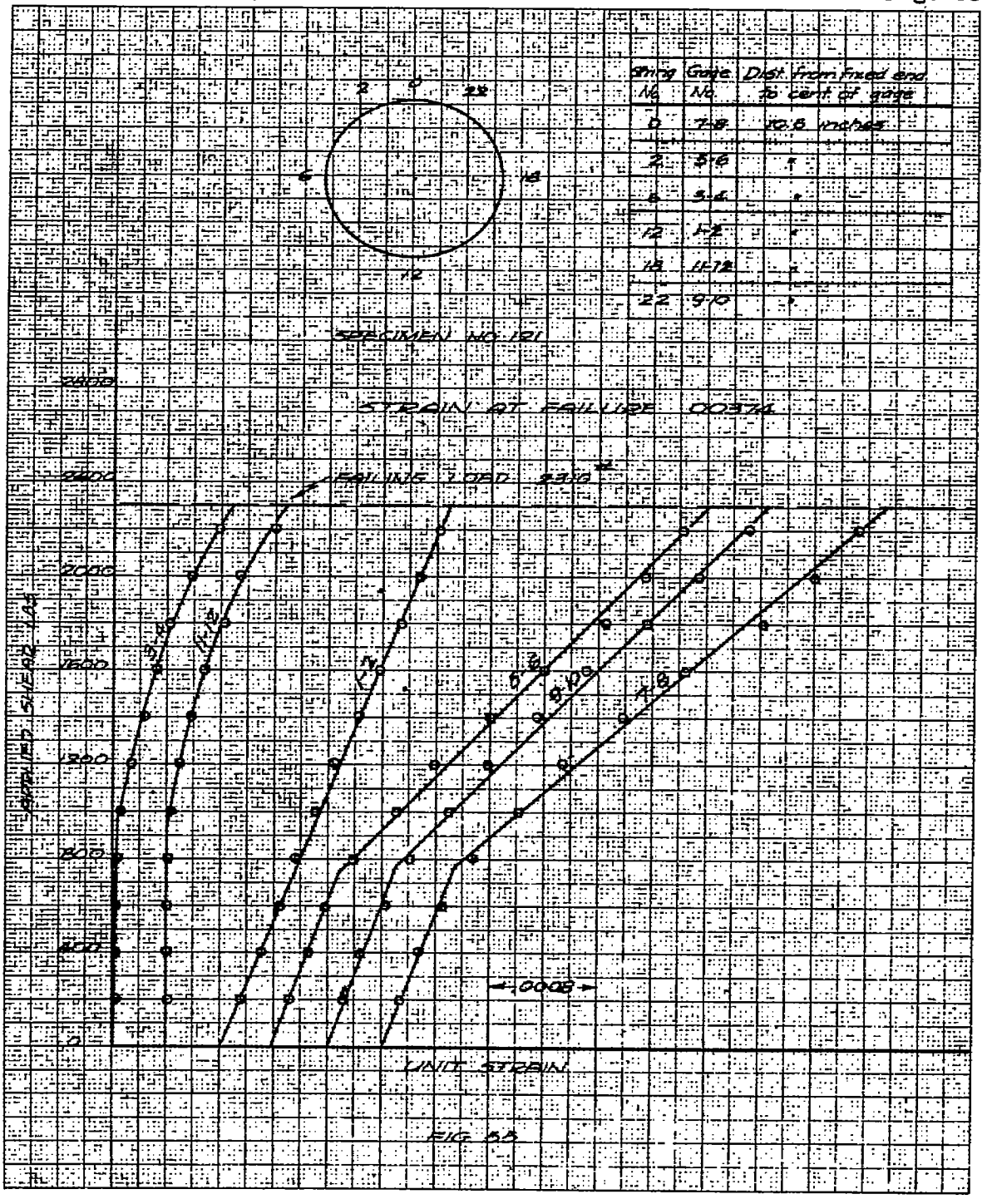
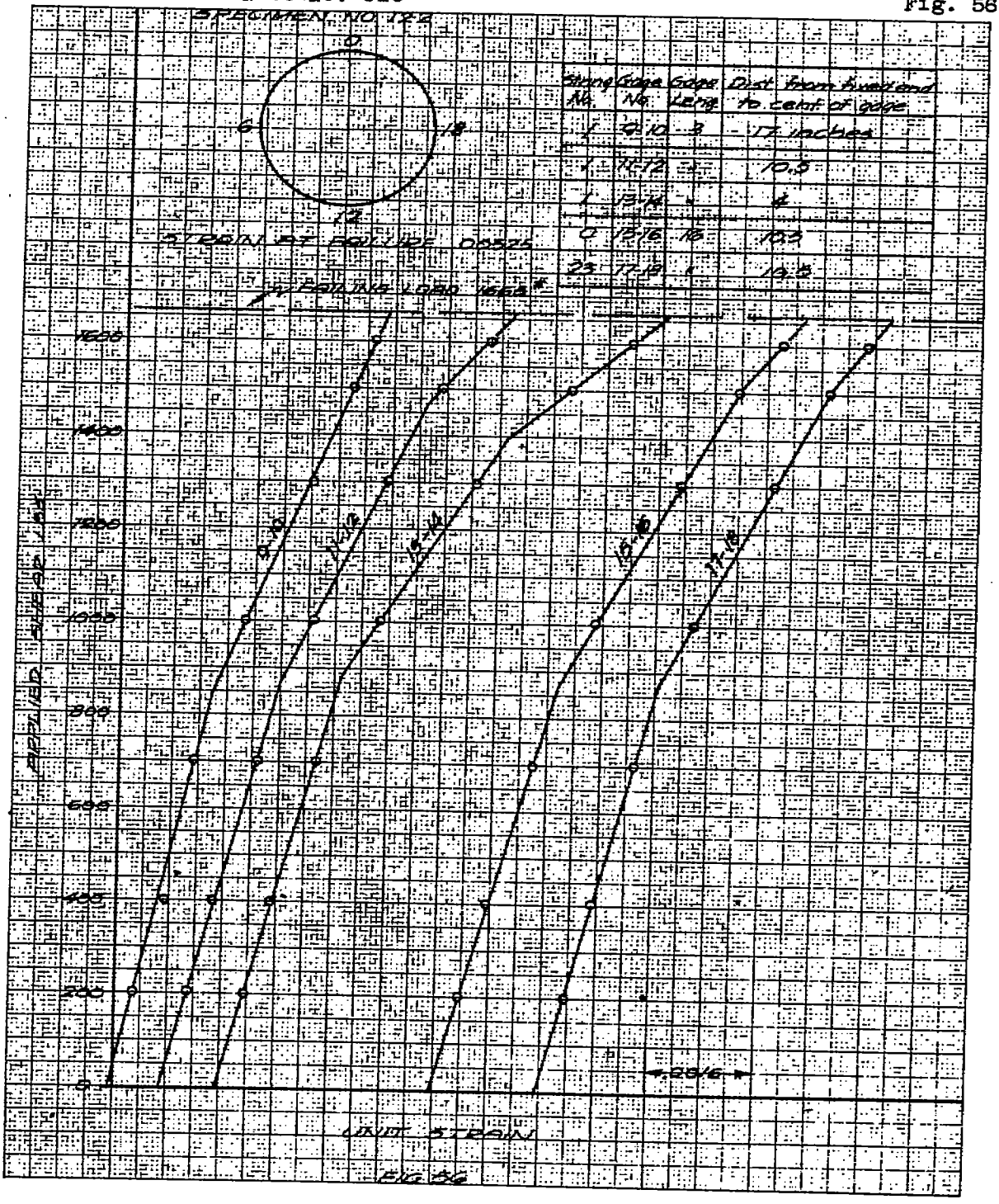
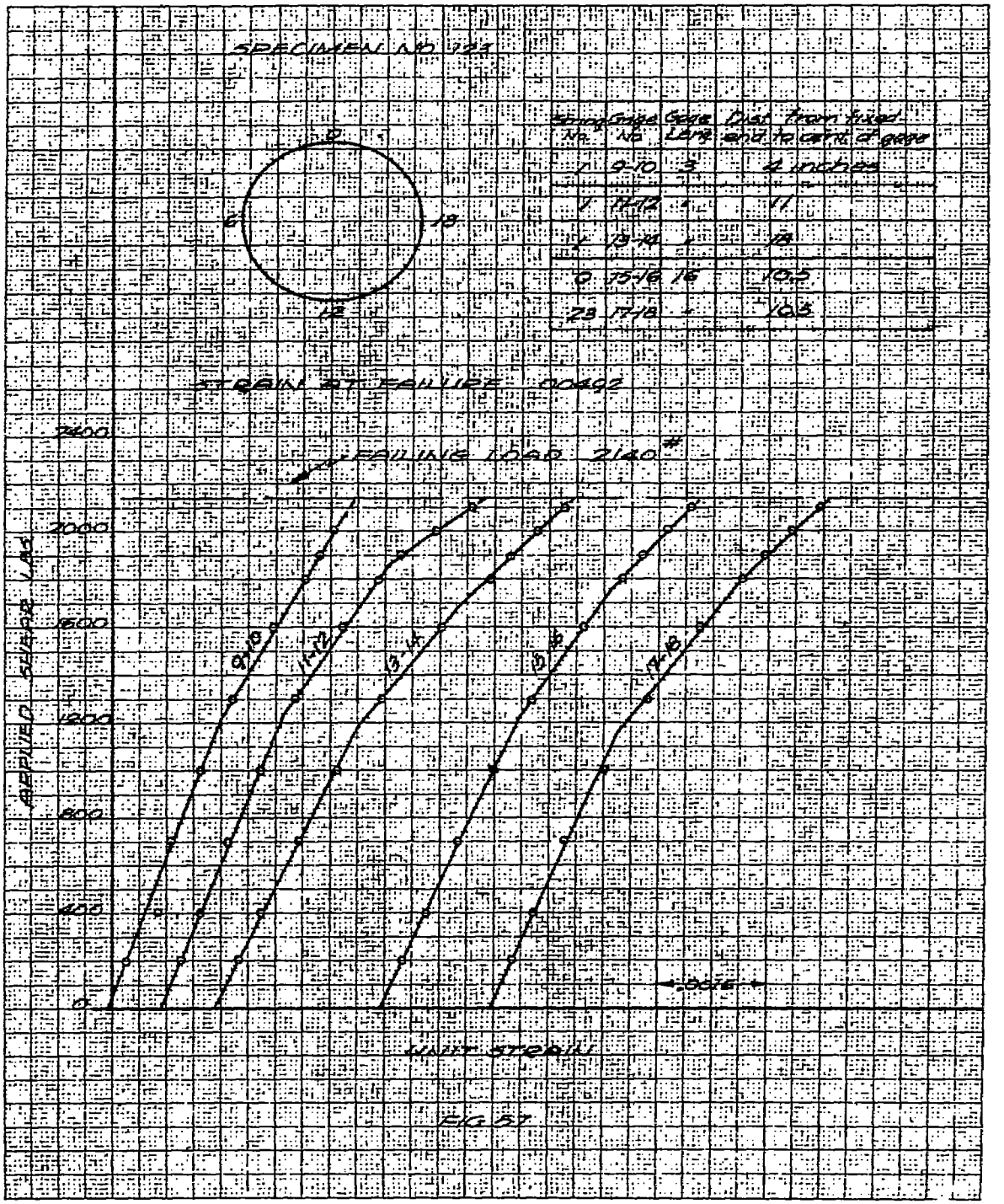

String No.
Gage No.
Dist. from Fixed end to cent. of gage
0 7-8 10.6 inches
2 5-6 "
4 3-4 "
12 1-2 "
18 11-12 "
22 9-10 "
SPECIMEN NO. 101
STRAIN AT FAILURE .0034
FAILURE LOAD 2310 #
STRESS - LBS. PER SQ. IN.
2400
2000
1600
1200
800
400
0
.0008
UNIT STRAIN
FIG. 55

Fig. 56


SPECIMEN NO. 122
Strain Gage No.
Gage Length
Dist. from Fixed end to Cent. of gage
1 0.10 3 17 inches
1 7.72 10.5
1 13.14 4
0 15.16 16 10.1
23 17.18 19.5
STRAIN AT EDGE HOLES
AT FAILING LOAD 1660
APPLIED STRESS - lbs.
1600
1400
1200
1000
800
600
400
200
0
-.2%
UNIT STRAIN
FIG 56


SPECIMEN NO 124
Strain Gage No. | Gage Length | Dist. from fixed end to center of gage
1 | 0-10 | 3 | 2 inches
2 | 11-12 | 1 | 11
3 | 13-14 | 1 | 18
4 | 15-16 | 16 | 10.5
5 | 17-18 | 1 | 10.5
STRAIN AT FAILURE .0042
FAILING LOAD 2140 #
APPLIED LOAD LBS.
2400
2000
1600
1200
800
400
0
.002
UNIT STRAIN
FIG 57

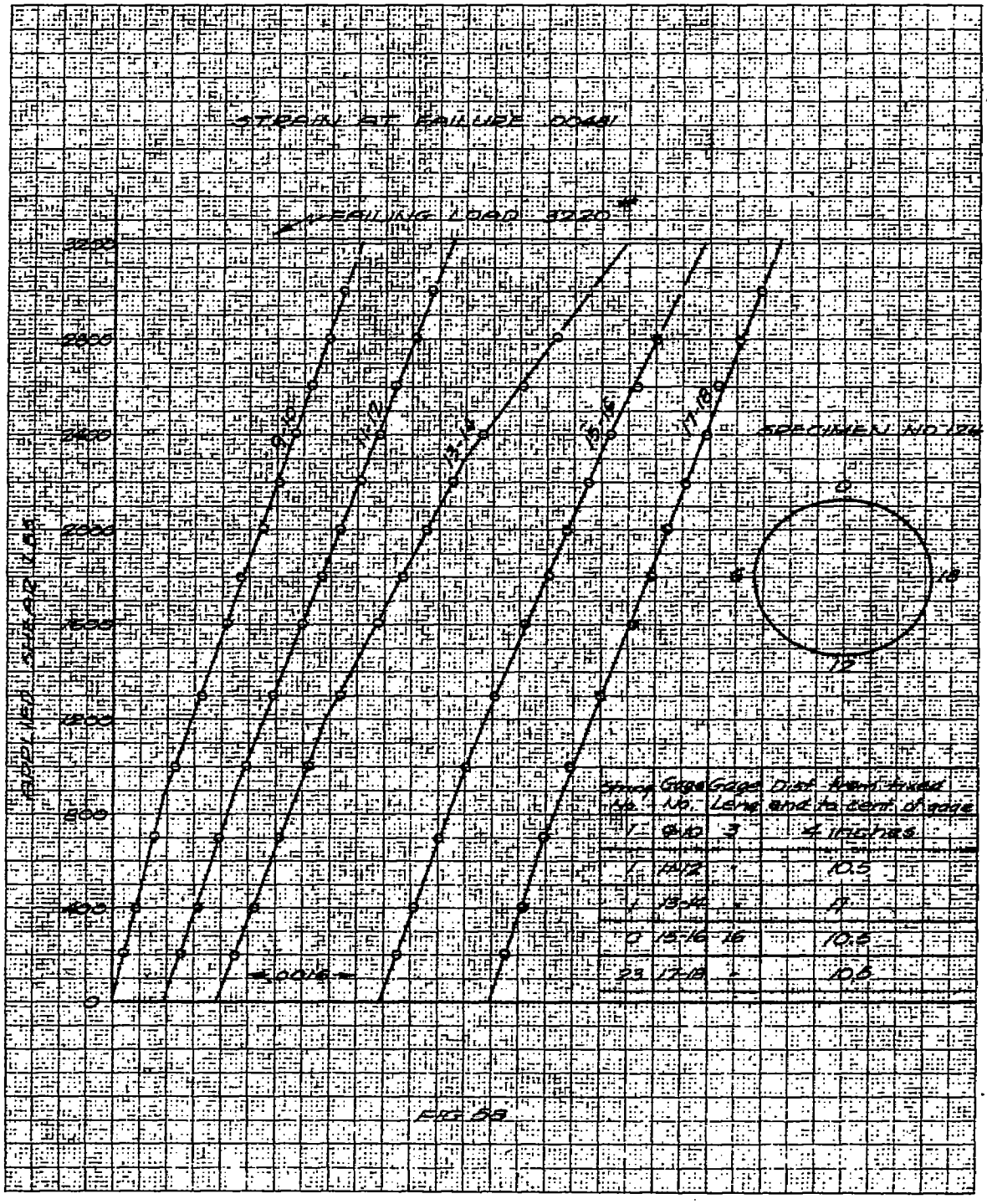
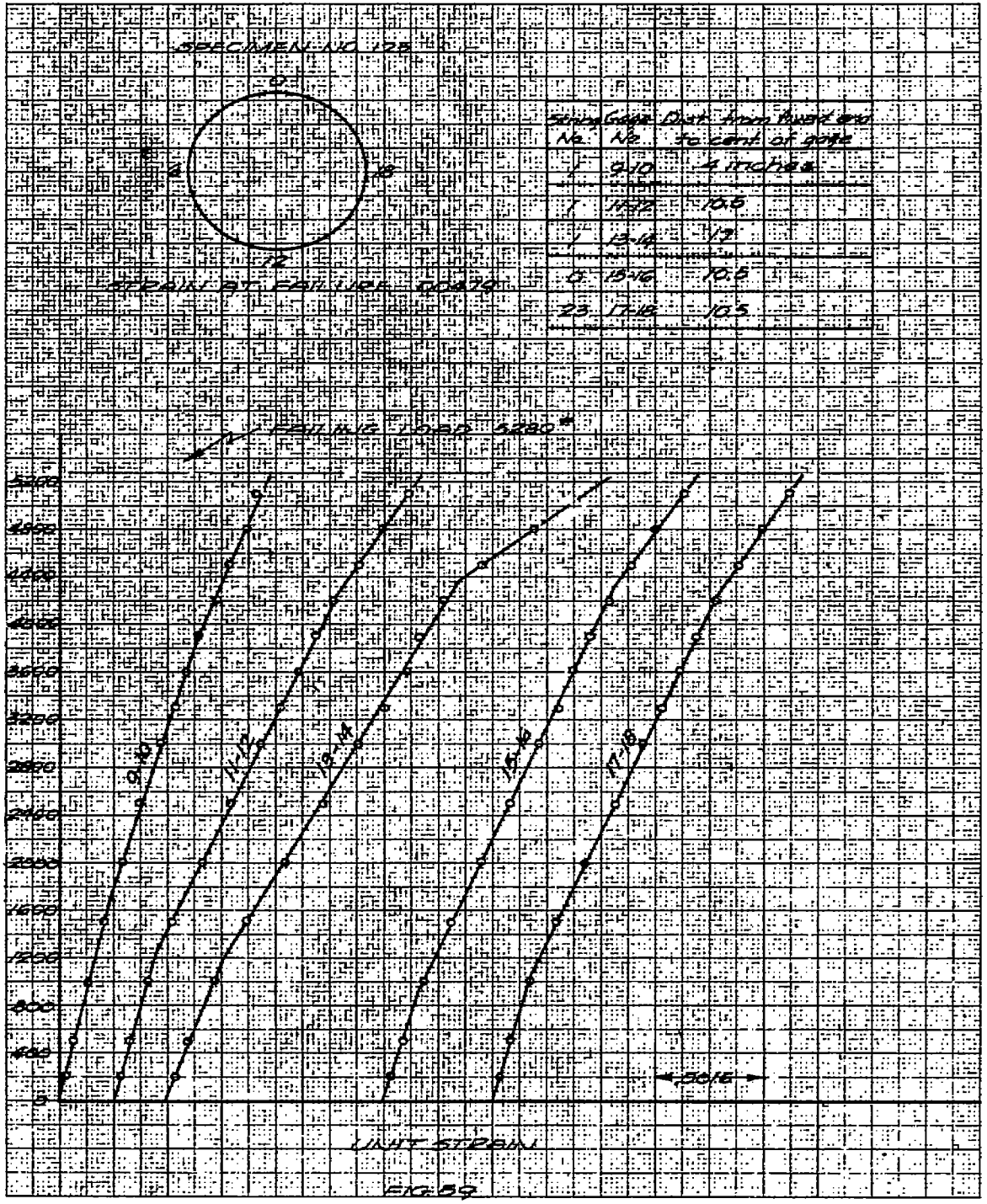
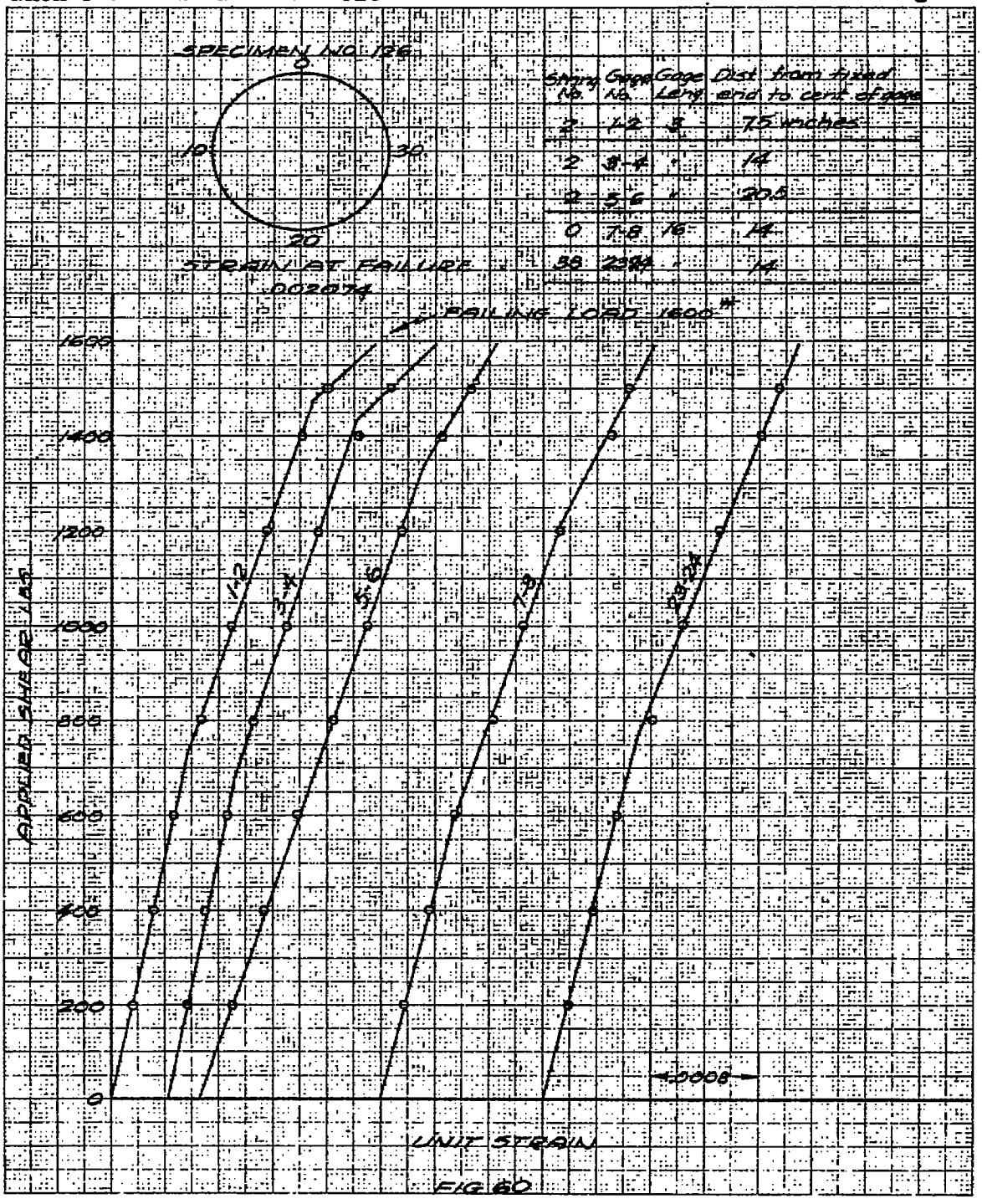
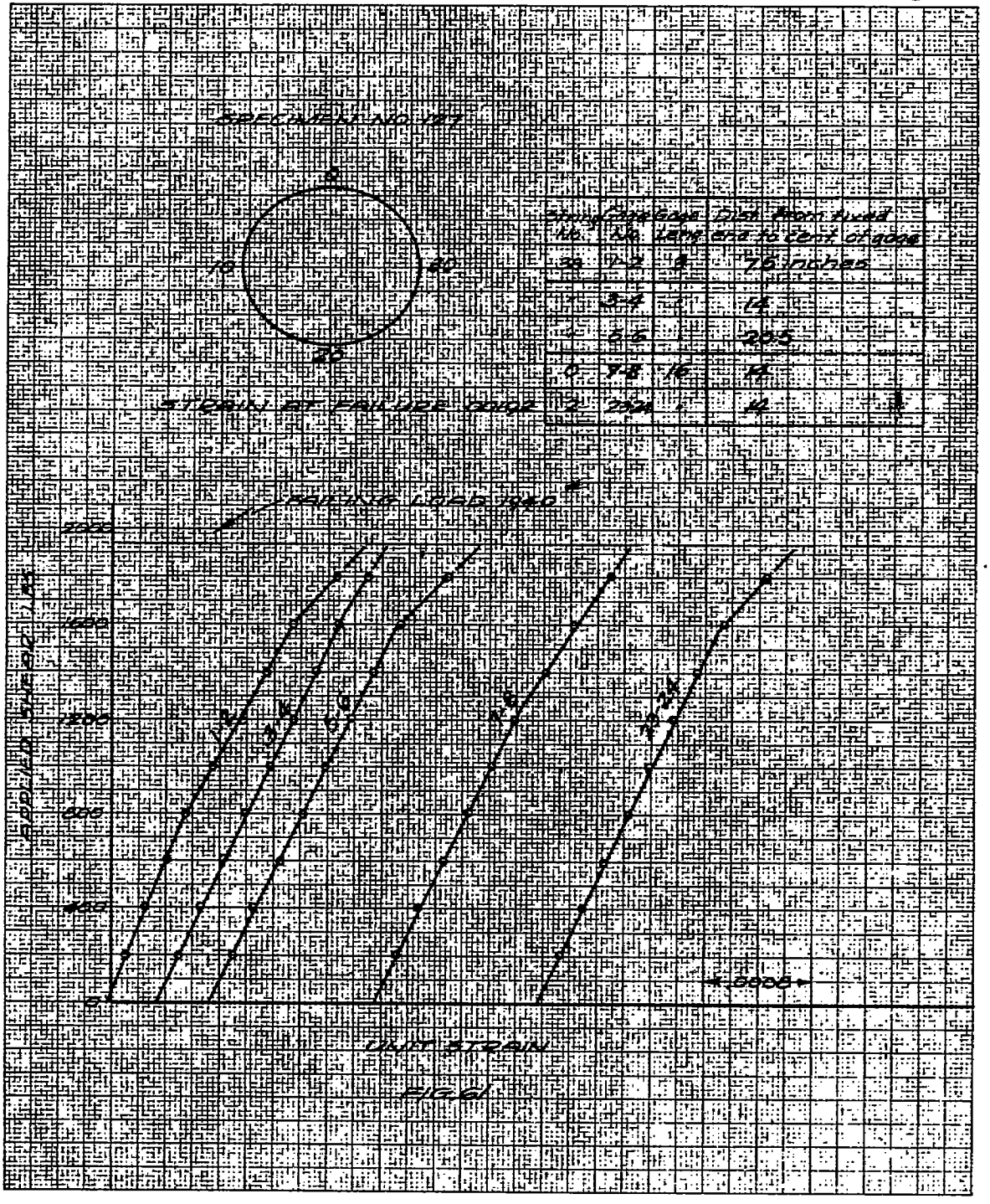

STRESS AT FAILURE 2040
FAILING LOAD 2320
SPECIMEN NO 104
STRESS IN POUNDS PER SQUARE INCH
3200
2800
2400
2000
1600
1200
800
400
0
0.002
0
6
12
18
Spring No.
Gage No.
Gage Length
Dist. from end of specimen to center of gage
1
9-10
3
4 inches
2
11-12
1
10.5
3
13-14
2
9
4
15-16
16
10.5
5
17-18
2
10.5

Fig. 58


SPECIMEN NO. 12B
STRAIN AT FAILURE 0.0079
FAILING LOAD 5280 #
Strain Gage No. 1
Strain Gage No. 2
Dist. from Rivet end to cent. of gage
4 inches
10.5
12
10.5
10.5
0.0016
UNIT STRAIN
FIG. 59


SPECIMEN NO. 126
Strip No.
Gage No.
Gage Length
Dist. from fixed end to cent. of gage
2 1-2 2 7.5 inches
2 3-4 " 14
2 5-6 " 20.5
0 7-8 16 14
39 2384 " 14
STRAIN AT FAILURE .002074
FAILING LOAD 1600 #
1600
1400
1200
1000
800
600
400
200
0
APPLIED STRESS LBS.
.0008
UNIT STRAIN
FIG 60


Specimen No. 127
Gage No.
Gage distance from fixed end to center of gage
76 inches
Stream at failure gage
Failure load, 2440 lb
Stress, lb/sq in.
Cut stream
Fixed

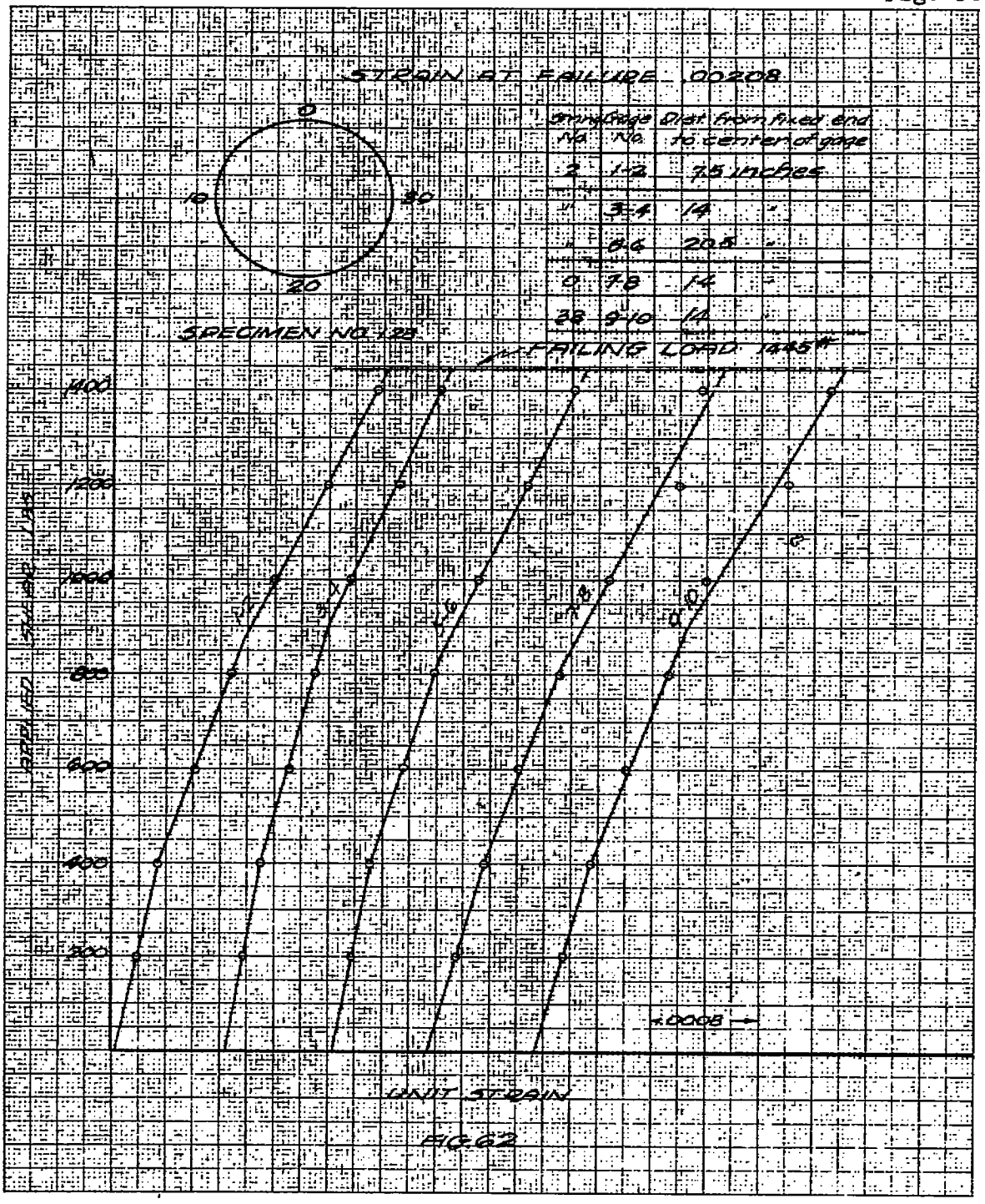

STRAIN AT FAILURE .00208
Strain Gage No.
Dist. from fixed end to center of gage
2 1/2 7.5 inches
3 4 14
4 6 20.5
0 7 8 14
28 9 10 14
SPECIMEN NO 128
FAILING LOAD 1445#
1400
1200
1000
800
600
400
200
.0005
UNIT STRAIN
APPLIED LOAD

FIG 62

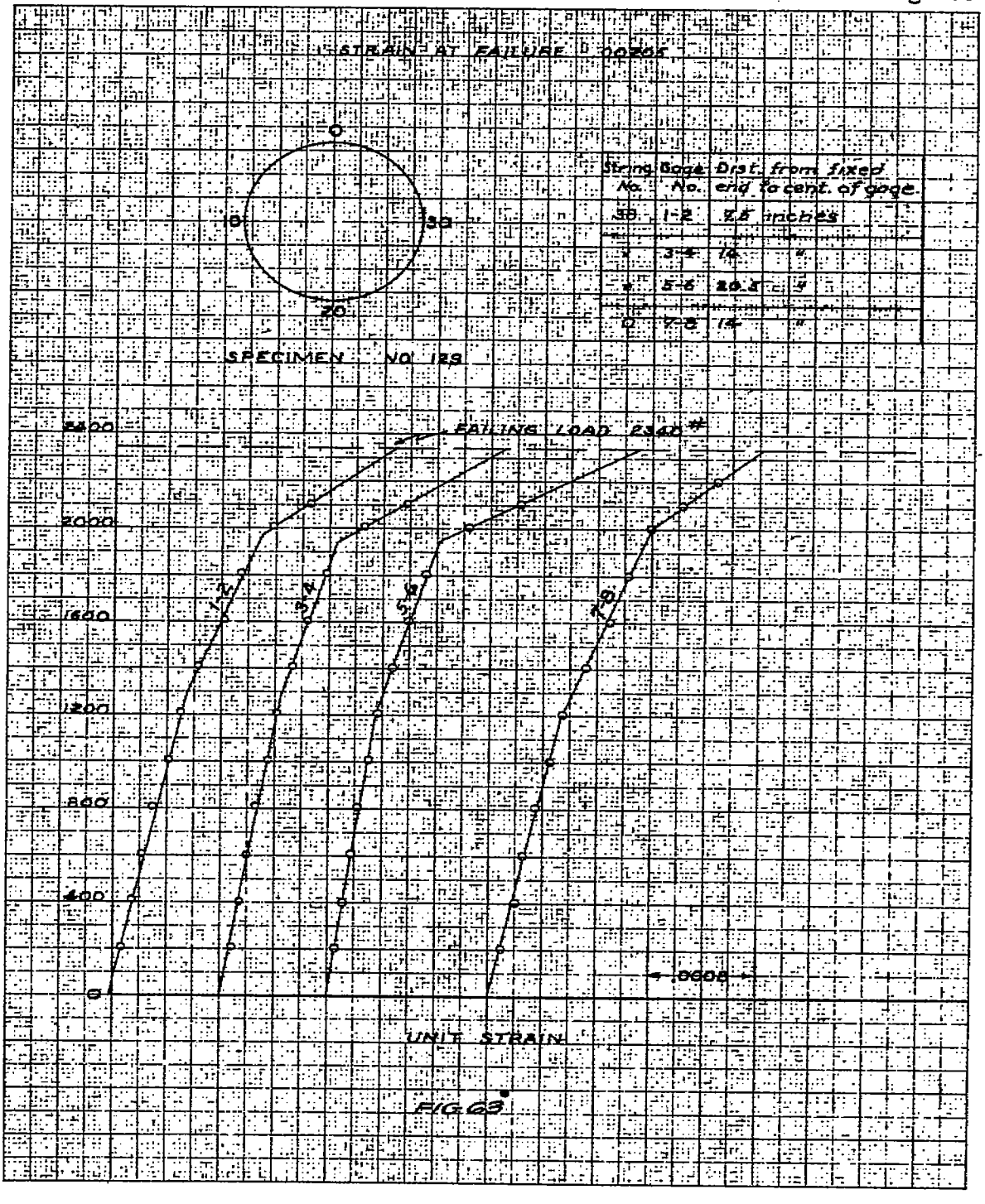

STRAIN AT FAILURE .00205
0
10
30
20
String No. | Gage No. | Dist. from fixed end to cent. of gage.
30 | 1-2 | 2.5 inches
" | 3-4 | 14 "
" | 5-6 | 20.5 "
0 | 7-8 | 14 "
SPECIMEN NO 128
FAILING LOAD 2340 #
2400
2000
1600
1200
800
400
0
.0006
UNIT STRAIN

FIG 63

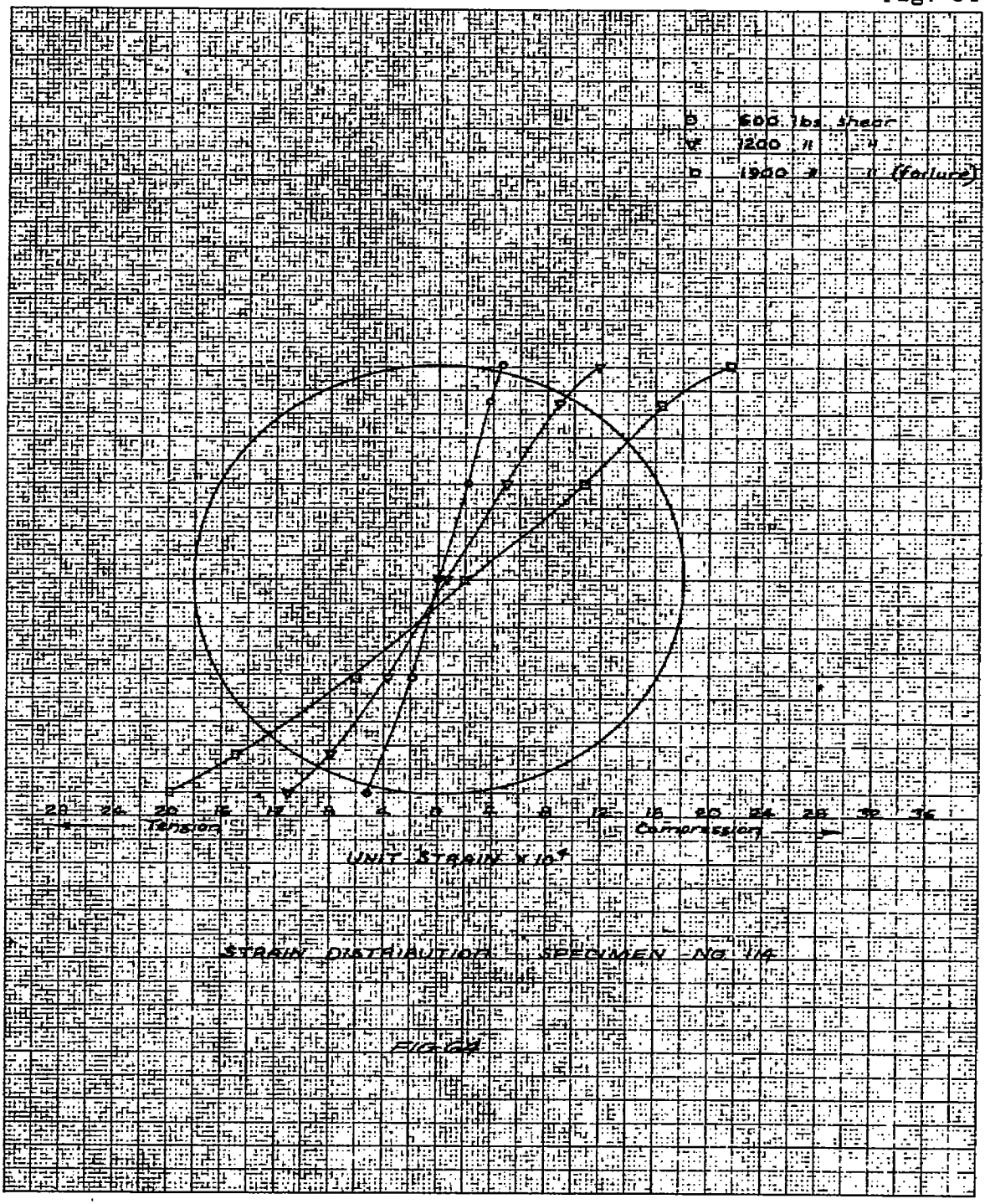
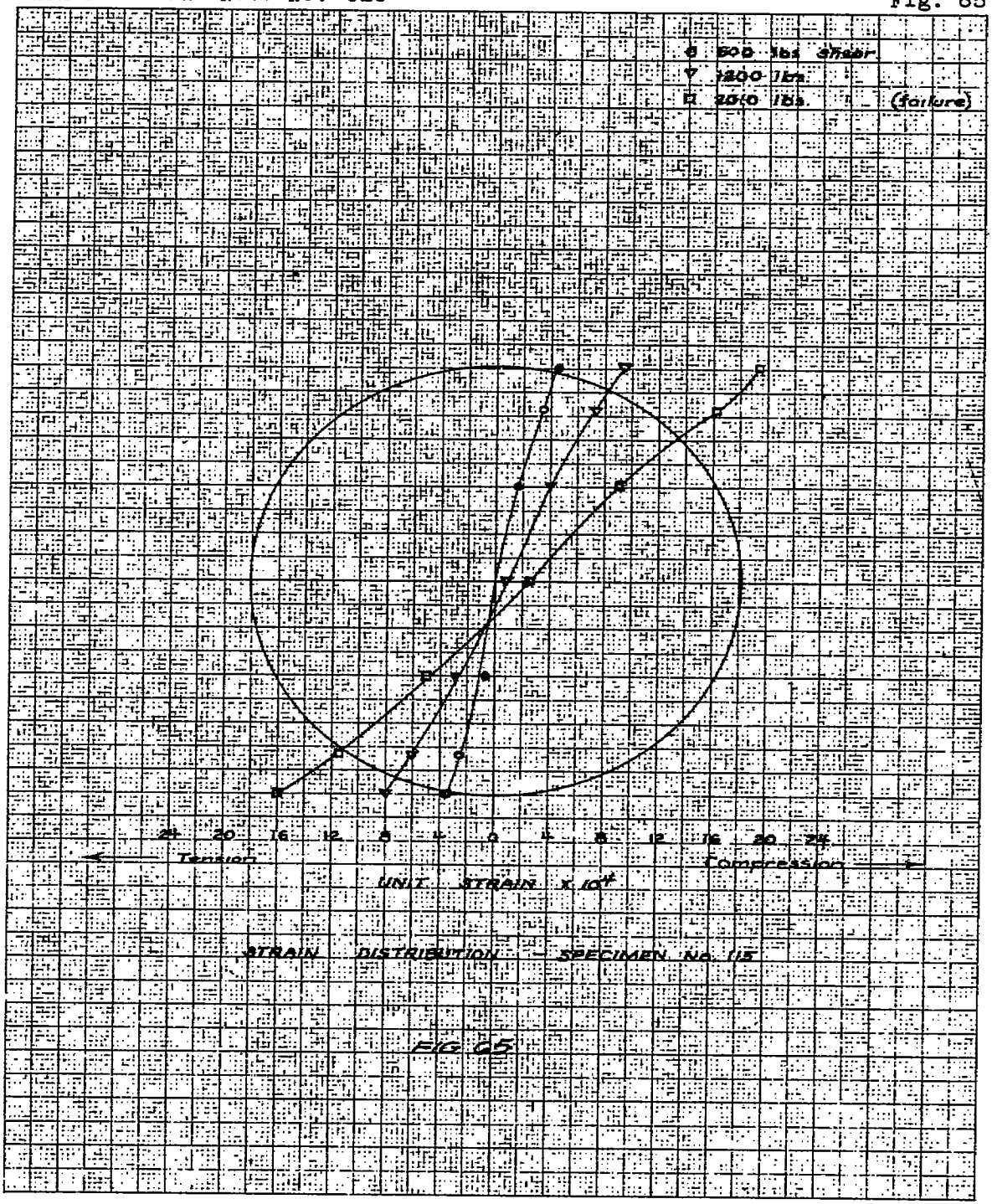
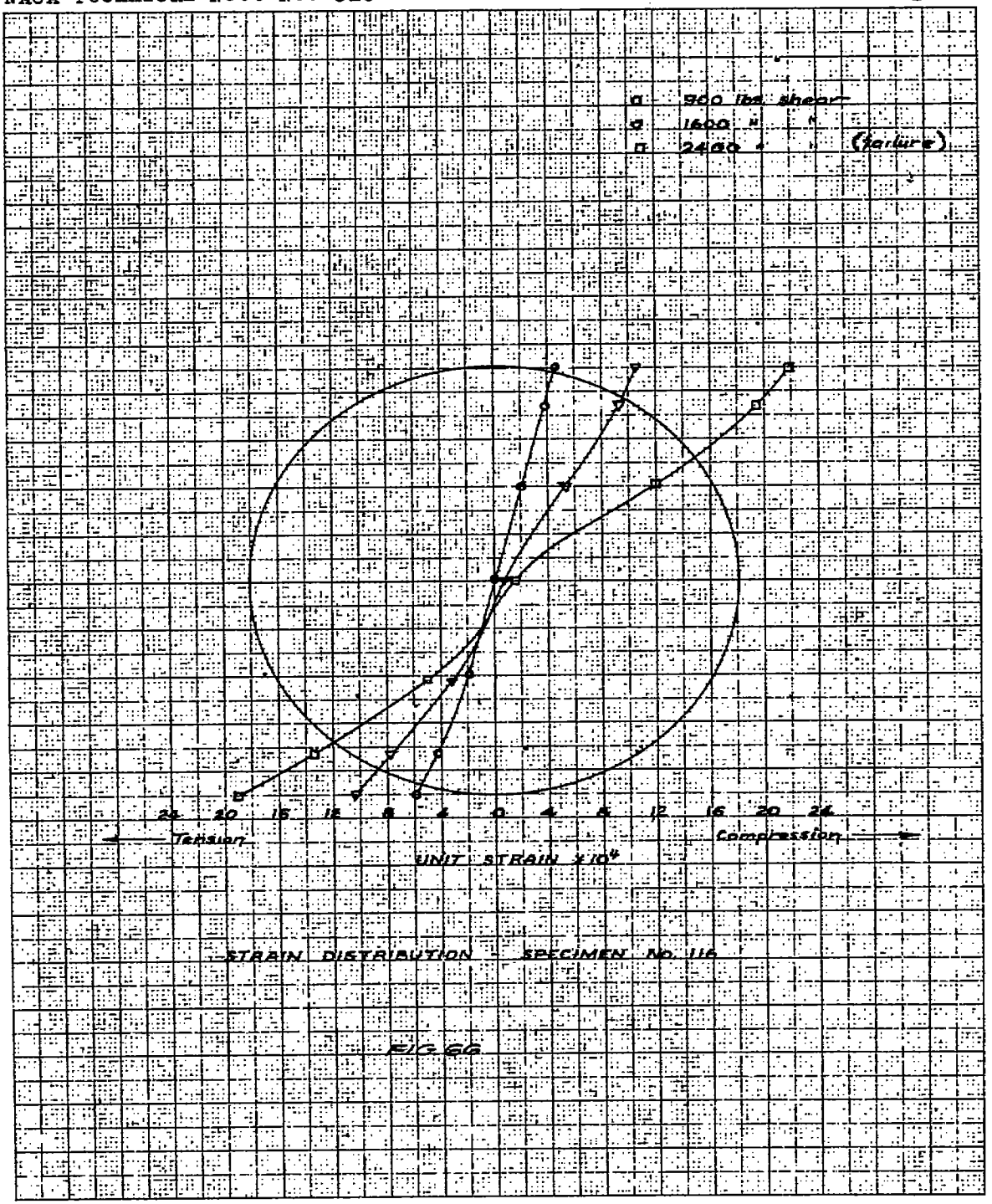

S 600 lbs. shear
V 1200 " "
D 1900 " " (failure)
Tension
Compression
UNIT STRAIN x 10⁻⁶
STRAIN DISTRIBUTION SPECIMEN NO. 114

Fig. 65



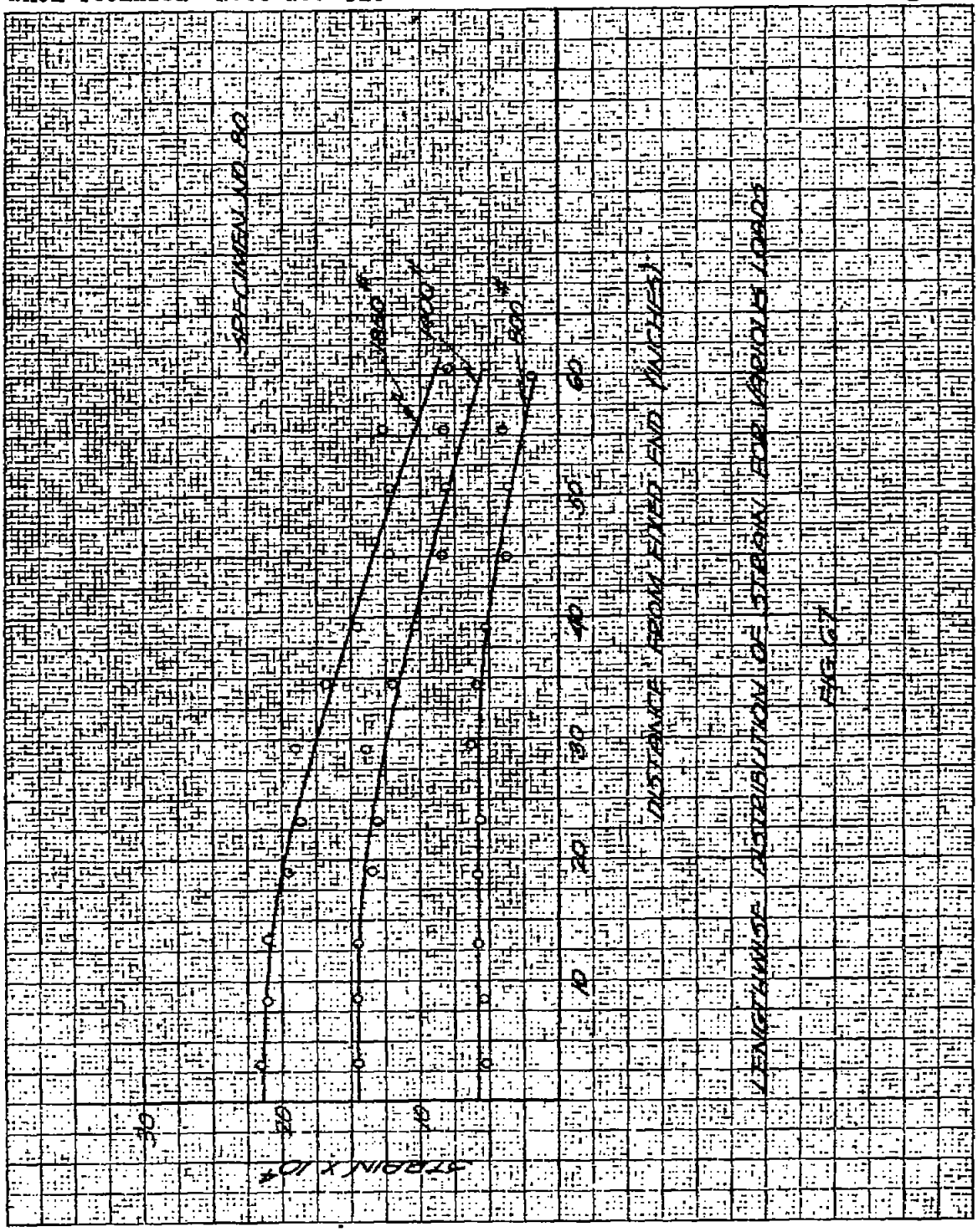
STRAIN DISTRIBUTION - SPECIMEN NO. 115

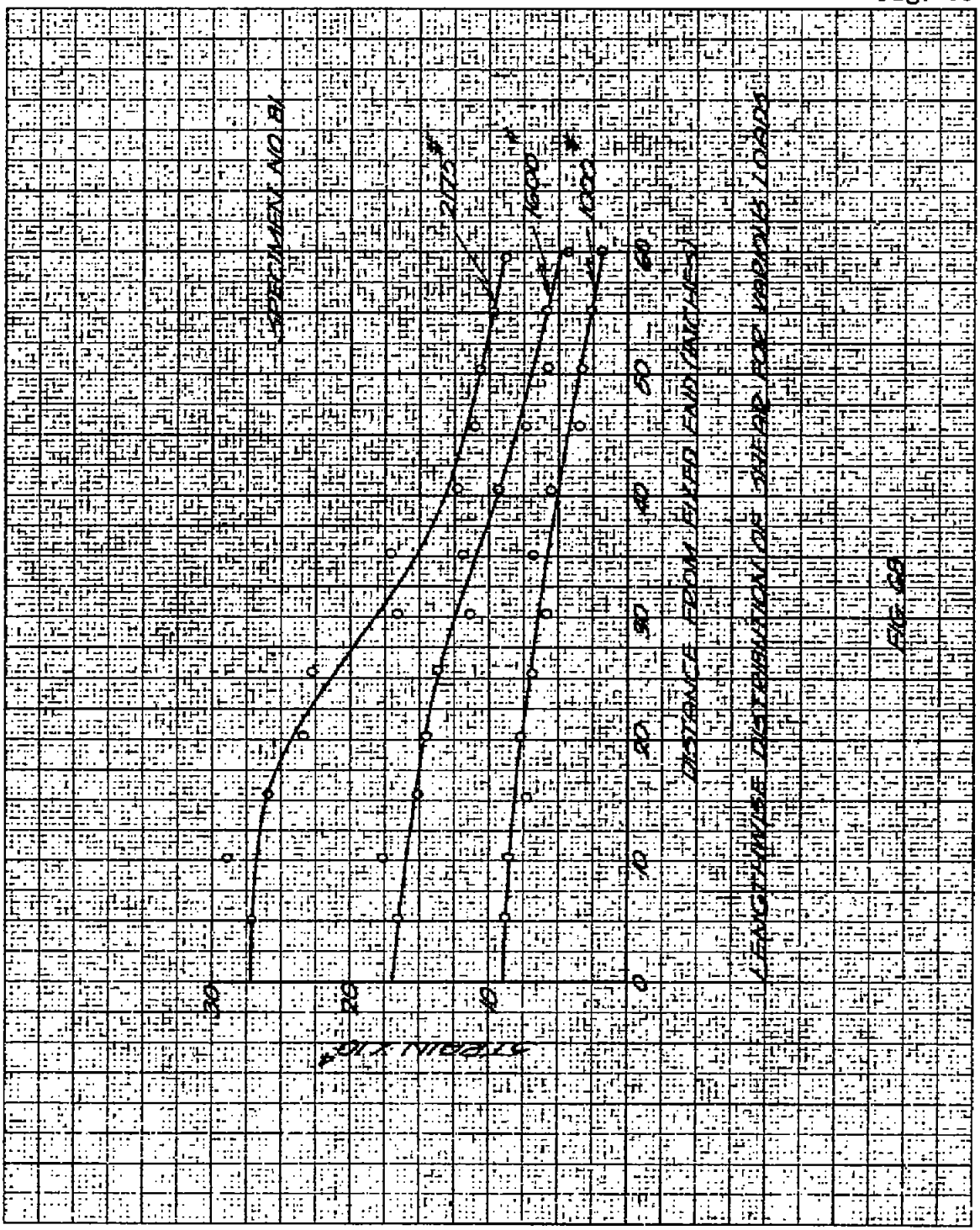
FIG. 65

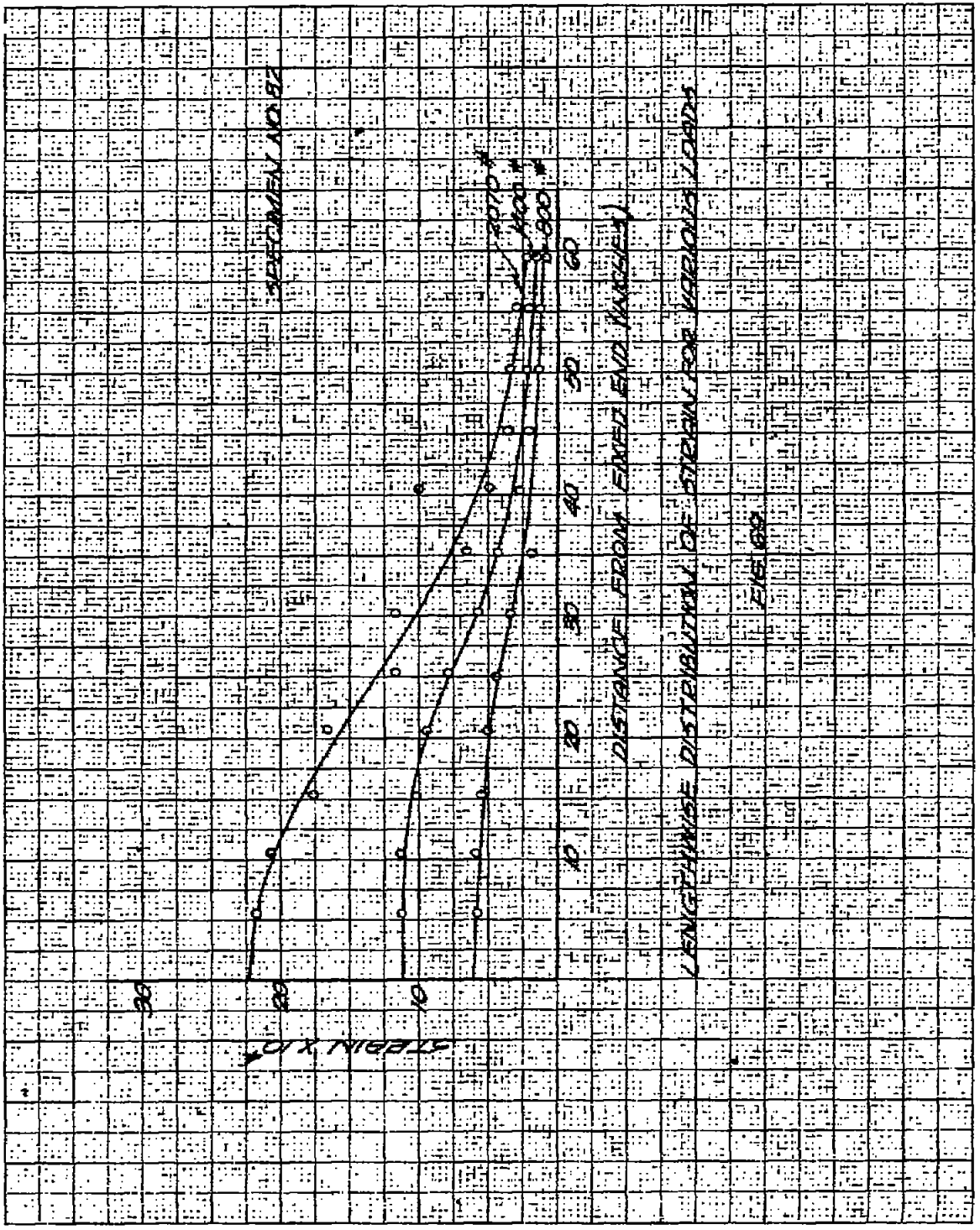


STRAIN DISTRIBUTION - SPECIMEN NO. 116

FIG 66


LENGTHWISE DISTRIBUTION OF STRAIN FOR VARIOUS LOADS
DISTANCE FROM FIXED END, INCHES
STRAIN X 10⁶
SPECIMEN NO. 80
600 #
800 #
1000 #
1200 #
1400 #
1600 #
1800 #
2000 #
2200 #
2400 #
2600 #
2800 #
3000 #
3200 #
3400 #
3600 #
3800 #
4000 #
4200 #
4400 #


SPECIMEN NO. B1
2000
1500
1000
500
0
10
20
30
40
50
60
DISTANCE FROM FILLED END (INCHES)
LEAKAGE RATE (CUBIC INCHES PER MINUTE)
FIG. 68


SPECIMEN NO. 82
2070 #
1420 #
800 #
LONGITUDINAL DISTRIBUTION OF STRAIN FOR VARIOUS LOADS
DISTANCE FROM ENCASED END, INCHES
60
50
40
30
20
10
0
10
20
30
OX RIVETS

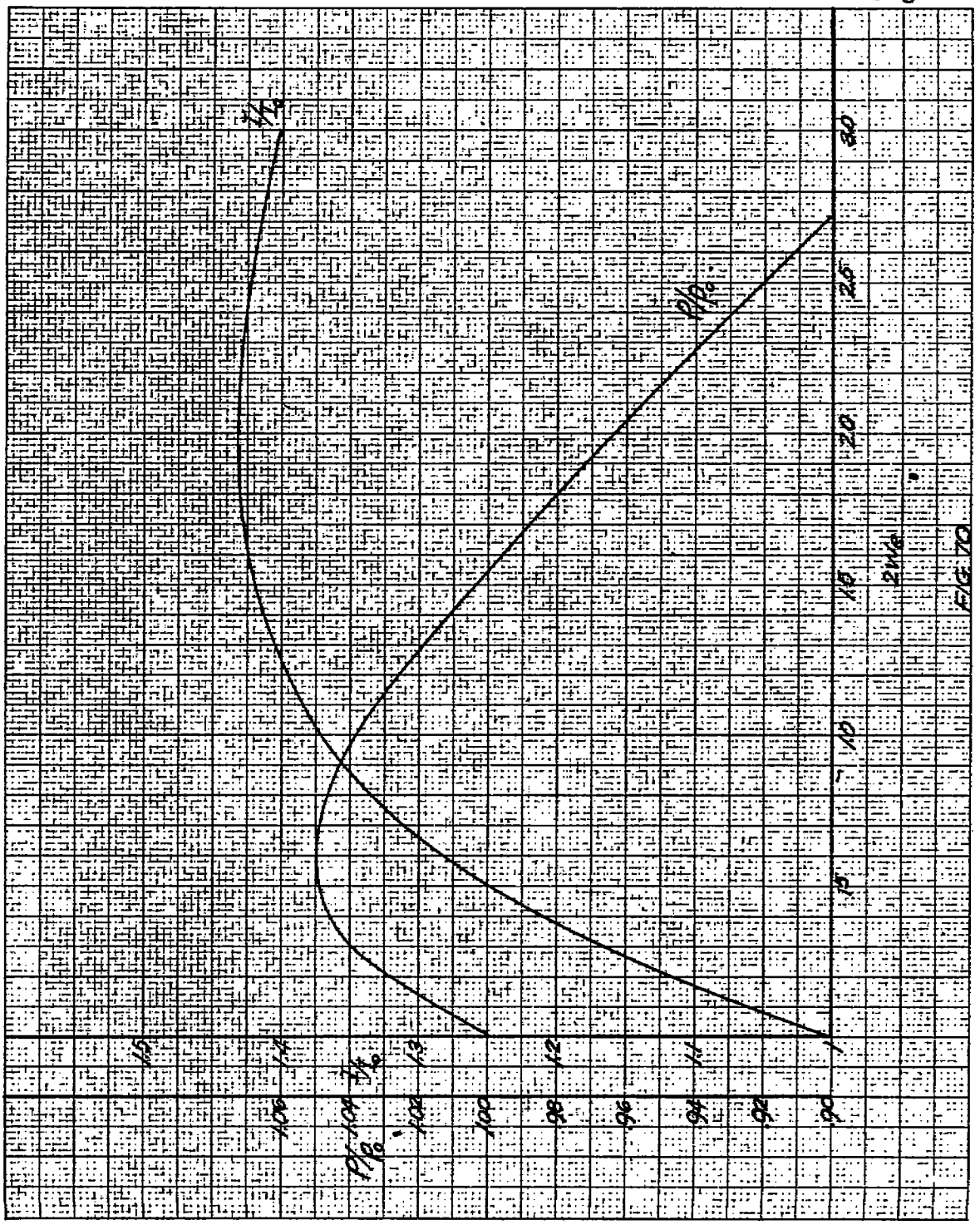


FIG 70
80
2.5
2.0
1.5
2 1/4
1.0
1
1/2
1/4
0
1/8
1/4
1/2
1
2
4
8
16
32
64
96
98
100
102
104
106
108
110
112
114
116

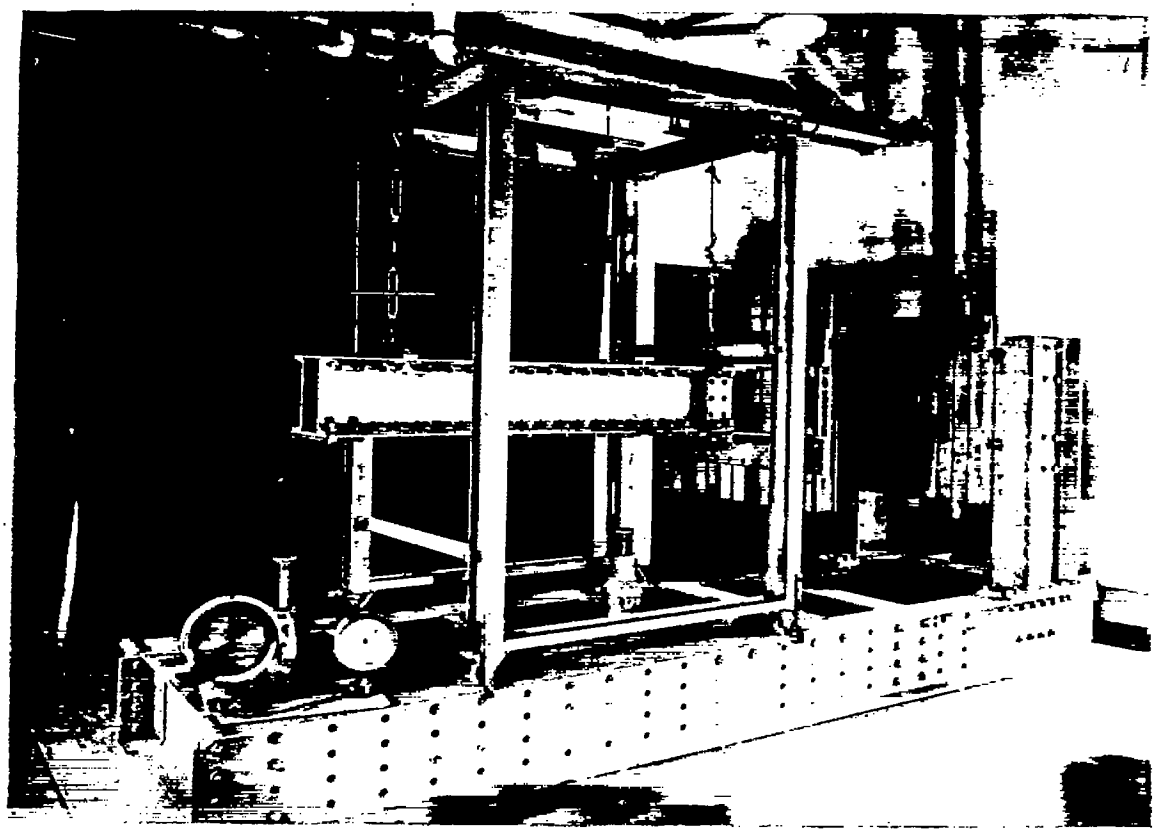


Fig. 71

Testing apparatus.

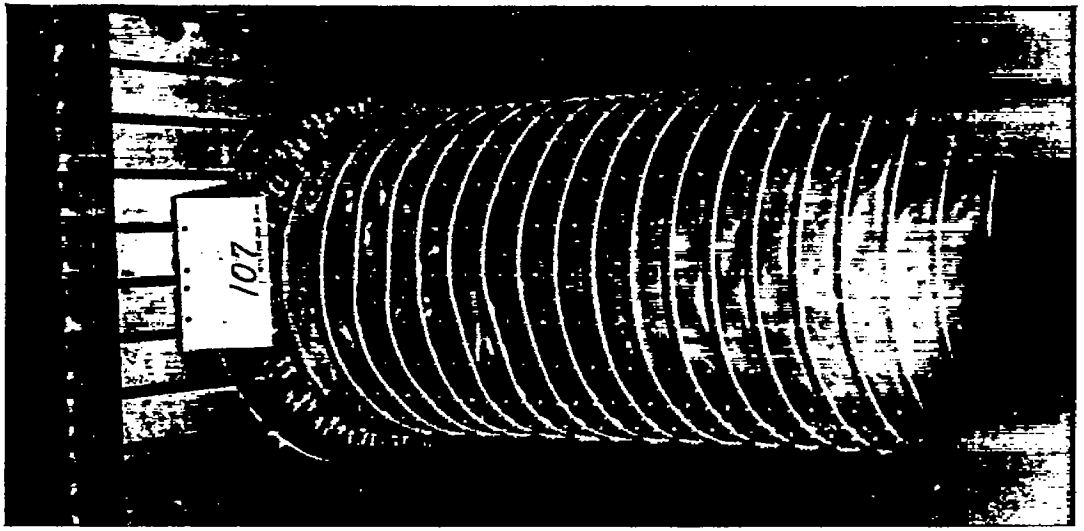


Fig. 72 (Specimen no. 107)

Top view of specimen which failed in compression. All specimens were loaded so that compression side was on top.

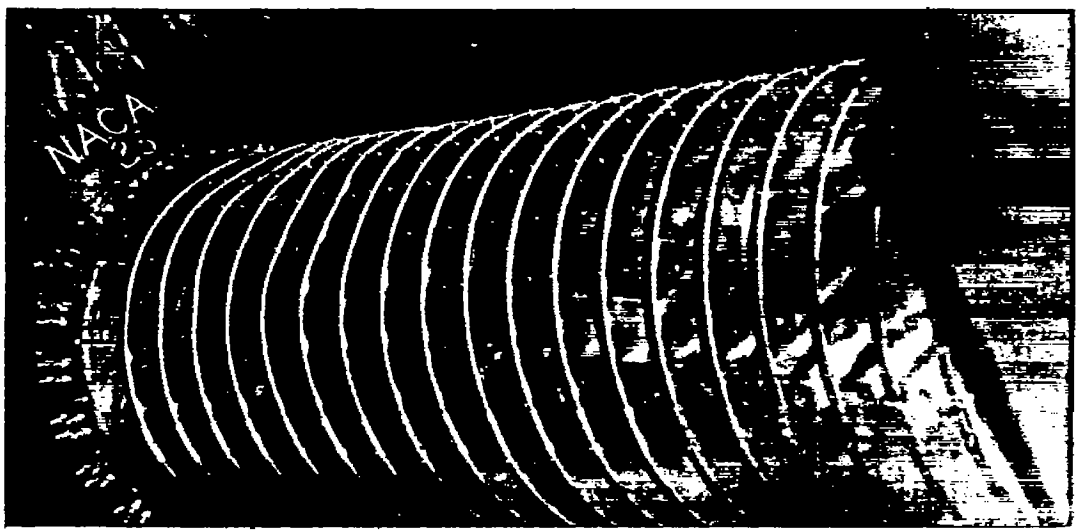


Fig. 73 (Specimen no. 107)

Side view of compression failure. Small buckles at sides appeared at failure.

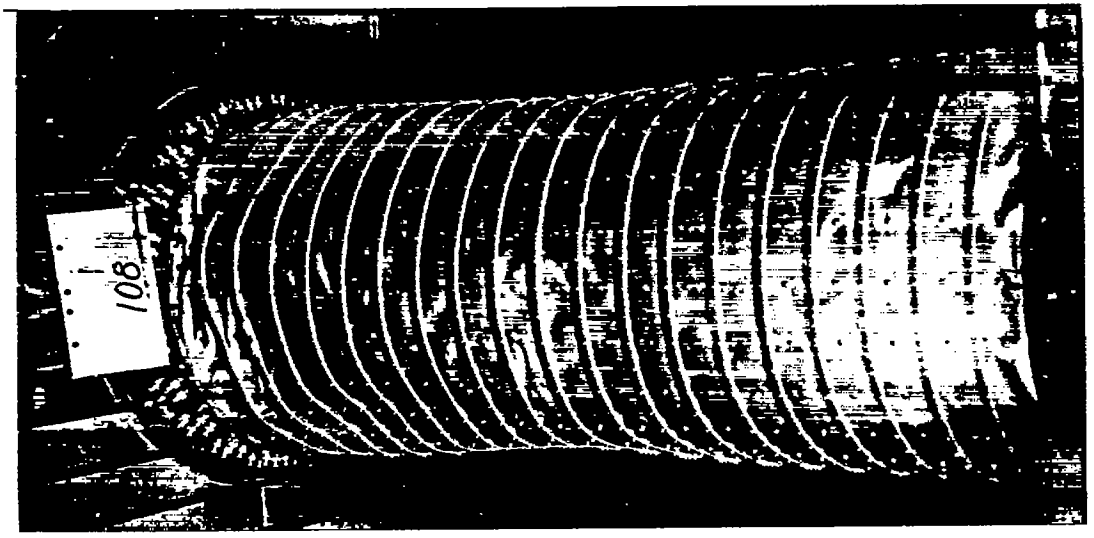


Fig. 74 (Specimen no. 108)

Top view showing compression failure. At failure, buckles extended slightly over the sides.

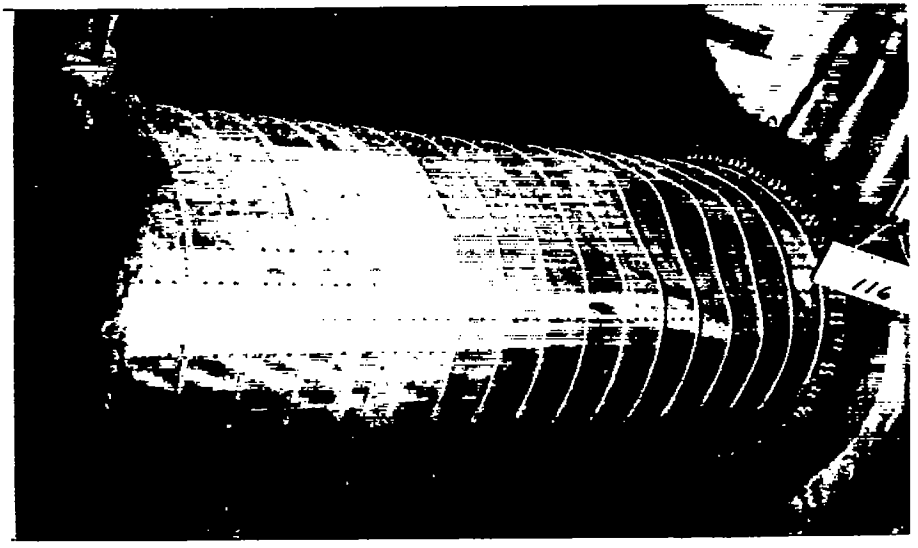


Fig. 75 (Specimen no. 116)

Failure occurred by simultaneous bending and shear. Note shear buckles on side and compression buckle on top.

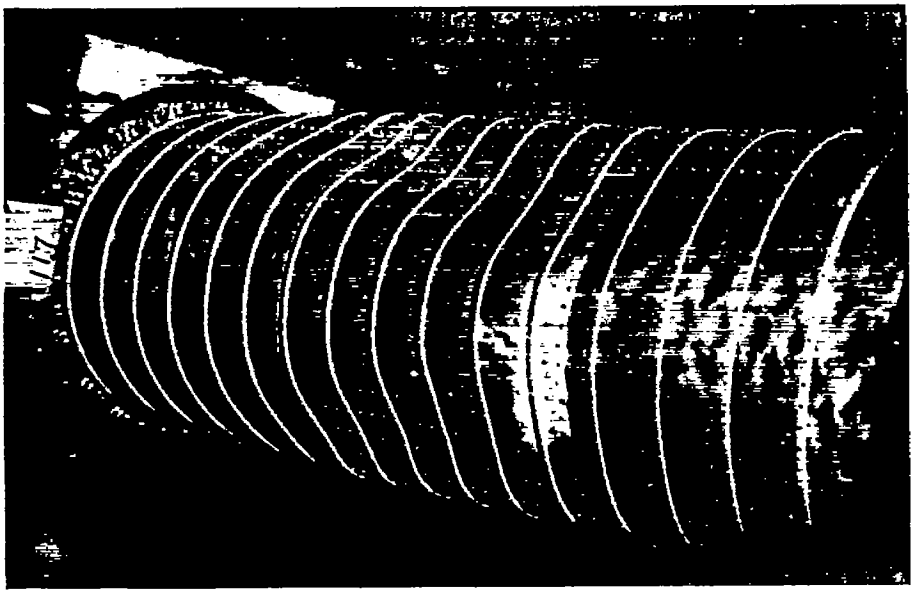


Fig. 76 (Specimen no. 117)

Top view of specimen which failed in shear. Note absence of compression buckles on top.

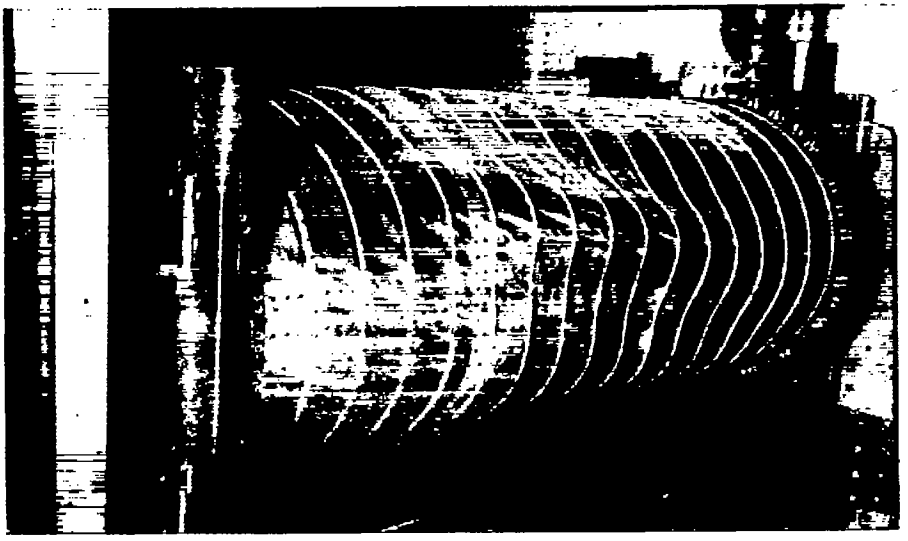


Fig. 77 (Specimen no. 117)

Side view of shear failure. Note that the shear buckles extend into the tension zone.

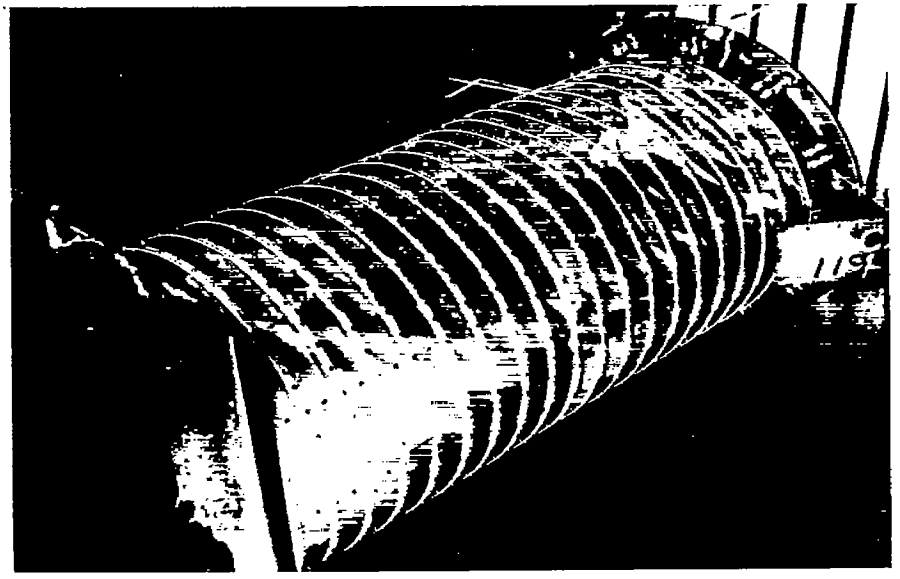


Fig. 78 (Specimen no. 119)

Compression failure. Note failure occurred over relatively small region near fixed end.

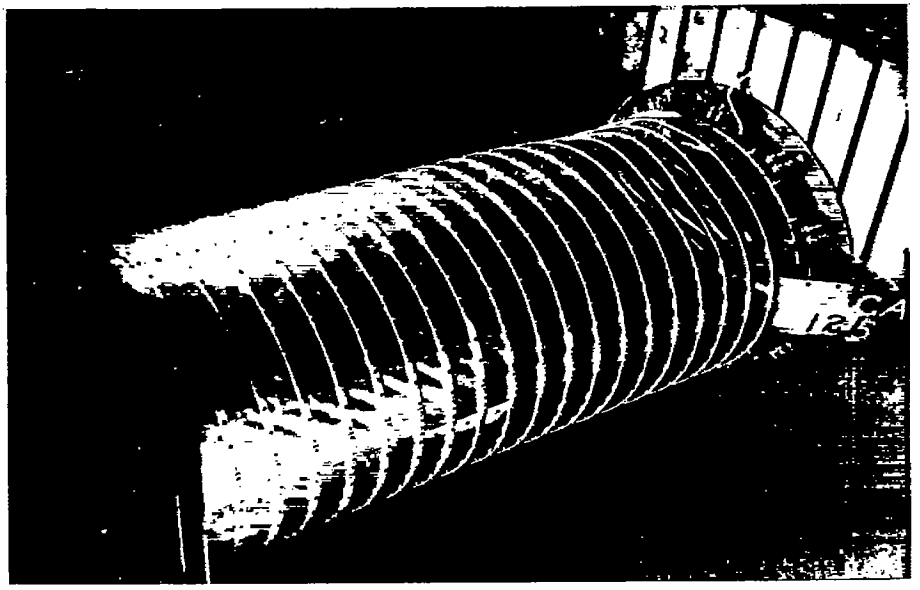


Fig. 79 (Specimen no. 121)

Compression failure.